

Ball Spline THK General Catalog

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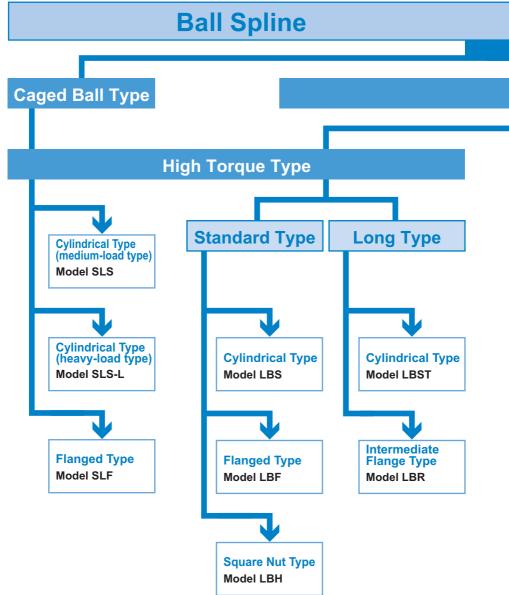
▲ Product Descriptions

Classification of Ball Splines	3-4 • Types and Features	АЗ-76
	 Housing Inner-diameter Tolerance 	АЗ-77
Point of Selection	3-6	
Flowchart for Selecting a Ball Spline A	3-6 Dimensional Drawing, Dimensional Table)
Steps for Selecting a Ball Spline	3-6 Model LT	A3-78
Selecting a Type A	3-8 Model LF	A3-80
Studying the Spline Shaft Strength		A3-82
Predicting the Service Life		e A3-84
Selecting a Preload		
Clearance in the Rotation Direction A		
Preload and Rigidity	3-29	
Conditions and Guidelines for Selecting of a Preload A		
Determining the Accuracy		АЗ-88
Accuracy Grades		
Accuracy Standards	F.	
, toodrady standards	Housing Inner-diameter Tolerance	A3-91
High Torque Caged Ball Spline	<u> </u>	
Models SLS, SLS-L and SLF	3-36 Dimensional Drawing, Dimensional Table)
Structure and Features	Marial IDO	
Types and Features	Madall DOT	
Housing Inner-diameter Tolerance	0 11 01 11	
_		
Dimensional Drawing, Dimensional Table	Rotary Ball Spline	
Model SLS	3-42 With Support Bearing Type Models LTR and LTR-	
Model SLF	3-44 • Structure and Features	
Spline Shaft	• Types and Features	
Accessories	Housing Inner-diameter Tolerance	A3-101
High Torque Type Bell Caline	Dimensional Drawing, Dimensional Table	è
High Torque Type Ball Spline	Model LTD A Compact Type	
Models LBS, LBST, LBF, LBR and LBH	ModelLTD	
Structure and Features And its attractions	J-DI • Calina Chaft	
Applications	3-32 - Dermissible Detetional Speed for Detany Poll Splines	
Types and Features	Maximum Manufacturing Longth by Accuracy	
Housing Inner-diameter Tolerance A	3-55	
Dimensional Drawing, Dimensional Table	Point of Design	
Miniature Ball Spline	3-56 Checking List for Spline Shaft End Shape	: A3-111
Model LBS (Medium Load Type)		A3-112
Model LBST (Heavy Load Type)	Positions of the Shilhe-hilt Keyway and Molinting Holes	3 A3-112
Model LBF (Medium Load Type)	3 62	
Model LBR		A3-113
Model LBH	Librication	АЗ-113
	wateriai and Surface Treatment	
Model LBS with Recommended Shaft End Shape A		АЗ-113
Spline Shaft Accessories	• Specifications of the Bellows	
	· · -	M 2 14F
Medium Torque Type Ball Spline	Model No.	
Models LT, LF and LT-X	Model Number Coding 3-74	АЗ-115
Structure and Features	3-75 Precautions on Use	АЗ-116

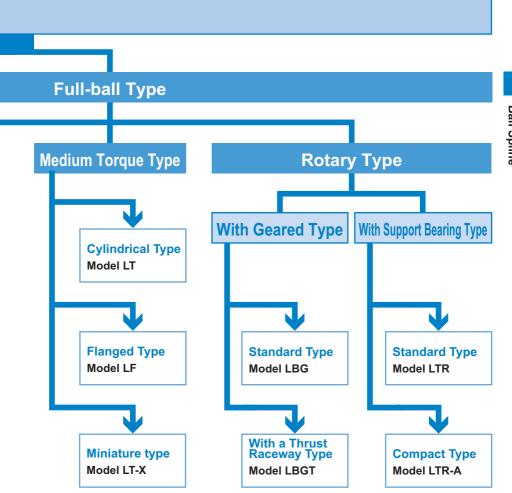
B Support Book (Separate)

Features and Types	. B 3-4
Features of the Ball Spline	. ■3-4
Structure and Features	. B 3-4
Classification of Ball Splines	. ■3-6
Point of Selection	
Flowchart for Selecting a Ball Spline	B3-8
Steps for Selecting a Ball Spline	B3-8
Selecting a Type	. B 3-10
Studying the Spline Shaft Strength	
Predicting the Service Life	. B 3-19
 Example of Calculating the Service Life 	
Mounting Procedure and Maintenance	■3-30
Assembling the Ball Spline	B3-30
Mounting the Spline	. B 3-30
Installing the Spline Nut	. ■ 3-32
Installation of the Spline Shaft	. B 3-32
Lubrication	. ■ 3-33
Options	. B 3-34
Material and Surface Treatment	B3-34
Contamination Protection	. B 3-34
Model No.	B3-35
Model Number Coding	B3-35
Precautions on Use	B3-36

Classification of Ball Splines



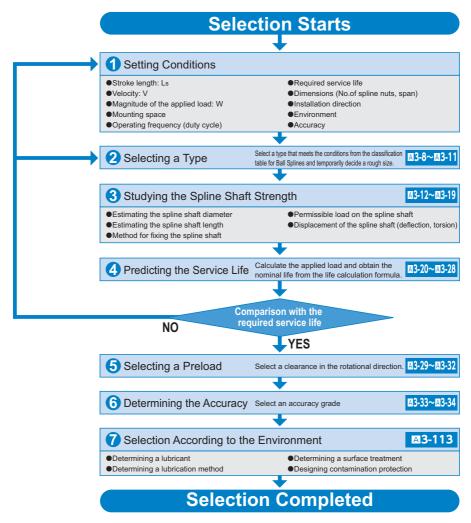
Classification of Ball Splines



Flowchart for Selecting a Ball Spline

Steps for Selecting a Ball Spline

The following is a flowchart to reference when selecting a Ball Spline.



Flowchart for Selecting a Ball Spline

Selecting a Type

There are three types of the Ball Spline: high torque type, medium torque type and rotary type. You can choose a type according to the intended use. In addition, wide arrays of spline nut shapes are available for each type, enabling the user to choose a desired shape according to the mounting or service requirements.

	Classification	Туре	Shape	Shaft diameter
ged Ball type ↑		Type SLS Type SLS-L		Nominal shaft diameter 25 to 100mm
High torque Caged Ball type		Type SLF		Nominal shaft diameter 25 to 100mm
		Type LBS Type LBST		Nominal shaft diameter 6 to 150mm
High torque type		Type LBF		Nominal shaft diameter 15 to 100mm
High tor	0000	Type LBR		Nominal shaft diameter 15 to 100mm
		Type LBH		Nominal shaft diameter 15 to 50mm

Selecting a Type

Specification Table	Structure and features	Major application				
⊠ 3-42	 Redesigning the shape of the conventional high torque type spline shaft to be more circular significantly improves its torsion and flexural rigidity. Models SLS/SLF adopt the caged-ball technology to enable the circulating motion of evenly spaced balls to be maintained and high-speed response to be achieved, the cycle time of the machine can be improved. Models SLS/SLF adopt the caged-ball technology, they eliminate collision and mu- 					
a 3-44	tual friction between balls, and realize low noise, pleasant running sound and low particle generation. • Models SLS/SLF adopt the caged-ball technology to substantially increase the grease retention, thus achieving long-term maintenance-free operation. • Models SLS/SLF adopt the caged-ball technology and a new circulation method, thus achieving stable and smooth motion with small rolling fluctuation.	 Column and arm of industrial robot Automatic loader Transfer machine Automatic conveyance system Tire molding machine Spindle of spot-welding machine Guide shaft of high-speed automatic coating 				
⊠3-56	The spline shaft has three crests equidistantly formed at angles of 120°. On both	machine Riveting machine Wire winder Work head of electric discharge machine Spindle drive shaft of grinding machine Speed gears Precision indexing machine				
⊠3-62	sides of each crest, two rows (six rows in total) of balls are arranged to hold the crest from both sides. The angular-contact design of the ball contact areas allows an appropriate preload to be evenly applied. Since the balls circulate inside the spline put the outer dimensions of the spline put.	Ğ				
⊠3-64	nut, the outer dimensions of the spline nut are compactly designed. Even under a large preload, smooth straight motion is achieved. Since the contact angle is large (45°) and the displacement is minimal, high rigidity is achieved.					
⊠3-66	 No angular backlash occurs. Capable of transmitting a large torque. 					

	Classification	Туре	Shape	Shaft diameter
ed.	ı	Type LT		Nominal shaft diameter 4 to 100mm
Medium torque type		Type LF		Nominal shaft diameter 6 to 50mm
Me	Med	Type LT-X		Nominal shaft diameter 4 to 6mm
Rotary type	Rotation	Type LBG Type LBGT		Nominal shaft diameter 20 to 85mm
Rotary	Rotation	Type LTR-A Type LTR		Nominal shaft diameter 8 to 60mm

Selecting a Type

Specification Table	Structure and features	Major application	
⊠3-78	The spline shaft has two to three crests. On both sides of each crest, two rows (four to six rows in total) of balls are arranged to hold the crest from both sides. This design allows	Die-set shaft and similar applications requiring industrial robot	
⊠3-80	 an appropriate preload to be evenly applied. The contact angle of 20° and an appropriate preload level eliminate angular backlash, providing high-torque moment rigidity. 	straight motion under a heavy load Loading system and similar applications requiring rotation to a given angle at a fixed position Automatic gas-welding Spot-welding machine Riveting machine Book-binding machine Automatic filler XY recorders	
A3-82	The length and external diameter of the LT-X ball spline's outer cylinder are the same as those of an LM-series linear bush, enabling the user to replace a linear bush with the LT-X.	machine spindle and similar applications requiring a whirl-stop on one shaft	
▲3-92	A unit type that has the same contact structure as model LBS. The flange circumference on the spline nut is machined to have gear teeth, and radial and thrust needle bearings are compactly combined on the circumference of the spline nut.	Speed gears for high torque transmission	
⊠3-102	A lightweight and compact type based on model LT, but has a spline nut circumference machined to have angular-contact type ball raceways to accommodate support bearings.	 Z axis of scalar robot Wire winder	

Studying the Spline Shaft Strength

The spline shaft of the Ball Spline is a compound shaft capable of receiving a radial load and torque. When the load and torque are large, the spline shaft strength must be taken into account.

[Spline Shaft Receiving a Bending Load]

When a bending load is applied to the spline shaft of a Ball Spline, obtain the spline shaft diameter using the equation (1) below.

$$\mathbf{M} = \mathbf{\sigma} \cdot \mathbf{Z}$$
 and $\mathbf{Z} = \frac{\mathbf{M}}{\mathbf{\sigma}}$ (1)

M : Maximum bending moment acting on the spline shaft (N-mm)

 σ : Permissible bending stress of the spline shaft (98N/mm²)

Z : Modulus section factor of the spline shaft (mm³) (see Table3 on M3-17, Table4 on M3-18 and Table5 on M3-19)

[Reference] Section Modulus (Solid Circle)

$$Z = \frac{\pi \cdot d^3}{32}$$

Z : Section Modulus (mm³)
d : Shaft outer diameter (mm)

[Spline Shaft Receiving a Torsion Load]

When a torsion load is applied on the spline shaft of a Ball Spline, obtain the spline shaft diameter using the equation (2) below.

$$T = \tau_a \cdot Z_P$$
 and $Z_P = \frac{T}{\tau_a}$ (2)

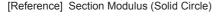
T : Maximum torsion moment (N-mm)

 τ_{a} : Permissible torsion stress of the spline shaft (49N/mm²)

spline snart (49N/mm²)

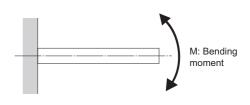
Z_p: Polar modulus of section of the spline nut (mm³)

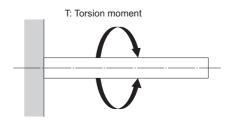
(see Table3 on \(\mathbb{A} 3-17, Table4 \) on \(\mathbb{A} 3-18 \) and Table5 on \(\mathbb{A} 3-19 \)



$$Z_P = \frac{\pi \cdot d^3}{16}$$

 Z_P : Section modulus (mm 3) d: Shaft outer diameter (mm)





Studying the Spline Shaft Strength

[When the Spline Shaft Simultaneously Receives a Bending Load and a Torsion Load]

When the spline shaft of a Ball Spline receives a bending load and a torsion load simultaneously, calculate two separate spline shaft diameters: one for the equivalent bending moment (M_e) and the other for the equivalent torsion moment (T_e). Then, use the greater value as the spline shaft diameter.

Equivalent bending moment

$$\mathbf{M}_{\circ} = \frac{\mathbf{M} + \sqrt{\mathbf{M}^2 + \mathbf{T}^2}}{2} = \frac{\mathbf{M}}{2} \left\{ 1 + \sqrt{1 + \left(\frac{\mathbf{T}}{\mathbf{M}}\right)^2} \right\} \cdots \cdots (3)$$

 $M_e = \sigma \cdot Z$

Equivalent torsion moment

$$T_{\circ} = \sqrt{M^2 + T^2} = M \cdot \sqrt{1 + \left(\frac{T}{M}\right)^2} \cdots (4)$$

 $T_e = \tau_a \cdot Z_p$

[Rigidity of the Spline Shaft]

The rigidity of the spline shaft is expressed as a torsion angle per meter of shaft length. Its value should be limited within 1°/4.

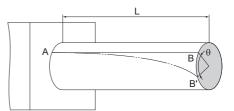
Rigidity of the shaft =
$$\frac{\text{Torsion angle}}{\text{Unit length}} = \frac{\theta \cdot \ell}{L} < \frac{1^{\circ}}{4}$$

θ : Torsion angle (°)
L : Spline shaft length (mm)

G : Transverse elastic modulus

(7.9×10⁴N/mm²) (1000mm)

△3-18 and Table5 on **△3-19**)



[Deflection and Deflection Angle of the Spline Shaft]

The deflection and deflection angle of the Ball Spline shaft need to be calculated using equations that meet the relevant conditions. Table1 and Table2 represent these conditions and the corresponding equations.

Table3 on **△3-17**, Table4 on **△3-18** and Table5 on **△3-19** show the modulus section (*Z*) and the geometrical moments of inertia (I) of the spline shaft. Using *Z* and I values in the tables, the strength and displacement (deflection) of a typical Ball Spline model can be obtained.

Table1 Deflection and Deflection Angle Equations

Support method	Condition	Deflection equation	Deflection angle equation
Both ends free	P i2	$\delta_{\text{max}} = \frac{P\ell^3}{48EI}$	$i_1 = 0$ $i_2 = \frac{P\ell^2}{16EI}$
Both ends fas- tened	P P	$\delta_{\text{max}} = \frac{P\ell^3}{192EI}$	$i_1 = 0$ $i_2 = 0$
Both ends free	Uniform load p	$\delta_{\text{max}} = \frac{5p\ell^4}{384\text{EI}}$	$i_2 = \frac{p\ell^3}{24EI}$
Both ends fas- tened	Uniform load p	$\delta_{\text{max}} = \frac{p\ell^4}{384\text{EI}}$	$i_2 = 0$

Studying the Spline Shaft Strength

Table2 Deflection and Deflection Angle Equations

Support method		Deflection equation	Deflection angle equation
One end fas- tened	P Eg	$\delta_{\text{max}} = \frac{P\ell^3}{3EI}$	$i_1 = \frac{P\ell^2}{2EI}$ $i_2 = 0$
One end fas- tened	Uniform load p	$\delta_{max} = \frac{p\ell^4}{8EI}$	$i_1 = \frac{p\ell^3}{6EI}$ $i_2 = 0$
Both ends free	VE WE	$\delta_{\text{max}} = \frac{\sqrt{3}\text{Mo}\ell^2}{216\text{EI}}$	$i_1 = \frac{M_0 \ell}{12EI}$ $i_2 = \frac{M_0 \ell}{24EI}$
Both ends fas- tened	× E	$\delta_{\text{max}} = \frac{\text{Mo}\ell^2}{216\text{EI}}$	$i_1 = \frac{Mo\ell}{16EI}$ $i_2 = 0$

 δ_{max} : Maximum deflection (mm)

M₀: Moment (N-mm)

ℓ: Span (mm)

I: Geometrical moment of inertia (mm⁴)

 i_1 : Deflection angle at loading point

i₂: Deflection angle at supporting point

P: Concentrated load (N)

p: Uniform load (N/mm)

E: Modulus of longitudinal elasticity 2.06×10⁵ (N/mm²)

[Dangerous Speed of the Spline Shaft]

When a Ball Spline shaft is used to transmit power while rotating, as the rotational speed of the shaft increases, the rotation cycle nears the natural frequency of the spline shaft. It may cause resonance and eventually result in inability to move. Therefore, the maximum rotational speed of the shaft must be limited to below the dangerous speed that does not cause resonance.

The dangerous speed of the spline shaft is obtained using the equation (6).

(0.8 is multiplied as a safety factor)

If the shaft's rotation cycle exceeds or nears the resonance point during operation, it is necessary to reconsider the spline shaft diameter.

Dangerous Speed

$$N_{c} = \frac{60\lambda^{2}}{2\pi \cdot \ell_{b}^{2}} \cdot \sqrt{\frac{E \times 10^{3} \cdot I}{\gamma \cdot A}} \times 0.8 \quad \cdots (6)$$

 N_c : Dangerous speed (min⁻¹) ℓ_b : Distance between two mounting surfaces (mm)

E: Young's modulus (2.06×10⁵ N/mm²)
I: Minimum geometrical moment of

inertia of the shaft

 $I = \frac{\pi}{64} d^4$ d:Minor diameter (mm)

(see Table8, Table9 and Table10 on A3-23)

(mm⁴)

γ : Density (specific gravity) (7.85×10⁻⁶kg/mm³)

 $A = \frac{\pi}{4} d^2 \qquad \text{d: Minor diameter} \qquad \text{(mm)}$

(see Table8, Table9 and Table10 on **A3-23**)

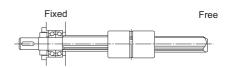
A : Spline shaft cross-sectional area (mm²)

Factor according to the mounting method
 (1) Fixed - free λ=1.875

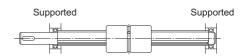
(2) Supported - supported λ =3.142

(3) Fixed - supported λ =3.927

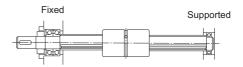
(4) Fixed - fixed λ =4.73



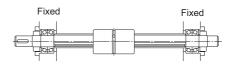
Fixed - free



Supported - supported



Fixed - supported



Fixed - fixed

Studying the Spline Shaft Strength

[Cross-sectional Characteristics of the Spline Shaft]

• Cross-sectional Characteristics of the Spline Shaft for Ball Spline Models SLS, SLS-L and SLF

Table3 Cross-sectional Characteristics of the Spline Shaft for Models SLS, SLS-L and SLF

Nominal shaft diameter		I: Geometrical moment of inertia mm ⁴	Z: Modulus section mm³	I _P : Polar moment of inertia mm⁴	Z _P : Section modulus mm³
05	Solid shaft	1.61×10⁴	1.29×10^{3}	3.22×10⁴	2.57×10^{3}
25	Hollow shaft	1.51×10⁴	1.20×10 ³	3.01×10⁴	2.41×10 ³
20	Solid shaft	3.33×10⁴	2.22×10 ³	6.65×10⁴	4.43×10³
30	Hollow shaft	3.00×10 ⁴	2.00×10 ³	6.01×10⁴	4.00×10 ³
40	Solid shaft	1.09×10⁵	5.47×10 ³	2.19×10 ⁵	1.09×10⁴
40	Hollow shaft	9.79×10⁴	4.90×10³	1.96×10 ⁵	9.79×10³
50	Solid shaft	2.71×10⁵	1.08×10⁴	5.41×10 ⁵	2.17×10⁴
50	Hollow shaft	2.51×10⁵	1.01×10⁴	5.03×10⁵	2.01×10⁴
60	Solid shaft	5.83×10⁵	1.94×10⁴	1.17×10 ⁶	3.89×10⁴
60	Hollow shaft	5.32×10⁵	1.77×10⁴	1.06×10 ⁶	3.54×10⁴
70	Solid shaft	1.06×10 ⁶	3.02×10⁴	2.11×10 ⁶	6.04×10⁴
00	Solid shaft	1.82×10 ⁶	4.55×10⁴	3.64×10 ⁶	9.10×10⁴
80	Hollow shaft	1.45×10 ⁶	3.62×10⁴	2.90×10 ⁶	7.24×10 ⁴
400	Solid shaft	4.50×10 ⁶	9.00×10⁴	9.00×10 ⁶	1.80×10⁵
100	Hollow shaft	3.48×10 ⁶	6.96×10⁴	6.96×10 ⁶	1.36×10⁵

Note) For the hole-shape of the hollow spline shaft, see **43-46**.

Cross-sectional Characteristics of the Spline Shaft for Ball Spline Models LBS, LBST, LBF, LBR, LBH, LBG and LBGT

Table4 Cross-sectional Characteristics of the Spline Shaft for Models LBS, LBST, LBF, LBR, LBH, LBG and LBGT

Nominal shaft diar	neter	I: Geometrical moment of inertia mm ⁴	Z: Modulus section mm³	I _P : Polar momentof inertia mm⁴	Z _P : Section modulus mm³
6	Solid shaft	50.6	17.8	1.03×10 ²	36.2
8	Solid shaft	1.64×10 ²	42.9	3.35×10 ²	87.8
10	Solid shaft	3.32×10 ²	73.0	6.80×10 ²	1.50×10 ²
15	Solid shaft	1.27×10³	2.00×10 ²	2.55×10 ³	4.03×10 ²
20	Solid shaft	3.82×10³	4.58×10 ²	7.72×10³	9.26×10 ²
20	Hollow shaft	3.79×10³	4.56×10 ²	7.59×10 ³	9.11×10 ²
05	Solid shaft	9.62×10³	9.14×10 ²	1.94×10⁴	1.85×10³
25	Hollow shaft	9.50×10³	9.05×10 ²	1.90×10⁴	1.81×10 ³
30	Solid shaft	1.87×10⁴	1.50×10 ³	3.77×10⁴	3.04×10 ³
30	Hollow shaft	1.78×10⁴	1.44×10³	3.57×10⁴	2.88×10 ³
40	Solid shaft	6.17×10⁴	3.69×10³	1.25×10⁵	7.46×10 ³
40	Hollow shaft	5.71×10⁴	3.42×10³	1.14×10⁵	6.84×10³
50	Solid shaft	1.49×10⁵	7.15×10³	3.01×10⁵	1.45×10 ⁴
50	Hollow shaft	1.34×10⁵	6.46×10 ³	2.69×10 ⁵	1.29×10 ⁴
60	Solid shaft	3.17×10⁵	1.26×10⁴	6.33×10⁵	2.53×10⁴
00	Hollow shaft	2.77×10 ⁵	1.11×10 ⁴	5.54×10 ⁵	2.21×10 ⁴
70	Solid shaft	5.77×10⁵	1.97×10⁴	1.16×10 ⁶	3.99×10⁴
70	Hollow shaft	5.07×10⁵	1.74×10⁴	1.01×10 ⁶	3.49×10 ⁴
85	Solid shaft	1.33×10 ⁶	3.69×10⁴	2.62×10 ⁶	7.32×10 ⁴
00	Hollow shaft	1.11×10 ⁶	3.10×10⁴	2.22×10 ⁶	6.20×10 ⁴
100	Solid shaft	2.69×10 ⁶	6.25×10⁴	5.33×10 ⁶	1.25×10⁵
100	Hollow shaft	2.18×10 ⁶	5.10×10⁴	4.37×10 ⁶	1.02×10⁵
120	Solid shaft	5.95×10 ⁶	1.13×10⁵	1.18×10 ⁷	2.26×10⁵
120	Hollow shaft	5.28×10 ⁶	1.01×10⁵	1.06×10 ⁷	2.02×10 ⁵
150	Solid shaft	1.61×10 ⁷	2.40×10 ⁵	3.20×10 ⁷	4.76×10⁵
150	Hollow shaft	1.40×10 ⁷	2.08×10⁵	2.79×10 ⁷	4.16×10⁵

Note) For the hole-shape of the hollow spline shaft, see **A3-69** and **A3-96**.

Studying the Spline Shaft Strength

• Cross-sectional Characteristics of the Spline Shaft for Ball Spline Models LT, LF, LT-X, LTR and LTR-A

Table5 Cross-sectional Characteristics of the Spline Shaft for Models LT, LF, LT-X, LTR and LTR-A

Nominal shaft diameter		I: Geometrical moment of inertia mm ⁴	Z: Modulus section mm ³	I _P : Polar moment of inertia mm⁴	Z _P : Section modulus mm³	
4	Solid shaft		11.39	5.84	22.78	11.68
5	Solid shaft		27.88	11.43	55.76	22.85
_	Solid shaft		57.80	19.7	1.19×10 ²	40.50
6	Hollow shaft	Туре К	55.87	18.9	1.16×10 ²	39.20
	Solid shaft		1.86×10 ²	47.4	3.81×10 ²	96.60
8	Hollow shaft	Type K	1.81×10 ²	46.0	3.74×10 ²	94.60
10	Solid shaft		4.54×10 ²	92.6	9.32×10 ²	1.89×10 ²
10	Hollow shaft	Туре К	4.41×10 ²	89.5	9.09×10 ²	1.84×10 ²
40	Solid shaft		1.32×10 ³	2.09×10 ²	2.70×10 ³	4.19×10 ²
13	Hollow shaft	Type K	1.29×10 ³	2.00×10 ²	2.63×10 ³	4.09×10 ²
	Solid shaft		3.09×10 ³	3.90×10 ²	6.18×10 ³	7.80×10 ²
16	I lalla ab aft	Type K	2.97×10 ³	3.75×10 ²	5.95×10 ³	7.51×10 ²
	Hollow shaft	Type N	2.37×10 ³	2.99×10 ²	4.74×10³	5.99×10 ²
	Solid shaft		7.61×10 ³	7.67×10 ²	1.52×10⁴	1.53×10 ³
20	Hollow shaft	Type K	7.12×10 ³	7.18×10 ²	1.42×10 ⁴	1.43×10 ³
	Hollow Shart	Type N	5.72×10 ³	5.77×10 ²	1.14×10 ⁴	1.15×10^{3}
	Solid shaft		1.86×10⁴	1.50×10 ³	3.71×10⁴	2.99×10^{3}
25	Hollow shaft	Type K	1.75×10⁴	1.41×10 ³	3.51×10⁴	2.83×10^{3}
	Tiollow shart	Type N	1.34×10⁴	1.08×10 ³	2.68×10⁴	2.16×10^{3}
	Solid shaft		3.86×10⁴	2.59×10 ³	7.71×10⁴	5.18×10 ³
30	Hollow shaft	Type K	3.53×10⁴	2.37×10 ³	7.07×10⁴	4.74×10 ³
	Hollow Shart	Type N	2.90×10⁴	1.95×10 ³	5.80×10⁴	3.89×10 ³
	Solid shaft		5.01×10⁴	3.15×10 ³	9.90×10⁴	6.27×10 ³
32	Hollow shaft	Type K	4.50×10⁴	2.83×10 ³	8.87×10⁴	5.61×10 ³
	Tiollow share	Type N	3.64×10⁴	2.29×10 ³	7.15×10⁴	4.53×10 ³
	Solid shaft		1.22×10⁵	6.14×10 ³	2.40×10 ⁵	1.21×10⁴
40	Hollow shaft	Type K	1.10×10⁵	5.55×10 ³	2.17×10 ⁵	1.10×10⁴
	Tiollow Share	Type N	8.70×10⁴	4.39×10 ³	1.71×10⁵	8.64×10 ³
	Solid shaft		2.97×10⁵	1.20×10⁴	5.94×10⁵	2.40×10⁴
50	Hollow shaft	Type K	2.78×10⁵	1.12×10⁴	5.56×10⁵	2.24×10⁴
	Tiollow Share	Type N	2.14×10 ⁵	8.63×10 ³	4.29×10 ⁵	1.73×10⁴
60	Solid shaft		6.16×10⁵	2.07×10⁴	1.23×10 ⁶	4.14×10⁴
- 00	Hollow shaft	Type K	5.56×10⁵	1.90×10⁴	1.13×10 ⁶	3.79×10⁴
80	Solid shaft		1.95×10 ⁶	4.91×10⁴	3.90×10 ⁶	9.82×10⁴
	Hollow shaft	Type K	1.58×10 ⁶	3.97×10⁴	3.15×10 ⁶	7.95×10⁴
100	Solid shaft		4.78×10 ⁶	9.62×10⁴	9.56×10 ⁶	1.92×10⁵
100	Hollow shaft	Type K	3.76×10 ⁶	7.57×10⁴	7.52×10 ⁶	1.51×10⁵

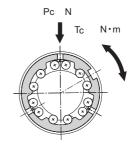
Note) For the hole-shape of the hollow spline shaft. For type K: see A3-85 and A3-106. For type N: see A3-85 and A3-106.

Predicting the Service Life

[Nominal Life]

The service life of a Ball Spline varies from unit to unit even if they are manufactured through the same process and used in the same operating conditions. Therefore, the nominal life defined below is normally used as a guidepost for obtaining the service life of a Ball Spline.

Nominal life is the total travel distance that 90% of a group of identical ball splines independently operating under the same conditions can achieve without showing flaking (scale-like pieces on a metal surface).



[Calculating the Nominal Life]

The nominal life of a Ball Spline varies with types of loads applied during operation: torque load, radial load and moment load. The corresponding nominal life values are obtained using the equations (7) to (10) below. (The basic load ratings in these loading directions are indicated in the specification table for the corresponding model number.)

When a Torque Load is Applied

$$L = \left(\frac{f_{\tau} \cdot f_{c}}{f_{w}} \cdot \frac{C_{\tau}}{T_{c}}\right)^{3} \times 50 \quad \cdots (7)$$

When a Radial Load is Applied

$$L = \left(\frac{f_{\tau} \cdot f_{c}}{f_{w}} \cdot \frac{C}{P_{c}}\right)^{3} \times 50 \quad \dots (8)$$

fc : Contact factor

(see Table6 on **△3-22**)

fw : Load factor (see Table7 on A3-22)

Predicting the Service Life

When a Torque Load and a Radial Load are Simultaneously Applied

When a torque load and a radial load are simultaneously applied, calculate the nominal life by obtaining the equivalent radial load using the equation (9) below.

$$\mathbf{P}_{E} = \mathbf{P}_{c} + \frac{4 \cdot \mathbf{T}_{c} \times 10^{3}}{i \cdot \mathbf{dp} \cdot \mathbf{cos}\alpha} \qquad \dots (9)$$

$$\mathbf{P}_{E} : \text{Equivalent radial load}$$

 $\cos \alpha$: Contact angle i=Number of rows of balls under a load

Type LBS
$$\alpha$$
=45° i =2 (LBS10 or smaller) Type SLS α =40° i =3 (LBS15 or greater)

Type LT α =70° i =2 (LT13 or smaller)
 i =3 (LT16 or greater)

dp : Ball center-to-center diameter (mm) (see Table8, Table9 and Table10 on 🔼 3-23)

When a Moment Load is Applied to a Single Nut or Two Nuts in Close Contact with Each Other

Obtain the equivalent radial load using the equation (10) below.

$$\mathbf{P}_{\mathbf{u}} = \mathbf{K} \cdot \mathbf{M} \qquad \cdots \cdots (10)$$

P_u : Equivalent radial load (N)

(with a moment applied)

K : Equivalent Factors

(see Table11 on **A3-26**, Table12 on **A3-27** and Table13 on **A3-28**)

M : Applied moment (N-mm)

However, M should be within the range of the static permissible moment.

• When a Moment Load and a Radial Load are Simultaneously Applied

Calculated the nominal life from the sum of the radial load and the equivalent radial load.

Calculating the Service Life Time

When the nominal life (L) has been obtained in the equation above, if the stroke length and the number of reciprocations per minute are constant, the service life time is obtained using the equation (11) below.

 $\begin{array}{lll} L_h & : Service \ life \ time & (h) \\ \ell_S & : Stroke \ length & (m) \\ n_1 & : Number \ of \ reciprocations \ per \ minute \\ & & (min^{-1}) \end{array}$

■f_T: Temperature Factor

If the temperature of the environment surrounding the operating Ball Spline exceeds 100°C, take into account the adverse effect of the high temperature and multiply the basic load ratings by the temperature factor indicated in Fig. 1.

In addition, the Ball Spline must be of a high

In addition, the Ball Spline must be of a high temperature type.

Note) If the environment temperature exceeds 80°C, hightem-perature types of seal and retainer are required. ContactTHK for details.

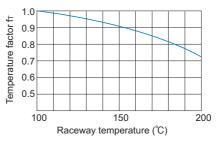


Fig.1 Temperature Factor (f_T)

■fc: Contact Factor

When multiple spline nuts are used in close contact with each other, their linear motion is affected by moments and mounting accuracy, making it difficult to achieve uniform load distribution. In such applications, multiply the basic load rating (C) and (C₀) by the corresponding contact factor in Table6.

Note) If uneven load distribution is expected in a large machine, take into account the respective contact factor

Table6 Contact Factor (fc)

Number of spline nuts in close contact with each other	Contact factor f₀
2	0.81
3	0.72
4	0.66
5	0.61
Normal use	1

■fw: Load Factor

indicated in Table6.

In general, reciprocating machines tend to involve vibrations or impact during operation. It is extremely difficult to accurately determine vibrations generated during high-speed operation and impact during frequent start and stop. When loads applied on a Ball Spline cannot be measured, or when speed and impact have a significant influence, divide the basic load rating (C or C₀), by the corresponding load factor in the table of empirically obtained data on Table7.

Table7 Load Factor (fw)

Vibrations/ impact	Speed (V)	f _w	
Faint	Very low V≦0.25m/s	1 to 1.2	
Weak	Slow 0.25 <v≦1m s<="" td=""><td colspan="2">1.2 to 1.5</td></v≦1m>	1.2 to 1.5	
Medium	Medium 1 <v≦2m s<="" td=""><td colspan="2">1.5 to 2</td></v≦2m>	1.5 to 2	
Strong	High V>2m/s	2 to 3.5	

Predicting the Service Life

Table8 Sectional Shape of the Spline Shaft for Models SLS, SLS-L and SLF

Unit: mm

Nominal shaft diameter	25	30	40	50	60	70	80	100
Minor diameter φ d	21.6	25.8	35.2	44.4	54.0	62.8	71.3	90.0
Major diameter φ D ₀ h7	25	30	40	50	60	70	80	100
Ball center-to-center diameter ϕ dp	25.2	30.2	40.6	50.6	61.0	71.0	80.8	101.2

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

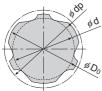


Table9 Sectional Shape of the Spline Shaft for Models LBS, LBST, LBF, LBR, LBH, LBG and LBGT

Unit: mm

Nominal shaft diameter	15	20	25	30	40	50	60	70	85	100	120	150
Minor diameter ø d	11.7	15.3	19.5	22.5	31	39	46.5	54.5	67	81	101	130
Outer diameter ϕ D ₀	14.5	19.7	24.5	29.6	39.8	49.5	60	70	84	99	117	147
Ball center-to-center diameter ϕ dp	15	20	25	30	40	50	60	70	85	100	120	150

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

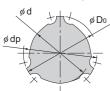
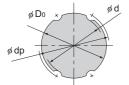


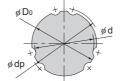
Table10 Sectional Shape of the Spline Shaft for Models LT, LF, LT-X, LTR and LTR-A

Unit: mm

Nominal shaft diameter	4	5	6	8	10	13	16	20	25	30	32	40	50	60	80	100
Minor diameter ϕ d	3.5	4.5	5	7	8.5	11.5	14.5	18.5	23	28	30	37.5	46.5	56.5	75.5	95
Outer diameter ϕ D ₀ h7	4	5	6	8	10	13	16	20	25	30	32	40	50	60	80	100
Ball center-to-center diameter ϕ dp	4.6	5.7	7	9.3	11.5	14.8	17.8	22.1	27.6	33.2	35.2	44.2	55.2	66.3	87.9	109.5
Outer diameter tolerance	_	0 0.012		0 -0.0)15	0 -0.0)18		0 -0.	021		0 -0.0)25	0 -0.	03	0 -0.035

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.





[Calculating the Average Load]

When the load applied on the spline shaft fluctuates according to varying conditions, such as an industrial robot arm traveling forward while holding a workpiece and traveling backward with empty weight, and a machine tool handling various workpieces, this varying load condition must be taken into account in service life calculation.

The average load (P_m) is a constant load under which the service life of an operating Ball Spline with its spline nut receiving a fluctuation load in varying conditions is equivalent to the service life under this varying load condition.

The following is the basic equation.

$$\mathbf{P}_{m} = \sqrt[3]{\frac{1}{\mathbf{L}} \cdot \sum_{n=1}^{n} (\mathbf{P}_{n}^{3} \cdot \mathbf{L}_{n})}$$

$$\mathbf{P}_{m} : \text{Average Load} \qquad (N)$$

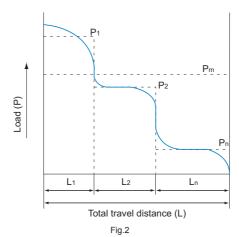
$$\mathbf{P}_{n} : \text{Varying load} \qquad (N)$$

$$\mathbf{L} : \text{Total travel distance} \qquad (mm)$$

$$\mathbf{L}_{n} : \text{Distance traveled under } \mathbf{P}_{n} \qquad (mm)$$

When the Load Fluctuates Stepwise

$$\mathbf{P}_{m} = \sqrt[3]{\frac{1}{L} \left(\mathbf{P}_{1}^{3} \cdot \mathbf{L}_{1} + \mathbf{P}_{2}^{3} \cdot \mathbf{L}_{2} \cdot \cdots + \mathbf{P}_{n}^{3} \cdot \mathbf{L}_{n} \right)} \cdot \cdots \cdot (12)$$



Predicting the Service Life

When the Load Fluctuates Monotonically

$$P_{m} \doteqdot \frac{1}{3} \left(P_{min} + 2 \cdot P_{max} \right) \cdots (13)$$

 $P_{\mbox{\tiny min}}$: Minimum load

(N)

P_{max}: Maximum load

(N)

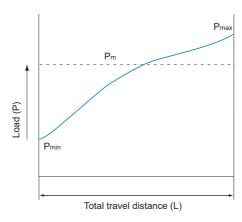
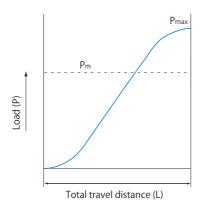


Fig.3

• When the Load Fluctuates Sinusoidally

(a)
$$P_m = 0.65P_{max} \cdots (14)$$



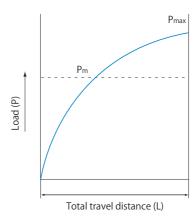


Fig.4

[Equivalent Factor]
Table11 on ▲3-26, Table12 on ▲3-27, Table13 and Table14 on ▲3-28 show equivalent radial load factors calculated under a moment load.

• Table of Equivalent Factors for Ball Spline Models SLS/SLF

M A1

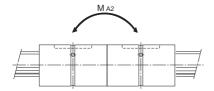
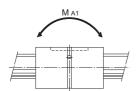


Table11

	Equivalen	t factor: K
Model No.	Single spline nut	Two spline nuts in close contact with each other
SLS/SLF 25	0.187	0.030
SLS 25L	0.148	0.027
SLS/SLF 30	0.153	0.027
SLS 30L	0.129	0.024
SLS/SLF 40	0.114	0.021
SLS 40L	0.102	0.019
SLS/SLF 50	0.109	0.018
SLS 50L	0.091	0.017
SLS/SLF 60	0.080	0.015
SLS 60L	0.072	0.014
SLS/SLF 70	0.101	0.016
SLS 70L	0.076	0.014
SLS/SLF 80	0.083	0.013
SLS 80L	0.072	0.012
SLS/SLF 100	0.068	0.011
SLS 100L	0.056	0.010

Predicting the Service Life

• Table of Equivalent Factors for Ball Spline Model LBS



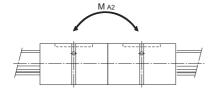


Table12

	Equivalen	t factor: K	
Model No.	Single spline nut	Two spline nuts in close contact with each other	
LBS 6	0.61	0.074	
LBS 8	0.46	0.060	
LBS 10	0.54	0.049	
LBS 15	0.22	0.039	
LBS 20	0.24	0.03	
LBST 20	0.17	0.027	
LBS 25	0.19	0.026	
LBST 25	0.14	0.023	
LBS 30	0.16	0.022	
LBST 30	0.12	0.02	
LBS 40	0.12	0.017	
LBST 40	0.1	0.016	
LBS 50	0.11	0.015	
LBST 50	0.09	0.014	
LBST 60	0.08	0.013	
LBS 70	0.1	0.013	
LBST 70	0.08	0.012	
LBS 85	0.08	0.011	
LBST 85	0.07	0.01	
LBS 100	0.08	0.009	
LBST 100	0.06	0.009	
LBST 120	0.05	0.008	
LBST 150	0.045	0.006	

Note1) Values of equivalent factor K for model LBF are the same as that for model LBS.

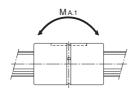
Note2) Values of equivalent factor K for models LBR, LBG, LBGT and LBH are the same as that for model LBST.

However the values of model LBF60 are the same

as that for model LBST60.

The values of model LBH15 are the same as that for model LBS15.

• Table of Equivalent Factors for Ball Spline Model LT



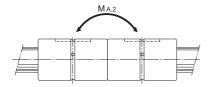


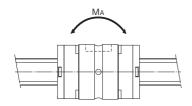
Table13

Table 13							
	Equivalen	t factor: K					
Model No.	Single spline nut	Two spline nuts in close contact with each other					
LT 4	0.65	0.096					
LT 5	0.55	0.076					
LT 6	0.47	0.06					
LT 8	0.47	0.058					
LT 10	0.31	0.045					
LT 13	0.3	0.042					
LT 16	0.19	0.032					
LT 20	0.16	0.026					
LT 25	0.13	0.023					
LT 30	0.12	0.02					
LT 40	0.088	0.016					
LT 50	0.071	0.013					
LT 60	0.07	0.011					
LT 80	0.062	0.009					
LT100	0.057	0.008					

Note) Values of equivalent factor K for models LF, LTR and LTR-A are the same as that for model LT. However, the equivalent factor for model LTR32 is the same as that for model LT30.

Table14

• Table of Equivalent Factors for Ball Spline Model LT-X



	Equivalent factor: K				
Model No.	Single spline nut	Two spline nuts in close contact with each other			
LT 4X	0.830	0.104			
LT 5X	0.818	0.097			
LT 5XL	0.359	0.063			
LT 6X	0.553	0.081			
LT 6XL	0.299	0.055			

Note) The values shown are those for models equipped with seals.

Selecting a Preload

Selecting a Preload

A preload on the Ball Spline significantly affects its accuracy, load resistance and rigidity. Therefore, it is necessary to select the most appropriate clearance according to the intended use.

Specific clearance values are standardized for each model, allowing you to select a clearance that meets the conditions.

Clearance in the Rotation Direction

With the Ball Spline, the sum of clearances in the circumferential direction is standardized as the clearance in the rotational direction. For models LBS and LT, which are especially suitable for transmission of rotational torque, clearances in the rotational directions are defined.

Clearance in the rotational direction (BCD)

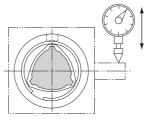
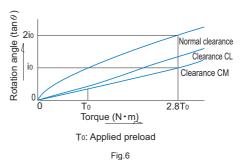


Fig.5 Measurement of Clearance in the Rotational Direction

Preload and Rigidity

Preload is defined as the load preliminarily applied to the ball in order to eliminate angular backlash (clearance in the rotational direction) and increase rigidity. When given a preload, the Ball Spline is capable of increasing its rigidity by eliminating the angular backlash according to the magnitude of the preload. Fig.6 shows the displacement in the rotational direction when a rotational torque is applied.

Thus, the effect of a preload can be obtained up to 2.8 times that of the applied preload. When given the same rotational torque, the displacement when a preload is applied is 0.5 or less of that without a preload. The rigidity with a preload is at least twice greater than that without a preload.



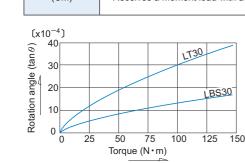
Conditions and Guidelines for Selecting of a Preload

Table15 provides guidelines for selecting a clearance in the rotational direction with given conditions of the Ball Spline.

The rotational clearance of the Ball Spline significantly affects the accuracy and rigidity of the spline nut. Therefore, it is essential to select a correct clearance according to the intended use. Generally, the Ball Spline is provided with a preload. When it is used in repeated circular motion or reciprocating straight motion, the Ball Spline is subject to a large vibration impact, and therefore, its service life and accuracy are significantly increased with a preload.

Table 15 Guidelines for Selecting a Clearance in the Rotational Direction for the Ball Spline

Clearance in the rotation direction	Condition	Examples of applications
Normal grade (No symbol)	 Smooth motion with a small force is desired. A torque is always applied in the same direction. 	Measuring instruments Automatic drafting machine Geometrical measuring equipment Dynamometer Wire winder Automatic welding machine Main shaft of horning machine Automatic packing machine
Light preload (CL)	 An overhang load or moment load is present. High positioning repeatability is required. Alternating load is applied. 	 Industrial robot arm Automatic loaders Guide shaft of automatic coating machine Main shaft of electric discharge machine Guide shaft for press die setting Main shaft of drilling machine
Medium preload (CM)	High rigidity is required and vibrations and impact are applied. Receives a moment load with a single spline nut.	Steering shaft of construction vehicle Shaft of spot-welding machine Indexing shaft of automatic lathe tool rest





25

 $[x10^{-4}]$

40

30 20

10 0

Rotation angle $(\tan \theta)$

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	СМ
25 30 40	+1 to -2	-2 to -6	−6 to −10
50 60	+2 to -4	-4 to −8	-8 to -12
70 80 100	+4 to -8	-8 to −12	-12 to -20

Table16 Clearance in the Rotational Direction for Models SLS, SLS-L and SLF

75

50

100

LT30

BS30

150

Unit:um

125

Selecting a Preload

Table17 Clearance in the Rotational Direction for Models LBS, LBF, LBST, LBR and LBH

Unit: µm

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	СМ
6 8	-2 to +1	−6 to −2	
10 15	−3 to +2	−9 to −3	−15 to −9
20 25 30	-4 to +2	−12 to −4	−20 to −12
40 50 60	-6 to +3	−18 to −6	−30 to −18
70 85	-8 to +4	−24 to −8	−40 to −24
100 120	–10 to +5	−30 to −10	−50 to −30
150	-15 to +7	−40 to −15	-70 to -40

Table18 Clearance in the Rotational Direction for Models LT and LF

Unit: µm

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	СМ
4 5 6 8 10 13	-2 to +1	−6 to −2	_
16 20	−2 to +1	−6 to −2	−9 to −5
25 30	-3 to +2	−10 to −4	−14 to −8
40 50	-4 to +2	−16 to −8	−22 to −14
60 80	−5 to +2	−22 to −12	−30 to −20
100	-6 to +3	−26 to −14	−36 to −24

Table19 Clearance in the Rotational Direction for Model LT-X

Unit: µm

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	CM
4	-2 to +1	-6 to −2	_
5	-2 to +1	-6 to -2	_
6	-2 to +1	-6 to -2	_

Table20 Clearance in the Rotational Direction for Models LBG and LBGT

Unit: μm

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	CM
20 25 30	-4 to +2	−12 to −4	−20 to −12
40 50 60	-6 to +3	−18 to −6	−30 to −18
70 85	-8 to +4	−24 to −8	-40 to -24

Table21 Clearance in the Rotational Direction for Model LTR

Unit: µm

Symbol	Normal	Light preload	Medium preload				
Nominal shaft diameter	No Symbol	CL	CM				
8 10	-2 to +1	−6 to −2	_				
16 20	–2 to +1	−6 to −2	−9 to −5				
25 32	−3 to +2	−10 to −4	−14 to −8				
40 50	-4 to +2	−16 to −8	−22 to −14				
60	-5 to +2	−22 to −12	-30 to -20				

Determining the Accuracy

Determining the Accuracy

Accuracy Grades

The accuracy of the Ball Spline is classified into three grades: normal grade (no symbol), high accuracy grade (H) and precision grade (P), according to the runout of spline nut circumference in relation to the support of the spline shaft. Fig.9 shows measurement items.

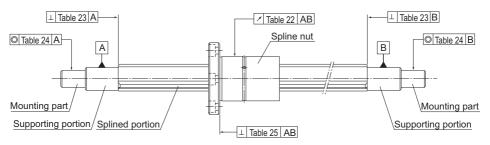


Fig.9 Accuracy Measurement Items of the Ball Spline

Accuracy Standards

Table22 to Table25 show measurement items of the Ball Spline.

Table22 Runout of the Spline Nut Circumference in Relation to the Support of the Spline Shaft

Unit: µm

Accu	ıracy		Runout(max)																						
diam Overall	al shaft neter I spline	4 1	to 8	Note		10		13	3 to 2	20	25	i to (32	4	0, 5	0	60) to 8	30	85	to 1	20		150	
shaft len	gth (mm)			_			_			_			_			_			_			_			_
Above	Or less	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision
_	200	72	46	26	59	36	20	56	34	18	53	32	18	53	32	16	51	30	16	51	30	16	_	_	_
200	315	133	(89)	_	83	54	32	71	45	25	58	39	21	58	36	19	55	34	17	53	32	17	_		-
315	400	_	_	_	103	68	_	83	53	31	70	44	25	63	39	21	58	36	19	55	34	17	_	_	_
400	500	_	_	_	123	_	_	95	62	38	78	50	29	68	43	24	61	38	21	57	35	19	46	36	19
500	630	_	_	_	_	_	_	112	_	_	88	57	34	74	47	27	65	41	23	60	37	20	49	39	21
630	800	_	_	_	_	_	_	_	_	_	103	68	42	84	54	32	71	45	26	64	40	22	53	43	24
800	1000	_	_	_	_	_	_	_	_	_	124	83	_	97	63	38	79	51	30	69	43	24	58	48	27
1000	1250	_	_	_	_	_	_	_	_	_	_	_	_	114	76	47	90	59	35	76	48	28	63	55	32
1250	1600	_	_	_	_	_	_	_	_	_	_	_	_	139	93	_	106	70	43	86	55	33	80	65	40
1600	2000	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	128	86	54	99	65	40	100	80	50
2000	2500	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	156	_	_	117	78	49	125	100	68
2500	3000	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	143	96	61	150	129	84

Note) Dimensions in parentheses do not apply to nominal shaft diameter of 4. Note) Applicable to models LBS, LBST, LBF, LBR, LT and LF.

Table23 Perpendicularity of the Spline Shaft End Face in Relation to the Support of the Spline Shaft

Accuracy	Perpendicularity (max)										
Nominal shaft diameter	Normal grade (No symbol) High accuracy grade (H) Precision Grad										
4 5 6 8 10	22	9	6								
13 15 16 20	27	11	8								
25 30 32	33	13	9								
40 50	39	16	11								
60 70 80	46	19	13								
85 100 120	54	22	15								
150	63	25	18								

Table24 Concentricity of the Part-mounting in Relation to the Support of the Spline Shaft

Unit: µm

Unit: µm

Accuracy	Concentricity (max)										
Nominal shaft diameter	Normal grade (No symbol) High accuracy grade (H) Precision Grade										
4 5 6 8	33	14	8								
10	41	17	10								
13 15 16 20	46	19	12								
25 30 32	53	22	13								
40 50	62	25	15								
60 70 80	73	29	17								
85 100 120	86	34	20								
150	100	40	23								

Table25 Straightness of the Flange-mounting Surface of the Spline Nut in Relation to the Support of the Spline Shaft Unit: μ m

Accuracy	Perpendicularity (max)									
Nominal shaft diameter	Normal grade (No symbol) High accuracy grade (H) Precision Gra									
6 8	27	11	8							
10 13	33	13	9							
15 16 20 25 30	39	16	11							
40 50	46	19	13							
60 70 80 85	54	22	15							
100	63	25	18							

Note) This table does not apply to models LBG, LBGT, LTR and LTR-A.

Determining the Accuracy

High Torque Caged Ball Spline



Models SLS, SLS-L and SLF

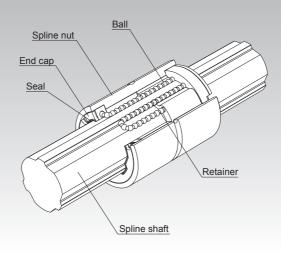


Fig.1 Structure of High Torque Caged Ball Spline

Point of Selection	A 3-6
Point of Design	A3-111
Options	△3-113
Model No.	A 3-115
Precautions on Use	△3-116
Accessories for Lubrication	A24-1
Mounting Procedure and Maintenance	₿3-30
Cross-sectional Characteristics of the Spline Shaft	A3-17
Equivalent factor	△3-26
Clearance in the Rotation Direction	▲3-29
Accuracy Standards	A3-33
Maximum Manufacturing Length by Accuracy	△ 3-109

High Torque Caged Ball Spline

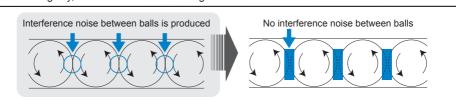
Structure and Features

The caged-ball technology, developed by bringing together THK's technologies and know-how, is now integrated in the new Ball Spline.

The integration of the ball cage enables the circulating motion of evenly spaced balls and highspeed response to be achieved.

It eliminates collision and mutual friction between balls, and realizes low noise, pleasant running sound and low particle generation. As the grease retention is increased, long-term maintenance-free operation is also achieved.

The high-torque design provides the nut with excellent torsional rigidity. The spline shaft also has enhanced rigidity, thanks to its rounded design.



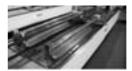
[High-speed Response]

Models SLS/SLF adopt the caged-ball technology to enable the circulating motion of evenly spaced balls to be maintained and high-speed response to be achieved, the cycle time of the machine can be improved.

[Conditions]

Model tested	SLS50				
Testing environment	22 to 27.5℃				
Stroke	1000mm				
Maximum speed	200m/min				
Acceleration/deceleration	5G(49m/s²)				
Applied load	Light preload(CL)				
Lubricant	THK AFB-LF Grease				
	L				

Appearance of the test machine (high-speed durability test)



Appearance of the specimen



[Test results]

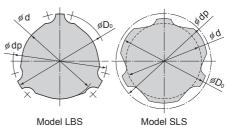
No anomaly after travelling 10.000 km

[Improvement on the spline shaft's rigidity]

Redesigning the shape of the conventional high torque type spline shaft to be more circular significantly improves its torsion and flexural rigidity.

Unit: mm

Nominal shaft diameter 25	LBS	SLS
Minor diameter φ d	19.5	21.6
Major diameter φ D ₀	24.5	25.0
Ball center-to-center diameter φ dp	25	25.2





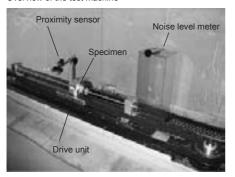
[Low Noise, Pleasant Running Sound and Low Particle Generation]

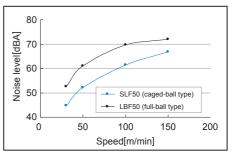
Models SLS/SLF adopt the caged-ball technology, they eliminate collision and mutual friction between balls, and realize low noise, pleasant running sound and low particle generation.

[Conditions]

Model tested	SLF50/LBF50				
Stroke	600mm				
Speeds	30,50,100,150m/min				
Measuring instrument	Noise level meter				

Overview of the test machine





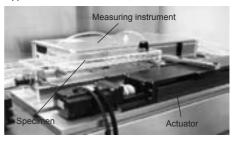
Noise level comparison

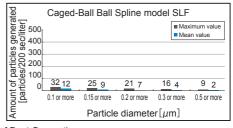
[Conditions]

Model tested	SLF50CL+350LP/ LBS50CL+350LP
Maximum speed	30m/min
Acceleration	2.84m/s ²
Stroke	200mm
Amount of air supplied	1ℓ/200sec
Lubricant	THK AFE-CA Grease
Equipment using the product	Particle counter

Ball Spline model LBS (full-ball type) Ball Spline model LBS (full-ball type)

Appearance of the test machine





Data on Comparison of Dust Generation

High Torque Caged Ball Spline

[Long-term Maintenance-free Operation]

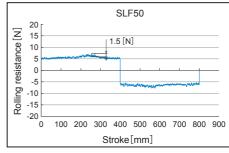
Models SLS/SLF adopt the caged-ball technology to substantially increase the grease retention, thus achieving long-term maintenance-free operation.

[Smooth Motion (Small Rolling Fluctuation)]

Models SLS/SLF adopt the caged-ball technology and a new circulation method, thus achieving stable and smooth motion with small rolling fluctuation.

[Conditions]

Model tested	SLF50				
Speed	10mm/sec				
Applied load	Medium preload(CM)				
Lubricant	THK AFB-LF Grease				



Rolling resistance test

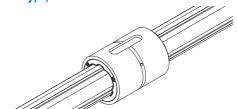
Types and Features

[Types of Spline Nuts]

Cylindrical Type Ball Spline Model SLS (Medium Load Type)

The circumference of the spline nut is shaped in a straight cylinder.

Using a key, this model can be secured to the housing, or transmit a torque.



Specification Table⇒A3-42

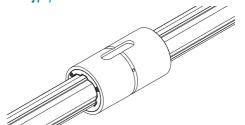
Specification Table⇒A3-42

Specification Table⇒A3-44

Cylindrical Type Ball Spline Model SLS-L (Heavy Load Type)

A heavy-load type with the same outer diameter as model SLS and a longer spline nut.

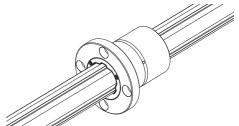
It is optimal in cases where a large torque is applied in a small space, and in cases where an overhang load or moment is applied.



Flanged Type Ball Spline Model SLF

The housing can be secured with bolts on models equipped with a flange.

This model is easily assembled and can accommodate a shorter housing compared to models with housing secured by a key.



High Torque Caged Ball Spline

[Types of Spline Shafts]

Precision Solid Spline Shaft (Standard Type)

The spline shaft is cold-drawn and its raceway is precision ground. It is used in combination with a spline nut.



Special Spline Shaft

THK manufactures a spline shaft with thicker ends or thicker middle area through special processing at your request.



Hollow Spline Shaft (Type K)

A drawn, hollow spline shaft is available for requirements such as piping, wiring, air-vent and weight reduction.



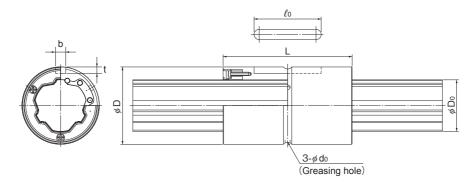
Housing Inner-diameter Tolerance

When fi tting the spline nut to the housing, transition fi t is normally recommended. If the accuracy of the Ball Spline does not need to be very high, clearance fi tting is also acceptable.

Table1 Housing Inner-diameter Tolerance

Housing Inner-diameter	General conditions	H7
Tolerance	When clearance needs to be small	J6

Model SLS



		Spline nut dimensions											
Model No.	Outer	diameter	Le	Length		way dimens	Greasing hole						
Wiodel No.	D	Tolerance	L	Tolerance	b H8	t +0.1 0	lo	d₀					
SLS25 SLS25L	37	0	60 70		5	3	33	2					
SLS30 SLS30L	45	-0.016	70 80 90 100 100 112	-	-					7	4	41	3
SLS40 SLS40L	60	0		0	10	4.5	55	3					
SLS50 SLS50L	75	-0.019			-0.3	15	5	60	4				
SLS60 SLS60L	90							18	6	68	4		
SLS70 SLS70L	100	0 -0.022	110 135		18	6	68	4					
SLS80 SLS80L	120		140 155	0	20	7	80	5					
SLS100 SLS100L	140	0 -0.025	160 185	-0.4	28	9	93	5					

Model number coding



Model No. Symbol for clearance in the rotational direction (*2)

Symbol for standard hollow spline shaft (*4)
Accuracy symbol (*3)

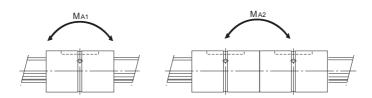
Contamination protection accessory symbol (*1) Overall spline shaft length (*5) (in mm)

Number of spline nuts on one shaft (no symbol for one nut)

(*1)See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-46. (*5) See A3-109.

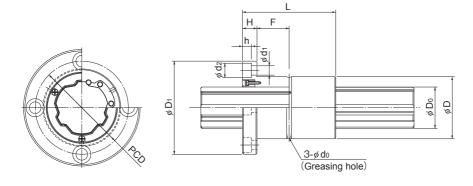






							O	
Basic torque rating		rating Basic load rating			rmissible nent	Mass		
C _⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _{A1} N-m	M _{A2} N-m	Spline Nut kg	Spline shaft kg/m	
219.9 261.9	306.8 394.5	18.2 21.7	22.5 29.0	136 220	851 1203	0.15 0.18	3.51	
366.5 416.4	513.3 616.0	25.4 28.9	31.5 37.8	233 330	1341 1803	0.30 0.34	5.05	
818.9 890.0	1135.4 1277.3	42.8 46.5	52.5 59.1	520 652	2801 3529	0.69 0.79	9.18	
1373.4 1571.2	1783.1 2165.2	57.6 65.9	66.2 80.4	687 996	4156 5349	1.30 1.47	14.45	
2506.7 2723.2	3321.0 3736.2	87.8 95.3	103.0 115.8	1452 1820	7733 9570	2.25 2.50	21.23	
2986.3 3708.4	3474.7 4738.2	89.7 111.4	92.5 126.1	1038 1867	6392 10135	2.13 2.71	28.57	
4664.6 5195.3	5477.4 6390.4	122.8 136.8	127.7 148.9	1739 2327	11482 14491	4.22 4.77	37.49	
8922.3 10424.4	10211.6 12764.6	188.2 219.8	190.7 238.4	3155 4816	19118 26463	5.20 6.22	58.97	

Model SLF



		Spline nut dimensions											
Model No.	Outer	diameter	L	ength	Flange	e diameter			Greasing hole				
	D	Tolerance	L	Tolerance	D ₁	Tolerance	Н	F	d₀	PCD			
SLF25	37	0	60		60		9	21	2	47			
SLF30	45	-0.016	70		70	0 -0.2	10	25	3	54			
SLF40	60		90	0	90		14	31	3	72			
SLF50	75	0 -0.019	100	-0.3	113		16	34	4	91			
SLF60	90		127		129	0	18	45.5	4	107			
SLF70	100	0	110		142	-0.3	20	47.5	4	117			
SLF80	120	-0.022	140	0	168		22	48	5	138			
SLF100	140	0 -0.025	160	-0.4	195	0 -0.4	25	55	5	162			

Model number coding

2 SLF50 UU CL +700L P K

Model No. Symbol for clearance in the rotational direction

Symbol for standard hollow spline shaft (*4)

Accuracy symbol (*3)

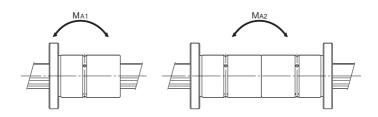
 $\begin{array}{ll} \text{Contamination protection} & \text{Overall spline shaft length (*5)} \\ \text{accessory symbol (*1)} & \text{(in mm)} \end{array}$

Number of spline nuts on one shaft (no symbol for one nut)

(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-46. (*5) See A3-109.







Offic. 1									
		Basic tore	Basic torque rating		Basic load rating		Static permissible moment		ass
	Mounting hole								
	$d_1 \times d_2 \times h$	C _⊤ N-m	C₀⊤ N-m	C kN	C₀ kN	M _{A1} N-m	M _{A2} N-m	Spline Nut kg	Spline shaft kg/m
	5.5×9.5×5.4	219.9	306.8	18.2	22.5	136	851	0.26	3.51
	6.6×11×6.5	366.5	513.3	25.4	31.5	233	1341	0.45	5.05
	9×14×8.6	818.9	1135.4	42.8	52.5	520	2801	1.06	9.18
	11×17.5×11	1373.4	1783.1	57.6	66.2	687	4156	1.90	14.45
	11×17.5×11	2506.7	3321.0	87.8	103.0	1452	7733	3.08	21.23
	14×20×13	2986.3	3474.7	89.7	92.5	1038	6392	3.25	28.57
	16×23×15.2	4664.6	5477.4	122.8	127.7	1739	11482	5.82	37.49
	18×26×17.5	8922.3	10211.6	188.2	190.7	3155	19118	7.66	58.97

Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (type K), as described on **A3-41**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table2 shows the sectional shape of a spline-shaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.

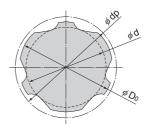


Table2 Sectional Shape of the Spline Shaft

Unit: mm

Nominal shaft diameter	25	30	40	50	60	70	80	100
Minor diameter φ d	21.6	25.8	35.2	44.4	54.0	62.8	71.3	90.0
Major diameter φ D₀ h7	25.0	30.0	40.0	50.0	60.0	60.0	80.0	100.0
Ball center-to-center diameter ϕ dp	25.2	30.2	40.6	50.6	61.0	71.0	80.8	101.2
Mass(kg/m)	3.51	5.05	9.18	14.45	21.23	28.57	37.49	58.97

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table3 shows the hole shape of the standard hollow type spline shaft. Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.

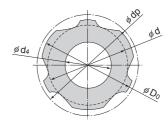


Table3 Sectional Shape of the Standard Hollow Type Spline Shaft

Unit: mm

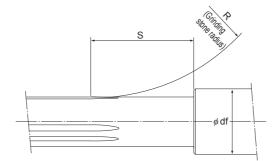
Nominal shaft diameter	25	30	40	50	60	70	80	100
Minor diameter φ d	21.6	25.8	35.2	44.4	54.0	62.8	71.3	90.0
Major diameter	25.0	30.0	40.0	50.0	60.0	60.0	80.0	100.0
Ball center-to-center diameter ϕ dp	25.2	30.2	40.6	50.6	61.0	71.0	80.8	101.2
Hole diameter(12	16	22	25	32	_	52.5	67.5
Mass(kg/m)	2.62	3.47	6.19	10.59	14.90	_	20.48	30.85

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

High Torque Caged Ball Spline

[Length of Imperfect Area of a Special Spline Shaft]

If the middle area or the end of a spline shaft is to be thicker than the minor diameter (ϕ d), an imperfect spline area is required to secure a recess for grinding. For the relationship between the length of the incomplete section (S) and the flange diameter (ϕ df), contact THK.



Accessories

Ball Spline models SLS and SLS-L are provided with a standard key as indicated in Table4.

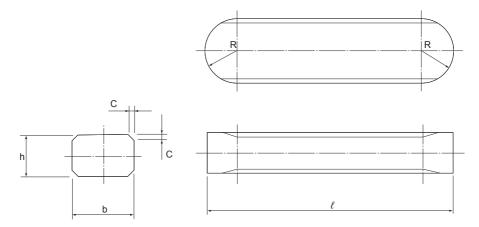


Table4 Standard Keys for Models SLS and SLS-L

Unit: mm

Nominal shaft		Width b		Height h		Length ℓ	R	С
diameter		Tolerance(p7)		Tolerance(h9)		Tolerance(h12)	K	C
SLS 25 SLS 25L	5	+0.024 +0.012	5	0 -0.030	33	0	2.5	0.5
SLS 30 SLS 30L	7	+0.030	7		41	-0.250	3.5	0.5
SLS 40 SLS 40L	10	+0.015	8	0 -0.036	55		5	0.8
SLS 50 SLS 50L	15		10		60	0	7.5	0.6
SLS 60 SLS 60L SLS 70 SLS 70L	18	+0.036 +0.018	12	0	68	-0.300	9	1.0
SLS 80 SLS 80L	20	+0.043	13	-0.043	80	80 0 -0.350		1.2
SLS 100 SLS 100L	28	+0.022	18		93	0 -0.400	14	

High Torque Type Ball Spline Models LBS, LBST, LBF, LBR and LBH Seal Seal Spline shaft Spline nut Snap ring

Fig.1 Structure of High Torque Type Ball Spline Model LBS

Point of Selection	A3-6
Point of Design	A3-111
Options	△ 3-113
Model No.	△3-115
Precautions on Use	△3-116
Accessories for Lubrication	A24-1
Mounting Procedure and Maintenance	■ 3-30
Cross-sectional Characteristics of the Spline Shaft	A3-17
Equivalent factor	A3-26
Clearance in the Rotation Direction	A3-29
Accuracy Standards	A3-33
Maximum Manufacturing Length by Accuracy	▲3-109

High Torque Type Ball Spline

Structure and Features

With the high torque type Ball Spline, the spline shaft has three crests positioned equidistantly at 120°, and along both sides of each crest, two rows of balls (six rows in total) are arranged so as to hold the crest, as shown in Fig.1.

The raceways are precision ground into R-shaped grooves whose diameters are approximate to the ball diameter. When a torque is generated from the spline shaft or the spline nut, the three rows of balls on the load-bearing side evenly receive the torque, and the center of rotation is automatically determined. When the rotation reverses, the remaining three rows of balls on the unloaded side receive the torque.

The rows of balls are held in a retainer incorporated in the spline nut so that they smoothly roll and circulate. With this design, balls will not fall even if the spline shaft is removed from the nut.

[No Angular Backlash]

With the high torque type Ball Spline, a single spline nut provides a preload to eliminate angular backlash and increase the rigidity.

Unlike conventional ball splines with circular-arc groove or Gothic-arch groove, the high torque type Ball Spline eliminates the need for twisting two spline nuts to provide a preload, thus allowing compact design to be achieved easily.

[High Rigidity and Accurate Positioning]

Since this model has a large contact angle and provides a preload from a single spline nut, the initial displacement is minimal and high rigidity and high positioning accuracy are achieved.

[High-speed Motion, High-speed Rotation]

Adoption of a structure with high grease retention and a rigid retainer enables the ball spline to operate over a long period with grease lubrication even in high-speed straight motion. Since the distance in the radius direction is almost uniform between the loaded balls and the unloaded balls, the balls are little affected by the centrifugal force and smooth straight motion is achieved even during high-speed rotation.

[Compact Design]

Unlike conventional ball splines, unloaded balls do not circulate on the outer surface of the spline nut with this model. As a result, the outer diameter of the spline nut is reduced and a space-saving and compact design is achieved.

[Ball Retaining Type]

Use of a retainer prevents the balls from falling even if the spline shaft is pulled out of the spline nut.

[Can be Used as a Linear Bushing for Heavy Loads]

Since the raceways are machined into R grooves whose diameter is almost equal to the ball diameter, the contact area of the ball is large and the load capacity is large also in the radial direction.

[Double, Parallel Shafts can be Replaced with a Single Shaft]

Since a single shaft is capable of receiving a load in the torque direction and the radial direction, double shafts in parallel configuration can be replaced with a single-shaft configuration. This allows easy installation and achieves space-saving design.

Applications

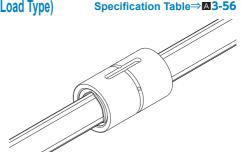
The high torque type Ball Spline is a reliable straight motion system used in a wide array of applications such as the columns and arms of industrial robot, automatic loader, transfer machine, automatic conveyance system, tire forming machine, spindle of spot welding machine, guide shaft of high-speed automatic coating machine, riveting machine, wire winder, work head of electric discharge machine, spindle drive shaft of grinding machine, speed gears and precision indexing shaft.

Types and Features

[Types of Spline Nuts]

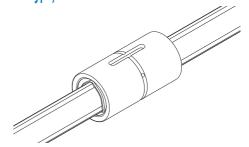
Cylindrical Type Ball Spline Model LBS (Medium Load Type)

The most compact type with a straight cylindrical spline nut. When transmitting a torque, a key is driven into the body. The outer surface of the spline nut is provided with anti-carbonation treatment.



Cylindrical Type Ball Spline Model LBST (Heavy Load Type)

A heavy load type that has the same spline nut diameter as model LBS, but has a longer spline nut length. It is optimal for locations where the space is small, a large torque is applied, and an overhang load or moment load is applied.

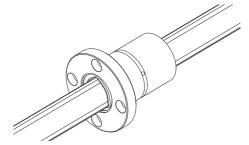


Specification Table⇒A3-60

Specification Table⇒A3-62

Flanged Type Ball Spline Model LBF

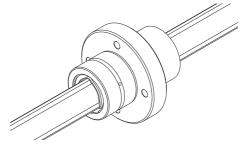
The spline nut can be attached to the housing via the flange, making assembly simple. It is optimal for locations where the housing may be deformed if a keyway is machined on its surface, and where the housing width is small.



Flanged Type Ball Spline Model LBR

Specification Table⇒A3-64

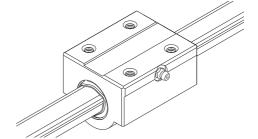
Based on the heavy load type model LBST, this model has a flange in the central area, making itself optimal for locations under a moment load such as arms of industrial robots.



Rectangular Type Ball Spline Model LBH

Specification Table⇒A3-66

Its rigid rectangular spline nut does not require a housing and can be directly mounted on the machine body. Thus, a compact, highly rigid linear guide system is achieved.



[Types of Spline Shafts]

Precision Solid Spline Shaft (Standard Type)

The spline shaft is cold-drawn and its raceway is precision ground. It is used in combination with a spline nut.



Special Spline Shaft

THK manufactures a spline shaft with thicker ends or thicker middle area through special processing at your request.



Hollow Spline Shaft (Type K)

A drawn, hollow spline shaft is available for requirements such as piping, wiring, air-vent and weight reduction.



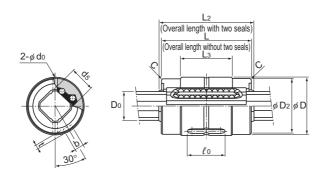
Housing Inner-diameter Tolerance

When fitting the spline nut to the housing, transition fit is normally recommended. If the accuracy of the Ball Spline does not need to be very high, clearance fitting is also acceptable.

Table1 Housing Inner-diameter Tolerance

Housing Inner-diameter	General conditions	H7
Tolerance	When clearance needs to be small	J6

Miniature Ball Spline

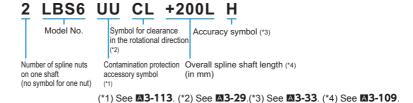


Models LBS6 and 8

	Spline nut dimensions										
Model No.	Outer diameter		Length					Keyway dimensions			
	D	Tolerance	L	Tolerance	L ₂	L ₃	D ₂	b H8	t +0.1 0	ℓ_0	С
LBS 6	12	0	20		20.8	11	11.5	2	0.8	10	0.3
LBS 8	16	-0.011	25	0 -0.2	26.4	14.5	15.5	2.5	1.2	12.5	0.3
LBS 10	19	0 -0.013	30	3.2	_	_		3	1.5	17	0.3

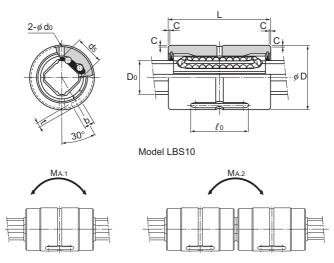
Note) Models LBS6 and 8 are of end cap type. Keep the end caps of models LBS6 and 8 from impact. THK does not offer a high temperature type of miniature Ball Spline.

Model number coding





High Torque Type Ball Spline



I Init: mm

Onic thin											
		shaft iameter	Basic torque rating			Basic load rating (radial)		rmissible nent	Mass		
Greasing hole											
d₀	Do	d₅	C _⊤ N-m	С₀т N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut g	Spline shaft kg/m	
1.2	6	5.3	1.53	2.41	0.637	0.785	2.2	19.4	6.6	0.22	
1.2	8	7.3	4.07	6.16	1.18	1.42	5.1	39.6	15.4	0.42	
1.5	10	8.3	7.02	10.4	1.62	1.96	8.1	67.6	36.7	0.55	

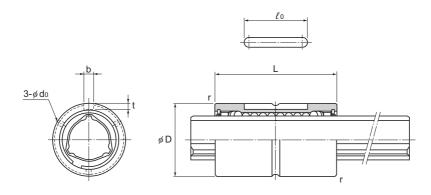
Note) ** MA1 indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the

**MA2 indicates the permissible moment value in the axial direction when two spline nuts in close contact with each other are used, as shown in the figure above.

(Single spline nut configuration is not stable in accuracy. We recommend using two spline nuts in close contact with

For details on the maximum lengths of ball spline shafts by accuracy, please see **43-109**.

Model LBS (Medium Load Type)



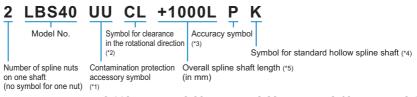
				Spline nut dim	ensions						
Model No.	Out	er diameter		Length		Keyway dimensions			Greasing hole		
iviouel No.	D	Tolerance	L	Tolerance	b H8	t +0.1 0	ℓ_{0}	r	d₀		
LBS 15	23	0 -0.013	40	0	3.5	2	20	0.5	2		
○● LBS 20	30		50	-0.2	4	2.5	26	0.5	2		
○● LBS 25	37	0 -0.016	60		5	3	33	0.5	2		
○● LBS 30	45		70		7	4	41	1	3		
○● LBS 40	60	0	90	0 -0.3	3 10	4.5	55	1	3		
○● LBS 50	75	-0.019	100		15	5	60	1.5	4		
○● LBS 70	100	0	110		18	6	68	2	4		
○● LBS 85	120	-0.022	140	0	20	7	80	2.5	5		
○● LBS 100	140	0 -0.025	160	-0.4	28	9	93	3	5		

Note) \bigcirc : indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C). (Example) LBS20 A CL+500L H

____ High temperature symbol

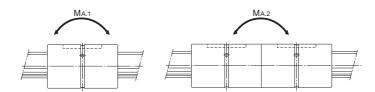
●: indicates model numbers for which felt seal types are available (see ▲3-113). A felt seal cannot be attached to Ball Spline models using metal retainer.

Model number coding



(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-69. (*5) See A3-109.





Basic tord	que rating	Basic load ra	ating (radial)	Static permis	sible moment	Ma	ass				
C _⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut kg	Spline shaft kg/m				
30.4	74.5	4.4	8.4	25.4	185	0.06	1				
74.5	160	7.8	14.9	60.2	408	0.14	1.8				
154	307	13	23.5	118	760	0.25	2.7				
273	538	19.3	33.8	203	1270	0.44	3.8				
599	1140	31.9	53.4	387	2640	1	6.8				
1100	1940	46.6	73	594	4050	1.7	10.6				
2190	3800	66.4	102	895	6530	3.1	21.3				
3620	6360	90.5	141	2000	12600	5.5	32				
5910	12600	126	237	3460	20600	9.5	45				

Note) ** MA.1 indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above.

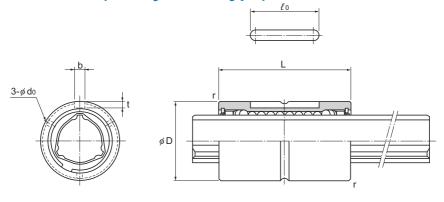
**M_{A2} indicates the permissible moment value in the axial direction when two spline nuts in close contact with each

For details on the maximum lengths of ball spline shafts by accuracy, please see **A3-109**.

other are used, as shown in the figure above.

⁽Single LBS-unit configuration is not stable in accuracy. We recommend using a single LBST unit or two LBS units in close contact with each other.)

Model LBST (Heavy Load Type)



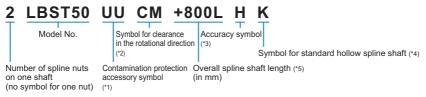
					Spline nut dim	ensions						
	/lodel No.	Out	er diameter		Length		Keyway dimensions			Greasing hole		
10	nodel No.	D	Tolerance	L	Tolerance	b H8	t +0.1 0	ℓ_{0}	r	d₀		
0	LBST 20	30	0	60	0 -0.2	4	2.5	26	0.5	2		
	LBST 25	37	-0.016	70		5	3	33	0.5	2		
	LBST 30	45		80	0 -0.3	7	4	41	1	3		
	LBST 40	60	0	100		10	4.5	55	1	3		
	LBST 50	75	-0.019	112		15	5	60	1.5	4		
0	LBST 60	90		127		18	6	68	1.5	4		
	LBST 70	100	0 -0.022	135		18	6	68	2	4		
	LBST 85	120	0.022	155	0	20	7	80	2.5	5		
0	LBST 100	140	0	175	-0.4	28	9	93	3	5		
0	LBST 120	160	-0.025	200	0	28	9	123	3.5	6		
0	LBST 150	205	0 -0.029	250	-0.5	32	10	157	3.5	6		

Note) \bigcirc : indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C). (Example) LBST25 A CM+400L H

High temperature symbol

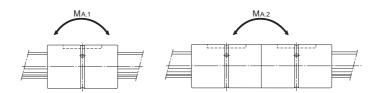
●: indicates model numbers for which felt seal types are available (see ▲3-113). A felt seal cannot be attached to Ball Spline models using metal retainer.

Model number coding



(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-69. (*5) See A3-109.





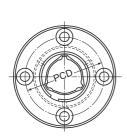
Office from											
Basic tord	que rating	Basic load ra	ating (radial)	Static permis	sible moment	Ma	ass				
C _⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut kg	Spline shaft kg/m				
90.2	213	9.4	20.1	103	632	0.17	1.8				
176	381	14.9	28.7	171	1060	0.29	2.7				
312	657	22.5	41.4	295	1740	0.5	3.8				
696	1420	37.1	66.9	586	3540	1.1	6.8				
1290	2500	55.1	94.1	941	5610	1.9	10.6				
1870	3830	66.2	121	1300	8280	3.3	15.6				
3000	6090	90.8	164	2080	11800	3.8	21.3				
4740	9550	119	213	3180	17300	6.1	32				
6460	14400	137	271	4410	25400	10.4	45				
8380	19400	148	306	5490	32400	12.9	69.5				
13900	32200	196	405	8060	55400	28	116.6				

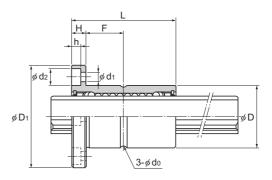
Note) **MA+ indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above.

**MA- indicates the permissible moment value in the axial direction when two spline nuts in close contact with each other are used, as shown in the figure above.

For details on the maximum lengths of ball spline shafts by accuracy, please see 3-109.

Model LBF (Medium Load Type)





				Splii	ne nut di	mensions					
Model No.	Outer	diameter	Length		Flange diameter			Greasing hole			
	D	Tolerance	L	Tolerance	D ₁	Tolerance	Н	F	d₀	PCD	
LBF 15	23	0 -0.013	40	0	43		7	13	2	32	
○● LBF 20	30	0 -0.016	50	-0.2	49		7	18	2	38	
○● LBF 25	37		60		60	0 -0.2	9	21	2	47	
○● LBF 30	45		70		70		10	25	3	54	
○● LBF 40	57		90	0	90		14	31	3	70	
○● LBF 50	70	0 -0.019	100	-0.3	108		16	34	4	86	
O LBF 60	85		127		124	0	18	45.5	4	102	
○● LBF 70	95	0	110		142	-0.3	20	35	4	117	
○● LBF 85	115	-0.022	140	0	168		22	48	5	138	
○● LBF 100	135	0	160	-0.4	195	0	25	55	5	162	

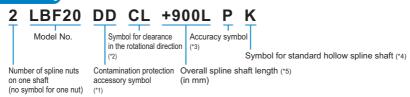
Note) \bigcirc : indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C). (Example) LBF20 A CL+500L H

Example) LBF20 A CL+500L H

High temperature symbol

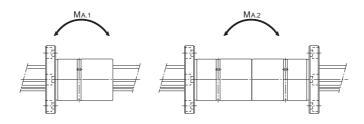
●: indicates model numbers for which felt seal types are available (see **△3-113**). A felt seal cannot be attached to Ball Spline models using metal retainer.

Model number coding



(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-69. (*5) See A3-109.





Onic. III										
		Basic tord	que rating		Basic load rating (radial)		rmissible nent	Mass		
	Mounting hole									
	$d_1 \times d_2 \times h$	C _⊤ N-m	С₀т N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut kg	Spline shaft kg/m	
	4.5×8×4.4	30.4	74.5	4.4	8.4	25.4	185	0.11	1	
	4.5×8×4.4	74.5	160	7.8	14.9	60.2	408	0.2	1.8	
	5.5×9.5×5.4	154	307	13	23.5	118	760	0.36	2.7	
	6.6×11×6.5	273	538	19.3	33.8	203	1270	0.6	3.8	
	9×14×8.6	599	1140	31.9	53.4	387	2640	1.2	6.8	
	11×17.5×11	1100	1940	46.6	73	594	4050	1.9	10.6	
	11×17.5×11	1870	3830	66.2	121	1300	8280	3.5	15.6	
	14×20×13	2190	3800	66.4	102	895	6530	3.6	21.3	
	16×23×15.2	3620	6360	90.5	141	2000	12600	6.2	32	
	18×26×17.5	5910	12600	126	237	3460	20600	11	45	

Note) **M_{A.1} indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the

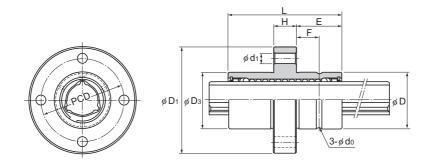
For details on the maximum lengths of ball spline shafts by accuracy, please see **A3-109**.

figure above.

**M_{x,2} indicates the permissible moment value in the axial direction when two spline nuts in close contact with each other are used, as shown in the figure above.

⁽Single spline nut configuration is not stable in accuracy. We recommend using two spline nuts in close contact with each other.)

Model LBR



				Spline	nut dimension	ons			
Model No.	Oute	r diameter	Outerdiameter	Length		Flangediameter			
	D	Tolerance	D₃	L	Tolerance	D ₁	Н	Е	PCD
LBR 15	25	0 -0.013	25.35	40	0	45.4	9	15.5	34
○● LBR 20	30		30.35	60	-0.2	56.4	12	24	44
○● LBR 25	40	0 -0.016	40.35	70	0	70.4	14	28	54
○● LBR 30	45	-0.010	45.4	80		75.4	16	32	61
○● LBR 40	60	0	60.4	100		96.4	18	41	78
○● LBR 50	75	-0.019	75.4	112	-0.3	112.4	20	46	94
O LBR 60	90		90.5	127		134.5	22	52.5	112
○● LBR 70	95	0 -0.022	95.6	135		140.6	24	55.5	117
○● LBR 85	120	0.022	120.6	155	0	170.6	26	64.5	146
○● LBR 100	140	0 -0.025	140.6	175	-0.4	198.6	34	70.5	170

Note) \bigcirc : indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C). (Example) LBR40 A CM+600L H

●: indicates model numbers for which felt seal types are available (see **△3-113**). A felt seal cannot be attached to Ball Spline models using metal retainer.

— High temperature symbol

Model number coding

2 LBR30 UU CM +700L H K

Model No.

Symbol for clearance in the rotational direction (*2)

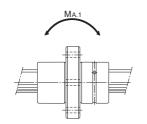
Accuracy symbol (*3)

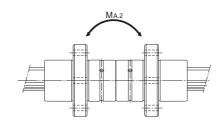
Symbol for standard hollow spline shaft (*4)

Number of spline nuts Contamination protection Overall spline shaft length (*5) on one shaft (no symbol for one nut) (*1)

(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-69. (*5) See A3-109.







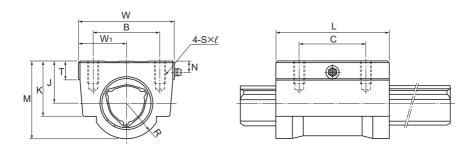
				Basic tord	que rating	Basic loa (rac	ad rating lial)	Static pe mon	rmissible nent	Mass			
	Mounting hole		Greasing hole										
	d₁	F	d₀	C _⊤ N-m	С₀т N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut kg	Spline shaft kg/m		
	4.5	7.5	2	30.4	74.5	4.4	8.4	25.4	185	0.14	1		
	5.5	12	2	90.2	213	9.4	20.1	103	632	0.33	1.8		
	5.5	14	2	176	381	14.9	28.7	171	1060	0.54	2.7		
	6.6	16	3	312	657	22.5	41.4	295	1740	0.9	3.8		
	9	20.5	3	696	1420	37.1	66.9	586	3540	1.7	6.8		
	11	23	4	1290	2500	55.1	94.1	941	5610	2.7	10.6		
	11	26	4	1870	3830	66.2	121	1300	8280	3.7	15.6		
	14	27	4	3000	6090	90.8	164	2080	11800	6	21.3		
	16	32	5	4740	9550	119	213	3180	17300	8.3	32		
	18	35	5	6460	14400	137	271	4410	25400	14.2	45		

Note) **MA: indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above.

**MA: indicates the permissible moment value in the axial direction when two spline nuts in close contact with each

other are used, as shown in the figure above. For details on the maximum lengths of ball spline shafts by accuracy, please see **3-109**.

Model LBH



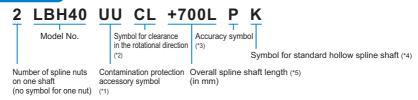
					;	Spline nut	dimension	S				
N	lodel No.	Height	Width	Length				J	W ₁			
		М	W	L	В	С	S×ℓ	±0.15	±0.15	Т	K	
0	LBH 15	29	34	43	26	26	M4×10	15	17	6	20	
	LBH 20	38	48	62	35	35	M6×12	20	24	7	26	
0	LBH 25	47.5	60	73	40	40	M8×16	25	30	8	33	
0	LBH 30	57	70	83	50	50	M8×16	30	35	10	39	
0	LBH 40	70	86	102	60	60	M10×20	38	43	15	50	
0	LBH 50	88	100	115	75	75	M12×25	48	50	18	63	

Note) O: indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C). (Example) LBH30 A CM+600L H

High temperature symbol

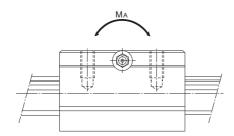
●: indicates model numbers for which felt seal types are available (see ▲3-113). A felt seal cannot be attached to Ball Spline models using metal retainer.

Model number coding



(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-69. (*5) See A3-109.

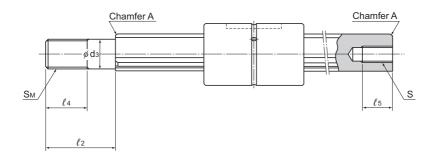




				Basic tord	que rating	Basic loa (rac	ad rating dial)	Static permissible moment Ma:		ass			
	R	N	Grease nipple	C _⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _A ** N-m	Spline Nut	Spline shaft kg/m			
	14	5	φ4 drive Nipple	30.4	74.5	4.4	8.4	25.4	0.23	1			
	18	7	A-M6F	90.2	213	9.4	20.1	103	0.58	1.8			
	22	6	A-M6F	176	381	14.9	28.7	171	1.1	2.7			
	26	8	A-M6F	312	657	22.5	41.4	295	1.73	3.8			
	32	10	A-M6F	696	1420	37.1	66.9	586	3.18	6.8			
	40	13.5	A-PT1/8	1290	2500	55.1	94.1	941	5.1	10.6			

Note) **MA indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above.
For details on the maximum lengths of ball spline shafts by accuracy, please see **\(\alpha 3-109**.

Model LBS with Recommended Shaft End Shape



Unit: mm

Model No.	d₃	Tolerance	ℓ_2	Sm	l ₄	S×ℓ₅
LBS 15	10	0 -0.015	23	M10×1.25	14	M6×10
LBS 20	14	0	30	M14×1.5	18	M8×15
LBS 25	18	-0.018	42	M18×1.5	25	M10×18
LBS 30	20	0	46	M20×1.5	27	M12×20
LBS 40	30	-0.021	70	M30×2	40	M18×30
LBS 50	36	0 -0.025	80	M36×3	46	M20×35

Note) For details of chamfer A, see **A3-70**.

High Torque Type Ball Spline

Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (type K), as described on **A3-55**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table2 shows the sectional shape of a spline shaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.

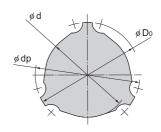


Table2 Sectional Shape of the Spline Shaft

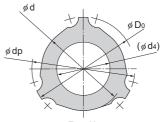
Unit: mm

Nominal shaft diameter	15	20	25	30	40	50	60	70	85	100	120	150
Minor diameter φd	11.7	15.3	19.5	22.5	31	39	46.5	54.5	67	81	101	130
Major diameter <i>φ</i> D₀	14.5	19.7	24.5	29.6	39.8	49.5	60	70	84	99	117	147
Ball center-to-center diameter ϕ dp	15	20	25	30	40	50	60	70	85	100	120	150
Mass (kg/m)	1	1.8	2.7	3.8	6.8	10.6	15.6	21.3	32	45	69.5	116.6

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table3 shows the hole shape of the standard hollow type spline shaft. Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.



Type K

Table3 Sectional Shape of the Standard Hollow Type Spline Shaft

Unit: mm

Nominal shaft diameter	20	25	30	40	50	60	70	85	100	120	150
Minor diameter φd	15.3	19.5	22.5	31	39	46.5	54.5	67	81	101	130
Major diameter φ D₀	19.7	24.5	29.6	39.8	49.5	60	70	84	99	117	147
Ball center-to-center diameter ϕ dp	20	25	30	40	50	60	70	85	100	120	150
Hole diameter (ϕ d ₄)	6	8	12	18	24	30	35	45	56	60	80
Mass (kg/m)	1.6	2.3	2.9	4.9	7	10	13.7	19.5	25.7	47.3	77.1

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

[Chamfering of the Spline Shaft Ends]

To facilitate the insertion of the spline shaft into a spline nut, the shaft ends are normally chamfered with dimensions as indicated below unless otherwise specified.

Chamfer A

If the spline shaft ends are stepped, tapped or drilled for specific use, they are machined with chamfer A dimensions indicated in Table4.

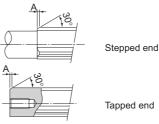


Fig.2 Chamfer A

Chamfer B

If either end of the spline shaft is not used, such as cantilever support, it is machined with chamfer B dimensions indicated in Table4.

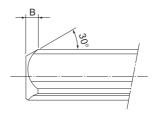


Fig.3 Chamfer B

Table4 Chamfer Dimensions of Spline Shaft Ends

Unit: mm

Nominal shaft diameter	15	20	25	30	40	50	60	70	85	100	120	150
Chamfer A	1	1	1.5	2.5	3	3.5	5	6.5	7	7	7.5	8
Chamfer B	3.5	4.5	5.5	7	8.5	10	13	15	16	17	17	18

Note) Spline shafts with nominal diameters 6, 8 and 10 are chamfered to C0.5.

High Torque Type Ball Spline

[Length of Imperfect Area of a Special Spline Shaft]

If the middle area or the end of a spline shaft is to be thicker than the minor diameter (ϕd) , an imperfect spline area is required to secure a recess for grinding. Table5 shows the relationship between the length of the incomplete section (S) and the flange diameter (ϕdf) .

(This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.)

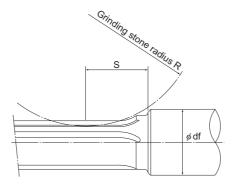


Table5 Length of Imperfect Spline Area: S

Unit: mm

														IIIL. IIIIII	
Flange diameter ϕ df Nominal shaft diameter	15	20	25	30	35	40	50	60	80	100	120	140	160	180	200
15	32	42	49	55	60	_	_	_	_	_	_	_	_	_	_
20		35	43	51	57	62	_	_	_	_		_		_	
25		_	51	64	74	82	97	_	_	_		_	_	_	
30	_	_	_	54	67	76	92	105	_	_	_	_	_	_	_
40	_	_	_	_	_	59	80	95	119	_	_	_	_	_	
50		_	_	_	_	_	63	83	110	131		_		l	_
60		_	_	_	_	_	_	66	100	123	140	_	_	ı	
70		_	_	_	_	_	_	_	89	115	134	150			_
85		_	_	_	_	_	_	_	61	98	122	140		I	
100		_	_		_	_	_	_	_	78	108	130	147		
120		_	_	_	_	_	_	_	_	_	81	111	133	150	
150		_	_	_	_	_	_	_	_	_		64	101	125	144

^{*}This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.

Accessories

Ball Spline models LBS and LBST are provided with a standard key as indicated in Table6.

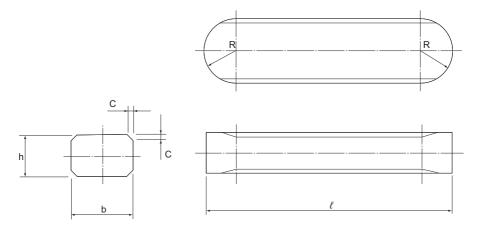


Table6 Standard Keys for Models LBS and LBST

Unit: mm

Nominal shaft		Width b		Height h		Length ℓ	R	С
diameter		Tolerance(p7)		Tolerance(h9)		Tolerance(h12)	K	
LBS 6	2	+0.016	1.3	0	10	0 -0.150	1	
LBS 8	2.5	+0.006	2	-0.025	12.5	0	1.25	0.3
LBS 10	3		2.5		17	-0.180	1.5	
LBS 15	3.5		3.5		20	0	1.75	
LBS 20 LBST 20	4	+0.024 +0.012	4	0 -0.030	26	-0.210	2	
LBS 25 LBST 25	5	+0.012	5	-0.030	33	0	2.5	0.5
LBS 30 LBST 30	7	+0.030	7		41	-0.250	3.5	
LBS 40 LBST 40	10	+0.015	8	0 -0.036	55		5	0.0
LBS 50 LBST 50	15	.0.020	10		60	0 -0.300	7.5	0.8
LBST 60 LBS 70 LBST 70	18	+0.036 +0.018	12		68	-0.300	9	
LBS 85 LBST 85	20	.0.042	13	0 -0.043	80	0 -0.350	14	1.2
LBS 100 LBST 100	28	+0.043 +0.022	18		93		14	
LBST 120	28		18		123	0 -0.400	14	
LBST 150	32	+0.051 +0.026	20	0 -0.052	157	0.700	16	2

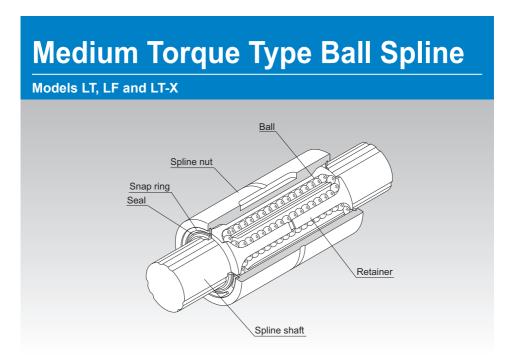


Fig.1 Structure of Medium Torque Type Ball Spline Model LT

Point of Selection	A3-6
Point of Design	A3-111
Options	A3-113
Model No.	A3-115
Precautions on Use	A3-116
Accessories for Lubrication	A24-1
Mounting Procedure and Maintenance	■3-30
Cross-sectional Characteristics of the Spline Shaft	△3-17
Equivalent factor	△3-26
Clearance in the Rotation Direction	△3-29
Accuracy Standards	A3-33
Maximum Manufacturing Length by Accuracy	▲3-109

Medium Torque Type Ball Spline

Structure and Features

With the medium torque type Ball Spline, the spline shaft has two to three crests on the circumference, and along both sides of each crest, two rows of balls (four or six rows in total) are arranged to hold the crest so that a reasonable preload is applied.

The rows of balls are held in a special resin retainer incorporated in the spline nut so that they smoothly roll and circulate. With this design, balls will not fall even if the nut is removed from the spline shaft.

[Large Load Capacity]

The raceways are formed into circular-arc grooves approximate to the ball curvature and ensure angular contact. Thus, this model has a large load capacity in the radial and torque directions.

[No Angular Backlash]

Two rows of balls facing one another hold a crest, formed on the circumference of the spline nut, at a contact angle of 20° to provide a preload in an angular-contact structure. This eliminates an angular backlash in the rotational direction and increases the rigidity.

[High Rigidity]

Since the contact angle is large and an appropriate preload is given, high rigidity against torque and moment is achieved.

[Ball Retaining Type]

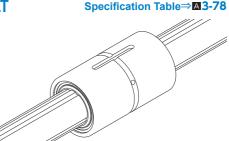
Use of a retainer prevents the balls from falling even if the spline shaft is pulled out of the spline nut. (except for models LT4 and 5)

Types and Features

[Types of Spline Nuts]

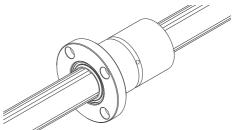
Cylindrical Type Ball Spline Model LT

The most compact type with a straight cylindrical spline nut. When transmitting a torque, a key is driven into the body.



Flanged Type Ball Spline Model LF

The spline nut can be attached to the housing via the flange, making assembly simple. It is optimal for locations where the housing may be deformed if a keyway is machined on its surface, and where the housing width is small.



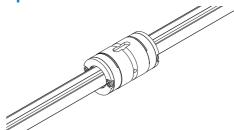
Specification Table⇒A3-80

Specification Table⇒A3-82

The model LT-X miniature ball spline

This model is much more compact than the standard LT.

The length and external diameter of the LT-X ball spline's outer cylinder are the same as those of an LM-series linear bush, enabling the user to replace a linear bush with the LT-X.



[Types of Spline Shafts]

Precision Solid Spline Shaft (Standard Type)

The raceway of the spline shaft is precision ground. It is used in combination with a spline nut.



Ball Spline

Medium Torque Type Ball Spline

Special Spline Shaft

THK manufactures a spline shaft with thicker ends or thicker middle area through special processing at your request.



Hollow Spline Shaft (Type K)

A drawn, hollow spline shaft is available for requirements such as piping, wiring, air-vent and weight reduction.



Hollow Spline Shaft (Type N)

A drawn, hollow spline shaft is available for requirements such as piping, wiring, air-vent and weight reduction.



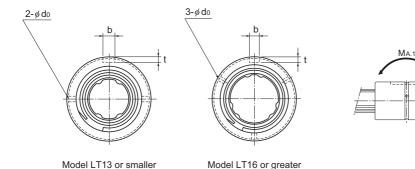
Housing Inner-diameter Tolerance

When fitting the spline nut to the housing, transition fit is normally recommended. If the accuracy of the Ball Spline does not need to be very high, clearance fitting is also acceptable.

Table1 Housing Inner-diameter Tolerance

Housing Inner-diameter	General conditions	H7
Tolerance	When clearance needs to be small	J6

Model LT



				Sp	line nut dim	ensions			
Model No.	Oute	er diameter	ı	Length		Keyway d	imensions		Greasing hole
	D	Tolerance	L	Tolerance	b H8	t +0.1 0	ℓ_0	r	d₀
Note) LT 4	10	0 -0.009	16		2	1.2	6	0.5	_
Note) LT 5	12	0	20		2.5	1.2	8	0.5	_
LT 6	14	0 -0.011	25		2.5	1.2	10.5	0.5	1
LT 8	16	-0.011	25	0 -0.2	2.5	1.2	10.5	0.5	1.5
LT 10	21	0	33	-0.2	3	1.5	13	0.5	1.5
LT 13	24	-0.013	36		3	1.5	15	0.5	1.5
O LT 16	31	-0.010	50		3.5	2	17.5	0.5	2
O LT 20	35	0	63		4	2.5	29	0.5	2
O LT 25	42	_0.016	71		4	2.5	36	0.5	3
O LT 30	47	=0.010	80	0	4	2.5	42	0.5	3
O LT 40	64	0	100	-0.3	6	3.5	52	0.5	4
O LT 50	80	-0.019	125		8	4	58	1	4
O LT 60	90	0	140		12	5	67	1	5
O LT 80	120	-0.022	160	0	16	6	76	2	5
O LT 100	150	0 -0.025	185	-0.4	20	7	110	2.5	5

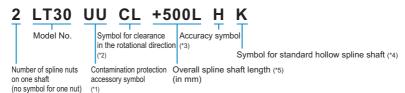
Note) Models LT4 and 5 do not have a retainer. Do not remove the shaft from the spline nut. (It will cause balls to fall off.)

C: indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C).

(Example) LT20 A CL+500L H

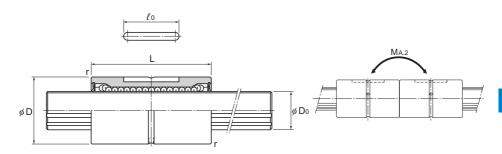
— High temperature symbol

Model number coding



(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-85. (*5) See A3-109.





Unit: mm

									Unit: mm
Spline shaft diameter	Rows of balls	Basic tord	que rating	Basic Loa	ad Rating	Static pe mon	rmissible nent	Ма	iss
D₀ h7		C _⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut g	Spline shaft kg/m
4	4	0.59	0.78	0.44	0.61	0.88	6.4	5.2	0.1
5	4	0.88	1.37	0.66	0.88	1.5	11.6	9.1	0.15
6	4	0.98	1.96	1.18	2.16	4.9	36.3	17	0.23
8	4	1.96	2.94	1.47	2.55	5.9	44.1	18	0.4
10	4	3.92	7.84	2.84	4.9	15.7	98	50	0.62
13	4	5.88	10.8	3.53	5.78	19.6	138	55	1.1
16	6	31.4	34.3	7.06	12.6	67.6	393	165	1.6
20	6	56.9	55.9	10.2	17.8	118	700	225	2.5
25	6	105	103	15.2	25.8	210	1140	335	3.9
30	6	171	148	20.5	34	290	1710	375	5.6
40	6	419	377	37.8	60.5	687	3760	1000	9.9
50	6	842	769	60.9	94.5	1340	7350	1950	15.5
60	6	1220	1040	73.5	111.7	1600	9990	2500	22.3
80	6	2310	1920	104.9	154.8	2510	16000	4680	39.6
100	6	3730	3010	136.2	195	3400	24000	9550	61.8

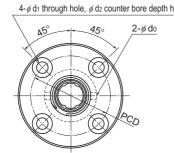
Note) **MA: indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the

(Single LT-unit configuration is not stable in accuracy. We recommend using two units in close contact with each other.) For details on the maximum lengths of ball spline shafts by accuracy, please see **M3-109**.

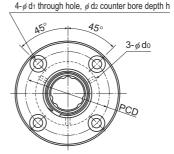
figure above.

* M_{x2} indicates the permissible moment value in the axial direction when two spline nuts in close contact with each other are used, as shown in the figure above.

Model LF







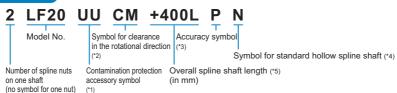
Model LF16 or greater

						Splin	e nut	dime	nsions	3				
Model No.	_	Outer ameter	Le	ength	Flange diameter						Greasing hole		Mounting hole	
	D	Tolerance	L	Tolerance	D ₁	Tolerance	olerance H F		С	r	d₀	PCD	$d_1 \times d_2 \times h$	
LF 6	14	0	25		30		5	7.5	0.5	0.5	1.5	22	3.4×6.5×3.3	
LF 8	16	-0.011	25		32		5	7.5	0.5	0.5	1.5	24	3.4×6.5×3.3	
LF 10	21		33	0	42		6	10.5	0.5	0.5	1.5	32	4.5×8×4.4	
LF 13	24	0 -0.013	36	-0.2	44		7	11	0.5	0.5	1.5	33	4.5×8×4.4	
O LF 16	31		50		51	51	0	7	18	0.5	0.5	2	40	4.5×8×4.4
O LF 20	35		63		58	-0.2	9	22.5	0.5	0.5	2	45	5.5×9.5×5.4	
O LF 25	42	0 -0.016	71		65		9	26.5	0.5	0.5	3	52	5.5×9.5×5.4	
O LF 30	47		80	0	75		10	30	0.5	0.5	3	60	6.6×11×6.5	
O LF 40	64	0	100	-0.3	100		14	36	1	0.5	4	82	9×14×8.6	
O LF 50	80	-0.019	125		124		16	46.5	1	1	4	102	11×17.5×11	

Note) O: indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C). (Example) LF30 A CL+700L H

____ High temperature symbol

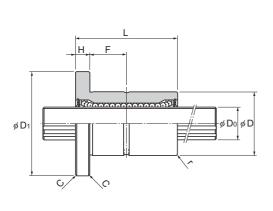
Model number coding

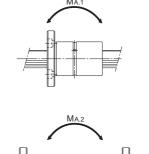


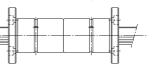
(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-85. (*5) See A3-109.



Medium Torque Type Ball Spline







Unit: mm

Spline shaft diameter	Rows of balls	Basic tord	que rating	Basic loa	ad rating	Static pe mon	rmissible nent	Mass		
D₀ h7		C _⊤ N-m	С₀т N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut g	Spline shaft kg/m	
6	4	0.98	1.96	1.18	2.16	4.9	36.3	35	0.23	
8	4	1.96	2.94	1.47	2.55	5.9	44.1	37	0.4	
10	4	3.92	7.84	2.84	4.9	15.7	98	90	0.62	
13	4	5.88	10.8	3.53	5.78	19.6	138	110	1.1	
16	6	31.4	34.3	7.06	12.6	67.6	393	230	1.6	
20	6	56.9	55.9	10.2	17.8	118	700	330	2.5	
25	6	105	103	15.2	25.8	210	1140	455	3.9	
30	6	171	148	20.5	34	290	1710	565	5.6	
40	6	419	377	37.8	60.5	687	3760	1460	9.9	
50	6	842	769	60.9	94.5	1340	7350	2760	15.5	

Note) **MA1 indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the

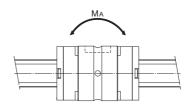
figure above.
**M_{A2} indicates the permissible moment value in the axial direction when two spline nuts in close contact with each

other are used, as shown in the figure above.

(Single LF-unit configuration is not stable in accuracy. We recommend using two units in close contact with each other.) For details on the maximum lengths of ball spline shafts by accuracy, please see A3-109.

Model LT-X





				Sp	line nut dim	ensions		
Model No.	Oute	er diameter	Length	า	Key	way dimens	ions	Greasing hole
	D	Tolerance	Ľ.	L ₁	b H8	t	ℓ_{0}	d₀
LT 4X	8	0 -0.009	14.4(12.0)	7.5	_	1	_	1
LT 5X LT 5XL	10	0 -0.009	15(13.6) 26(24.6)	7.3 18.3	2	1.2	4.7	1
LT 6X LT 6XL	12	0 -0.011	19(17.6) 30(28.6)	10.2 21.2	2	1.2	6	1

Note) External cylinder lengths (L) shown are for units equipped with the dust-prevention (UU) option. Values shown in parentheses are schematic lengths for units not equipped with the dust-prevention option.

Model number coding



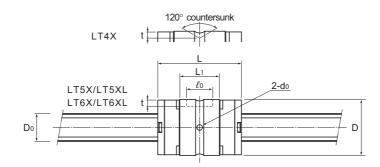
Model No. Contamination protection accessory symbol symbol for clearance in the rotational direction

Number of spline nuts

on one shaft (no symbol for one nut) Accuracy symbol (*3)

Overall spline shaft length (*4) (in mm)

(*1) See A3-113. (*2) See A3-29.(*3) See A3-33. (*4) See A3-109.

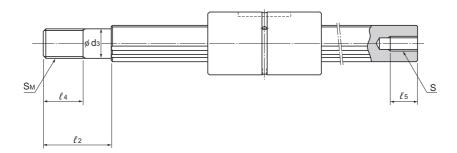


Unit: mm

Spline shaft diameter	Basic tord	que rating	Basic load rating		Static permissible moment	'		
D₀ h7	С _т N-m	С _{от} N-m	C C ₀ N		M _A N-m	Spline Nut g	Spline shaft kg/m	
4	0.49	0.82	420	700	0.84	2.4	0.10	
5	0.82 1.59	1.25 3.20	560 1090	850 2190	1.04 6.11	3.7 8.4	0.15	
6	1.73 2.81	2.77 5.54	980 1600	1580 3150	2.85 10.6	7.2 13.9	0.21	

Note) **M₁ indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the For details on the maximum lengths of ball spline shafts by accuracy, please see **\(\mathbb{A}3-109**.

Model LT with Recommended Shaft End Shape



Unit: mm

Model No.	d₃	Tolerance	ℓ_2	S _M	l4	S×ℓ₅
LT 6	5	0	12	M5×0.8	7	M2.5×4
LT 8	6	-0.012	14	M6×1	8	M3×5
LT 10	8	0	18	M8×1	11	M4×6
LT 13	10	-0.015	23	M10×1.25	14	M5×8
LT 16	14	0	30	M14×1.5	18	M6×10
LT 20	16	-0.018	38	M16×1.5	22	M8×15
LT 25	22	0	50	M22×1.5	28	M10×18
LT 30	27	-0.021	60	M27×2	34	M14×25
LT 40	36	0	80	M36×3	45	M18×30
LT 50	45	-0.025	100	M45×4.5	58	M22×40

Medium Torque Type Ball Spline

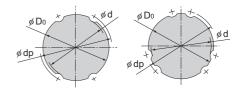
Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (types K and N), as described on **3-76**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table2 shows the sectional shape of a spline shaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.



Model LT13 or smaller Model LT16 or greater

Table2 Sectional Shape of the Spline Shaft

Unit: mm

Nominal shaft diameter	4	5	6	8	10	13	16	20	25	30	40	50	60	80	100
Minor diameter ød	3.5	4.5	5	7	8.5	11.5	14.5	18.5	23	28	37.5	46.5	56.5	75.5	95
Major diameter ϕ D₀ h7	4	5	6	8	10	13	16	20	25	30	40	50	60	80	100
Ball center-to-center diameter ϕ dp	4.6	5.7	7	9.3	11.5	14.8	17.8	22.1	27.6	33.2	44.2	55.2	66.3	87.9	109.5
Mass(kg/m)	0.1	0.15	0.23	0.4	0.62	1.1	1.6	2.5	3.9	5.6	9.9	15.5	22.3	39.6	61.8

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table3 shows the hole shape of the standard hollow type spline shaft (types K and N).

Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.

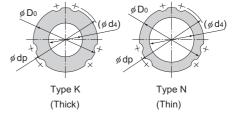


Table3 Sectional Shape of the Standard Hollow Type Spline Shaft

Unit: mm

Nominal shaft diameter		6	8	10	13	16	20	25	30	40	50	60	80	100
Major diameter <i>φ</i> D₀ h7		6	8	10	13	16	20	25	30	40	50	60	80	100
Ball center-to-center diameter ϕ dp		7	9.3	11.5	14.8	17.8	22.1	27.6	33.2	44.2	55.2	66.3	87.9	109.5
Time V	Hole diameter (ϕ d ₄)	2.5	3	4	5	7	10	12	16	22	25	32	52.5	67.5
Type K	Mass(kg/m)	0.2	0.35	0.52	0.95	1.3	1.8	3	4	6.9	11.6	16	22.6	33.7
Tuno N	Hole diameter (ϕ d ₄)	_	_	_	_	11	14	18	21	29	36	_	_	
Type N	Mass(kg/m)		_	_		0.8	1.3	1.9	2.8	4.7	7.4	_		

Note) The standard hollow type Spline Shaft is divided into types K and N. Indicate "K" or "N" at the end of the model number to distinguish between them when placing an order.

[Length of Imperfect Area of a Special Spline Shaft]

If the middle area or the end of a spline shaft is to be thicker than the minor diameter (ϕ d), an imperfect spline area is required to secure a recess for grinding. Table4 shows the relationship between the length of the incomplete section (S) and the flange diameter (ϕ df).

(This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.)

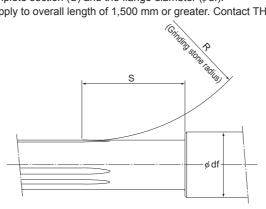


Table4 Length of Imperfect Spline Area: S Miniature type

				UI	III. IIIII
Flange diameter	1	5	6	8	10
Nominal shaft diameter	7	3	O	0	10
4	23	25	27	31	_
5	_	24	26	29	33

Standard Type Unit: mm

Flange diameter ϕ df	6	8	40	40	10	20	25	20	40	5 0	00	00	400	400	440	400
Nominal shaft diameter	Ь	ŏ	10	13	16	20	25	30	40	50	60	80	100	120	140	160
6	24	28	31	39	_	_	_	_	_	_	_	_	_	_	_	
8	_	25	29	35	41	_	_	_	_	_	_	_	_	_	_	_
10	_	_	26	31	38	45	_	_	_	_	_	_	_	_	_	
13	_		_	33	39	46	56		_		_	_	_	_	_	
16	_	_	_	_	36	47	58	67	_	_	_	_	_	_	_	_
20	_		_	_	_	37	50	60	76		_	_	_	_	_	_
25	_	_	_	_	_	_	38	51	72	88	_	_	_	_	_	_
30	_	_	_	_	_	_	_	40	62	80	95	_	_	_	_	_
40	_	_	_	_	_	_	_	_	42	63	81	107	_	_	_	
50	_	_	_	_	_	_	_	_	_	45	65	96	118	_	_	_
60	_		_		_	_	_		_	_	50	87	114	134		
80	_		_	_	_		_		_		_	53	89	115	135	
100	_	_	_	_	_	_	_	_	_	_	_	_	57	90	116	136

^{*}This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.

R

Accessories

Ball Spline model LT is provided with a standard key as indicated in Table5.

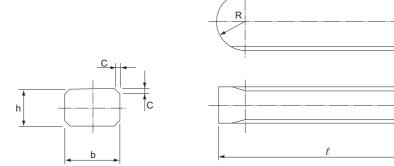


Table5 Standard Key for Model LT

Unit: mm

Tables Standard Rey Isl Model E1										
Nominal shaft		Width b		Height h		Length ℓ	R	С		
diameter		Tolerance(p7)		Tolerance(h9)		Tolerance(h12)	K			
LT 4	2		2		6	0 -0.120	1	0.3		
LT 5	2.5	+0.016	2.5	0	8	0 -0.150	1.25	0.5		
LT 6 LT 8	2.5	+0.006	2.5	-0.025	10.5		1.25			
LT 10	3		3		13	0 -0.180	1.5			
LT 13	3		3		15	-0.160	1.5			
LT 16	3.5		3.5		17.5		1.75			
LT 20	4	+0.024	4	0	29	0 -0.210	2	0.5		
LT 25	4	+0.012	4	-0.030	36	0	2			
LT 30	4		4		42	-0.250	2			
LT 40	6		6		52		3			
LT 50	8	+0.030 +0.015	7	0	58	0 -0.300	4			
LT 60	12	+0.036	8	-0.036	67	-0.300	6			
LT 80	16	+0.018	10		76		8	0.8		
LT 100	20	+0.043 +0.022	13	0 -0.043	110	0 -0.350	10	0.0		

Rotary Ball Spline

With Geared Type Models LBG and LBGT

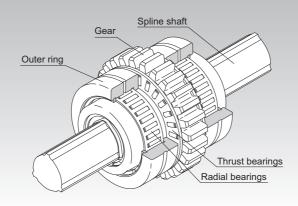


Fig.1 Structure of Rotary Ball Spline Model LBG

Point of Selection	A3-6
Point of Design	A3-111
Options	△3-113
Model No.	△3-115
Precautions on Use	△3-116
Accessories for Lubrication	A24-1
Mounting Procedure and Maintenance	B 3-30
Cross-sectional Characteristics of the Spline Shaft	A3-17
Equivalent factor	A 3-26
Clearance in the Rotation Direction	▲3-29
Accuracy Standards	A3-33
Maximum Manufacturing Length by Accuracy	△3-109

Rotary Ball Spline

Structure and Features

With the Rotary Ball Spline, the spline shaft has three crests, and along both sides of each crest, two rows of balls (six rows in total) are arranged to hold the crest so that a reasonable preload is applied.

These models are unit types based on model LBR, but have gear teeth on the flange circumference and radial and thrust bearings on the spline nut, all compactly integrated.

The rows of balls are held in a special resin retainer so that they smoothly roll and circulate. With this design, balls will not fall even if the spline shaft is removed.

[No Angular Backlash]

The spline shaft has three crests positioned equidistantly at 120° and along both sides of each crest, two rows of balls (six rows in total) are arranged so as to hold the crest at a contact angle of 45° and provide a preload. As a result, backlash in the rotational direction is eliminated and the rigidity is increased.

[Compact Design]

The spline nut is compactly integrated with radial and thrust bearings, allowing compact design to be achieved.

[High Rigidity]

Since the contact angle is large and an appropriate preload is given, high rigidity against torque and moment is achieved.

Use of needle bearings in the support unit achieves a rigid nut support strong against a radial load.

[Optimal for Torque Transmission with Spline Nut Drive]

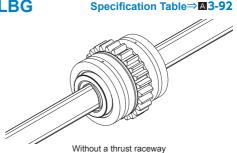
Since the support bearings allow a rigid nut support, these models are optimal for torque transmission with spline nut drive.

Types and Features

[Types of Spline Nuts]

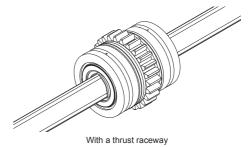
Ball Spline with Gears Model LBG

These models are unit types based on model LBR, but have gear teeth on the flange circumference and radial and thrust bearings on the spline nut, all compactly integrated. It is optimal for a torque transmission mechanism with spline nut drive.



Ball Spline with Gears Model LBGT

These models are unit types based on model LBR, but have gear teeth on the flange circumference and radial and thrust bearings on the spline nut, all compactly integrated. It is optimal for a torque transmission mechanism with spline nut drive.



Specification Table⇒A3-94

[Types of Spline Shafts]

For details, see 43-55.

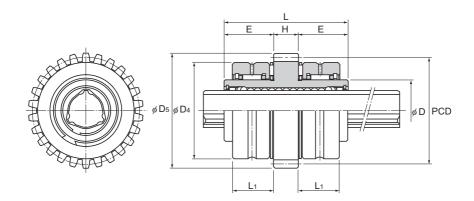
Housing Inner-diameter Tolerance

Table1 shows housing inner-diameter tolerance for models LBG and LBGT.

Table1 Housing Inner-diameter Tolerance

Housing Inner-diameter	General conditions	H7
Tolerance	When clearance needs to be small	J6

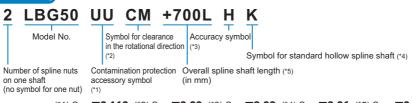
Model LBG



		Spline nut dimensions											
					opinie i	ilut dililelisit	J113						
Model No.		Spline nut outer diameter		Length		Outer diameter		Width					
	D	Tolerance	L	Tolerance	D ₄	Tolerance	L ₁	Tolerance	Н	E			
● LBG 20	30	0 -0.009	60		47	0 -0.011	20	0 -0.16	12	24			
● LBG 25	40	0	70	0	60	0	23	0	14	28			
● LBG 30	45	-0.011	80	-0.2	65	-0.013	27	-0.19	16	32			
● LBG 40	60	0	100		85		31		18	41			
● LBG 50	75	-0.013	112		100	0 -0.015	32	0	20	46			
LBG 60	90	0	127	0 -0.3	120		38	-0.25	22	52.5			
● LBG 85	120	-0.015	155	3.0	150	0 -0.025	40		26	64.5			

Note) : indicates model numbers for which felt seal types are available (see **A3-113**).

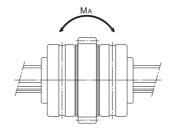
Model number coding



(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-96. (*5) See A3-109.







Unit: mm

	Gear spec	cifications*		Basic tord	que rating	Basic loa	ad rating	Static permissible moment	Mass	
Tip circle diameter D ₅	Standard pitch diameter PCD	Module m	Number of teeth z	C _τ N-m	С₀т N-m	C kN	C₀ kN	M _A ** N-m	Spline nut unit kg	Spline shaft kg/m
56	52	2	26	90.2	213	9.4	20.1	103	0.61	1.8
70	65	2.5	26	176	381	14.9	28.7	171	1.4	2.7
75	70	2.5	28	312	657	22.5	41.4	295	2.1	3.8
96	90	3	30	696	1420	37.1	66.9	586	3	6.8
111	105	3	35	1290	2500	55.1	94.1	941	4.1	10.6
133	126	3.5	36	1870	3830	66.2	121	1300	6.3	15.6
168	160	4	40	4740	9550	119	213	3180	11.8	32

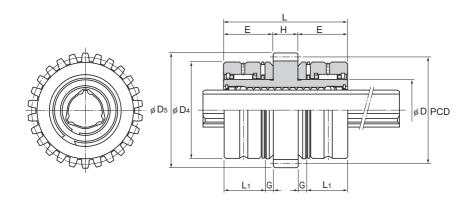
Note) *The gear specifications in the table represent the dimensions with maximum module.

figure above. For details on the maximum lengths of ball spline shafts by accuracy, please see **A3-109**.

Special gear types such as helical gear and worm gear can also be manufactured at your request.

**M. indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above.

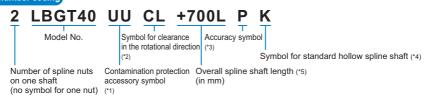
Model LBGT



		Spline nut dimensions											
Model No.		Spline nut outer diameter		Length		Outer diameter		Width					
	D	Tolerance	L	Tolerance	D ₄	Tolerance	L ₁	Tolerance	G	Н	E		
● LBGT 20	30	0 -0.009	60		47	0 -0.011	20	0 -0.16	4	12	24		
● LBGT 25	40	0	70	0	60	0	23	0	5	14	28		
● LBGT 30	45	-0.011	80	-0.2	65	-0.013	27	-0.19	5	16	32		
● LBGT 40	60	0	100		85		31		8	18	41		
● LBGT 50	75	-0.013	112		100	0 -0.015	32	0	10	20	46		
LBGT 60	90	0	127	0 -0.3	120		38	-0.25	12	22	52.5		
● LBGT 85	120	-0.015	155	3.0	150	0 -0.025	40		16	26	64.5		

Note) ●: indicates model numbers for which felt seal types are available (see **△3-113**).

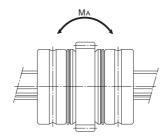
Model number coding



(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-96. (*5) See A3-109.







Unit: mm

										Offic. Hilli
	Gear spec	ifications*		Basic tord	que rating	Basic loa	ad rating	Static permissible moment	Ма	ss
Tip circle diameter D₅	Standard pitch diameter PCD	Module m	Number of teeth z	C _⊤ N-m	С₀т N-m	C kN	C₀ kN	M _A ** N-m	Spline nut unit kg	Spline shaft kg/m
56	52	2	26	90.2	213	9.4	20.1	103	0.67	1.8
70	65	2.5	26	176	381	14.9	28.7	171	1.5	2.7
75	70	2.5	28	312	657	22.5	41.4	295	2.2	3.8
96	90	3	30	696	1420	37.1	66.9	586	3.3	6.8
111	105	3	35	1290	2500	55.1	94.1	941	4.8	10.6
133	126	3.5	36	1870	3830	66.2	121	1300	7.2	15.6
168	160	4	40	4740	9550	119	213	3180	13.4	32

For details on the maximum lengths of ball spline shafts by accuracy, please see **A3-109**.

Note) *The gear specifications in the table represent the dimensions with maximum module.

Special gear types such as helical gear and worm gear can also be manufactured at your request.

**MA indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the

Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (type K), as described on **A3-55**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table2 shows the sectional shape of a spline shaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.

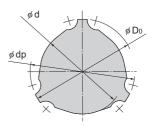


Table2 Sectional Shape of the Spline Shaft

Unit: mm

Nominal shaft diameter	20	25	30	40	50	60	85
Minor diameter ϕ d	15.3	19.5	22.5	31	39	46.5	67
Major diameter φ D₀	19.7	24.5	29.6	39.8	49.5	60	84
Ball center-to- center diameter	20	25	30	40	50	60	85
Mass (kg/m)	1.8	2.7	3.8	6.8	10.6	15.6	32

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table3 shows the hole shape of the standard hollow type spline shaft (type K) for models LBG and LBGT.

Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.

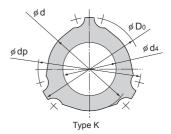


Table3 Sectional Shape of the Standard Hollow Type Spline Shaft

Unit: mm

Nominal shaft diameter	20	25	30	40	50	60	85
Minor diameter φ d	15.3	19.5	22.5	31	39	46.5	67
Major diameter	19.7	24.5	29.6	39.8	49.5	60	84
Ball center-to-center diameter ϕ dp	20	25	30	40	50	60	85
Hole diameter	6	8	12	18	24	30	45
Mass (kg/m)	1.6	2.3	2.9	4.9	7	10	19.5

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

[Chamfering of the Spline Shaft Ends]

For details, see A3-70.

[Length of Imperfect Area of a Special Spline Shaft]

If the middle area or the end of a spline shaft is to be thicker than the minor diameter (ϕ d), an imperfect spline area is required to secure a recess for grinding. Table4 shows the relationship between the length of the incomplete section (S) and the flange diameter (ϕ df).

(This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.)

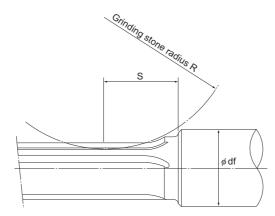


Table4 Length of Imperfect Spline Area: S

Unit: mm

Flange diameter ϕ df	20	25	30	35	40	50	60	80	100	120	140
Nominal shaft diameter											
20	35	43	51	57	62	_	_	_	_	_	_
25	I	51	64	74	82	97	-	I	_	I	_
30	_	_	54	67	76	92	105	_	_	_	_
40	_	_	_	_	59	80	95	119	_	_	_
50	_	_	_	_	_	63	83	110	131	_	_
60	_	_	_	_	_	_	66	100	123	140	_
70	_	_	_	_	_	_	_	89	115	134	150
85	_	_	_	_	_	_	_	61	98	122	140

Rotary Ball Spline

With Support Bearing Type Models LTR and LTR-A

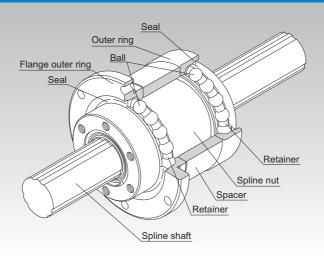


Fig.1 Structure of Rotary Ball Spline Model LTR

A3-6
A3-111
A 3-113
A 3-115
A 3-116
A24-1
■ 3-30
△3-17
▲3-26
▲3-29
▲3-33
△ 3-109

Structure and Features

With the Rotary Ball Spline model LTR, the spline shaft has three crests on the circumference, and along both sides of each crest, two rows of balls (six rows in total) are arranged to hold the crest so that a reasonable preload is applied.

Angular-contact ball raceways are machined on the outer surface of the spline nut to constitute support bearings, allowing the whole body to be compactly and lightly designed.

The rows of balls are held in a special resin retainer so that they smoothly roll and circulate. With this design, balls will not fall even if the spline shaft is removed.

In addition, a dedicated seal for preventing foreign material from entering the support bearings is available.

[No Angular Backlash]

Two rows of balls facing one another hold a crest, formed on the circumference of the spline nut, at a contact angle of 20° to provide a preload in an angular-contact structure. This eliminates an angular backlash in the rotational direction and increases the rigidity.

[Compact Design]

The spline nut is integrated with the support bearings, allowing highly accurate, compact design to be achieved.

[Easy Installation]

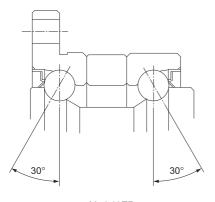
This ball spline can easily be installed by simply securing it to the housing using bolts.

[High Rigidity]

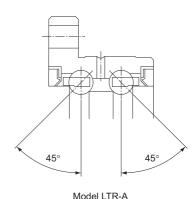
Since the contact angle is large and an appropriate preload is given, high rigidity against torque and moment is achieved.

The support bearing has a contact angle of 30° to secure high rigidity against a moment load, thus to achieve a rigid shaft support.

Model LTR-A, a compact type of LTR, has a contact angle of 45°.







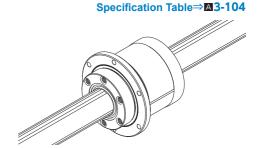
Model LIK-A

Types and Features

[Types of Spline Nuts]

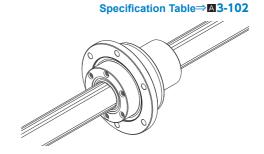
Ball Spline Model LTR

A compact unit type whose support bearings are directly integrated with the outer surface of the spline nut.



Ball Spline Model LTR-A

A compact type even smaller than LTR.



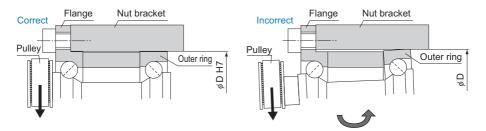
[Types of Spline Shafts]

For details, see A3-76.

Housing Inner-diameter Tolerance

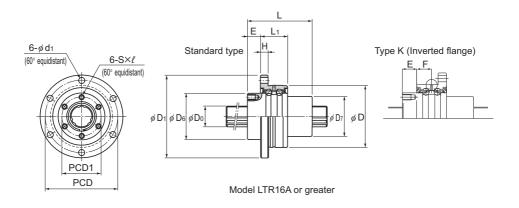
For the housing inner-diameter tolerance for model LTR, class H7 is recommended.

[Important note concerning model LTR]



Note) Because of the divided outer ring, it is necessary to incorporate inner-diameter tolerance in the nut bracket (H7 is recommended) to prevent shifting of the outer ring on the side opposite the flange.

Model LTR-A Compact Type



						SI	pline	nut dir	nensio	ins					
Model No.		Outer ameter	Length	Flange diameter					Standard type	Туре К	Oil hole position				
	D	Toler- ance	L	D₁	D₅ h7	D ₇	Н	L ₁	E	Е	F	E ₁	PCD	PCD1	S×ℓ
LTR8 A	32		25	44	24	16	3	10.5	6	8.5	4	3	38	19	M2.6×3
LTR10 A	36	-0.009 -0.025	33	48	28	21	3	10.5	9	11.5	4	_	42	23	M3×4
LTR16 A	48		50	64	36	31	6	21	10	10	10.5	_	56	30	M4×6
LTR20 A	56		63	72	43.5	35	6	21	12	12	10.5	_	64	36	M5×8
LTR25 A	66	-0.010 -0.029	71	86	52	42	7	25	13	13	12.5	_	75	44	M5×8
LTR32 A	78		80	103	63	52	8	25	17	17	12.5	_	89	54	M6×10
LTR40 A	100	-0.012 -0.034	100	130	79.5	64	10	33	20	20	16.5	_	113	68	M6×10

Model number coding

UU ZZ CL A +500L LTR32

accessory symbol(*3)

Model No. Flange orientation Compact Support Type symbol(*1) Symbol for clearance Spline nut contamination protection accessory symbol(*2) Number of spline nuts Support bearings on one shaft

Symbol for standard hollow spline shaft (*6) in the rotational direction(*4) contamination protection

Overall spline shaft length (*7) (in mm)

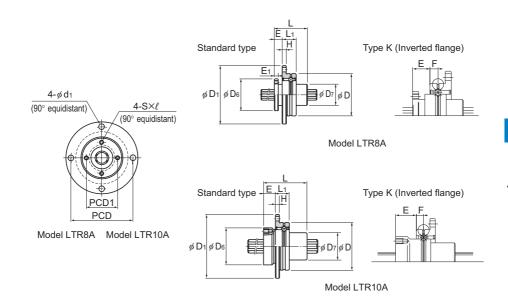
Accuracy symbol

(*2) See A3-113. (*3) See A3-113. (*4) See A3-29. (*5) See A3-33. (*6) See A3-106. (*7) See A3-109.

(*1) No Symbol: standard K: flange inversed



(no symbol for one nut)



Unit: mm

	Spline shaft diameter		Basic torque rating		Basic load rating		Static permissible moment	Support bearing basic load rating		Mass	
d₁	D₀ h7	Rows of balls	C _⊤ N-m	С₀т N-m	C KN	C₀ KN	M _A ** N-m	C kN	C₀ kN	Spline Nut kg	Spline shaft kg/m
3.4	8	4	1.96	2.94	1.47	2.55	5.9	0.69	0.24	0.08	0.4
3.4	10	4	3.92	7.84	2.84	4.9	15.7	0.77	0.3	0.13	0.62
4.5	16	6	31.3	34.3	7.06	12.6	67.6	6.7	6.4	0.35	1.6
4.5	20	6	56.8	55.8	10.2	17.8	118	7.4	7.8	0.51	2.5
5.5	25	6	105	103	15.2	25.8	210	9.7	10.6	0.79	3.9
6.6	32	6	180	157	20.5	34	290	10.5	12.5	1.25	5.6
9	40	6	418	377	37.8	60.4	687	16.5	20.7	2.51	9.9

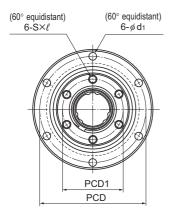
Note) **M_a indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure below.

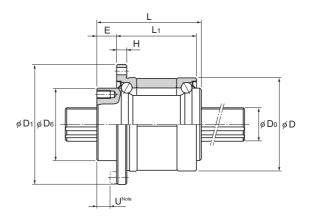
For details on the maximum lengths of ball spline shofts by accuracy please see [83,109]

For details on the maximum lengths of ball spline shafts by accuracy, please see **A3-109**.



Model LTR





		Spline nut dimensions									
Model No.	Outer	diameter	Length	Flange diameter							
	D	Tolerance	L	D ₁	D₅ h7	Н	L ₁	Е	PCD	PCD1	S×ℓ
LTR 16	52		50	68	39.5	5	37	10	60	32	M5×8
LTR 20	56	0	63	72	43.5	6	48	12	64	36	M5×8
LTR 25	62	-0.007	71	78	53	6	55	13	70	45	M6×8
LTR 32	80		80	105	65.5	9	60	17	91	55	M6×10
LTR 40	100	0	100	130	79.5	11	74	23	113	68	M6×10
LTR 50	120	-0.008	125	156	99.5	12	97	25	136	85	M10×15
LTR 60	134	0 -0.009	140	170	115	12	112	25	150	100	M10×15

Model number coding

K UU ZZ CM +1000L LTR50

Model No. Flange orientation symbol(*1)

Symbol for clearance Accuracy symbol in the rotational (*5) direction(*4)

Symbol for standard hollow spline shaft (*6)

(no symbol for one nut) accessory symbol(*2)

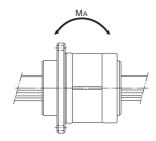
Support bearings contamination protection contamination protection (in mm) accessory symbol(*3)

Overall spline shaft length (*7)

(*2) See A3-113. (*3) See A3-113. (*4) See A3-29. (*5) See A3-33. (*6) See A3-106. (*7) See A3-109.

(*1) No Symbol: standard K: flange inversed





Unit: mm

												Offic. Hilli
		Spline shaft diameter		Basic torque rating		Basic load rating		Static permissible moment	Support bearing basic load rating		Ма	SS
d₁	U ^{Note}	D₀ h7	Rows of balls	C _⊤ N-m	С₀т N-m	C kN	C₀ kN	M _A ** N-m	C KN	C₀ kN	Spline Nut kg	Spline shaft kg/m
4.5	5	16	6	31.4	34.3	7.06	12.6	67.6	12.7	11.8	0.51	1.6
4.5	7	20	6	56.9	55.9	10.2	17.8	118	16.3	15.5	0.7	2.5
4.5	8	25	6	105	103	15.2	25.8	210	17.6	18	0.93	3.9
6.6	10	32	6	180	157	20.5	34	290	20.1	24	1.8	5.6
9	13	40	6	419	377	37.8	60.5	687	37.2	42.5	3.9	9.9
11	13	50	6	842	769	60.9	94.5	1340	41.7	54.1	6.7	15.5
11	13	60	6	1220	1040	73.5	111.7	1600	53.1	68.4	8.8	22.3

Note) **M₁ indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the

Figure above.

Dimension U represents the dimension from the head of the hexagonal-socket-head type bolt to the spline nut end. For details on the maximum lengths of ball spline shafts by accuracy, please see

3-109.

Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (types K and N), as described on **A3-76**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table1 shows the sectional shape of a spline shaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.

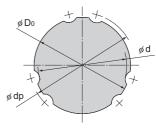


Table1 Sectional Shape of the Spline Shaft

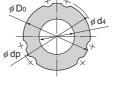
Unit: mm

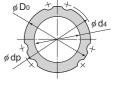
Nominal shaft diameter	8	10	16	20	25	32	40	50	60
Minor diameter ϕ d	7	8.5	14.5	18.5	23	30	37.5	46.5	56.5
Major diameter ø D₀ h7	8	10	16	20	25	32	40	50	60
Ball center-to-center diameter ϕ dp	9.3	11.5	17.8	22.1	27.6	35.2	44.2	55.2	66.3
Mass (kg/m)	0.4	0.62	1.6	2.5	3.9	5.6	9.9	15.5	22.3

^{*}The minor diameter ϕ d must be a value at which no groove is left after machining.

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table2 shows the hole shape of the standard hollow type spline shaft (types K and N). Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.





Type K (Thick)

Type N (Thin)

Table2 Sectional Shape of the Standard Hollow Type Spline Shaft

Unit: mm

Nominal sh	naft diameter	8	10	16	20	25	32	40	50	60
Major dian	meter <i>φ</i> D₀ h7	8	10	16	20	25	32	40	50	60
Ball center-to-ce	enter diameter ødp	9.3	11.5	17.8	22.1	27.6	35.2	44.2	55.2	66.3
Type K	Hole diameter ød₄	3	4	7	10	12	18	22	25	32
Type K	Mass(kg/m)	0.35	0.52	1.3	1.8	3	4.3	6.9	11.6	16
Type N	Hole diameter ød₄	_	_	11	14	18	23	29	36	_
туретч	Mass(kg/m)	_		0.8	1.3	1.9	3.1	4.7	7.4	_

Note) The standard hollow type Spline Shaft is divided into types K and N. Indicate "K" or "N" at the end of the model number to distinguish between them when placing an order.

[Chamfering of the Spline Shaft Ends]

For details, see A3-70.

[Length of Imperfect Area of a Special Spline Shaft]

If the middle area or the end of a spline shaft is to be thicker than the minor diameter (ϕ d), an imperfect spline area is required to secure a recess for grinding. Table3 shows the relationship between the length of the incomplete section (S) and the flange diameter (ϕ df).

(This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.)

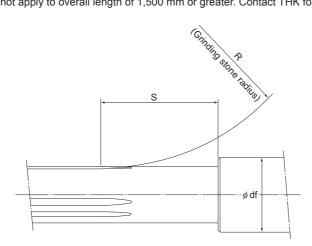


Table3 Length of Imperfect Spline Area: S

Unit: mm

Flange diameter ϕ df	6	8	10	13	16	20	25	30	40	50	60	80	100	120	140	160
Nominal shaft diameter	0	0	10	2	2	20	2	30	40	30	00	00	100	120	140	100
8	_	25	29	35	41	_	_	_	_		_	_	_	_	_	_
10	_		26	31	38	45	-	-	-	_	_	_	_	_	_	_
16	_	_	_	_	36	47	58	67	_	_	_	_	_	_	_	_
20		_	_	_	_	37	50	60	76		_	_		_	_	_
25	_	_	_	_	_	_	38	51	72	88	_	_	_	_	_	_
32	_	_	_	_	_	_	_	_	40	75	88	109	_	_	_	_
40	_		_	_	_	_	_	_	42	63	81	107	_	_	_	_
50									_	45	65	96	118	_		
60					_	_	_	_	_		50	87	114	134	_	

Permissible Rotational Speed for Rotary Ball Splines

For model LTR rotary ball splines, the speed is restricted by whichever is lower of the support bearing permissible rotational speed and the critical speed of the spline. When using the product, do not exceed the permissible rotational speed.

Table4 Model LTR permissible rotational speed

Unit:min-1

	Permissible Ro	otational Speed	
Model No.	Ball spline	Support	bearing
	Calculated using shaft length	Grease Lubrication	Oil Lubrication
LTR16		4000	5400
LTR20		3600	4900
LTR25		3200	4300
LTR32	see A3-16.	2400	3300
LTR40		2000	2700
LTR50		1600	2200
LTR60		1400	2000

Table5 Model LTR-A permissible rotational speed

Unit:min-1

	Permissible Ro	otational Speed				
Model No.	Ball spline	Support bearing				
	Calculated using shaft length	Grease Lubrication	Oil Lubrication			
LTR8A		6900	9300			
LTR10A		5900	7900			
LTR16A		4000	5400			
LTR20A	see A3-16 .	3600	4900			
LTR25A		3200	4300			
LTR32A		2400	3300			
LTR40A		2000	2700			

Dimensional Drawing, Dimensional Table

Maximum Manufacturing Length by Accuracy

Maximum Manufacturing Length by Accuracy

Table1, Table2, Table3 and Table4 show the maximum manufacturing lengths of ball spline shafts by accuracy.

Table1 Maximum Manufacturing Length of Models SLS, SLS-L and SLF

Unit: mm

Nominal shaft diameter	Accuracy				
Nominal Shart diameter	Normal grade (No symbol) High accuracy grade (H)		Precision Grade (P)		
25	2000	1500	1000		
30	2000	1600	1250		
40	2000	2000	1500		
50	3000	2000	1500		
60	4000	2000	2000		
70	4000	2000	2000		
80	4000	2000	2000		
100	4000	3000	3000		

Table2 Maximum Manufacturing Length of Models LBS, LBST, LBF, LBR, LBH, LBG and LBGT by Accuracy

Unit: mm

Nominal shaft diameter	Accuracy				
Nominal shall diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)		
6	200	150	100		
8	600	200	150		
10	600	400	300		
15	1800	600	600		
20	1800	700	700		
25 3000		1400	1400		
30	3000	1400	1400		
40	3000	1400	1400		
50	3000	1400	1400		
60	3800	2500	2000		
70	3800	2500	2000		
85	3800	3000	3000		
100	4000	3000	3000		
120	3000	3000	3000		
150 3000		3000	3000		

Table3 Maximum Manufacturing Length of Model LT-X

Unit: mm

Nominal shaft diameter	Accuracy				
Norminal Shall diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)		
4	200	150	100		
5	250	200	100		
6	315	250	200		

Table4 Maximum Manufacturing Length of Models LT, LF, LTR and LTR-A by Accuracy

Unit: mm

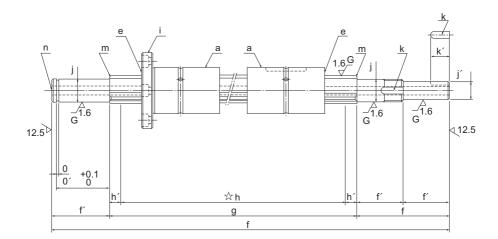
Nominal shaft diameter	Accuracy			
Nominal shart diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)	
4	600	200	200	
5	600	315	200	
6	600	400	315	
8	1000	500	400	
10	1000	630	500	
13 1000		800	630	
16	16 2000		1000	
20 2000		1500	1000	
25	25 3000		1000	
30	3000	1600	1250	
40	3000	2000	1520	
50	3000	2000	1500	
60	4000	2000	2000	
80	4000	2000	2000	
100	4000	3000	3000	

The length in the table represents the overall shaft length.
 With standard hollow shaft type (K), the values in the table apply.
 With standard hollow shaft type (N), the available maximum length for both the normal grade and the high accuracy grade is up to the length defined for the precision grade in the table.

Checking List for Spline Shaft End Shape

If desiring a ball spline type with its end specially machined, check the following items when placing an order.

The diagram below shows a basic configuration of the Ball Spline.



[Check Items]

- a. Type of the spline nut to be fit
- b. Number of spline nuts
- c. Clearance in the rotation direction
- d. Accuracy
- e. With/without a seal (for a single seal, check its orientation)
- f. Overall length (including all dimensions? Total value correct?)
- g. Effective spline length
- h. Hardened area (mark the location with symbol ☆ and indicate the purpose of hardening)
- i. Orientation of the flange (for flanged type)

- j. Spline shaft end shape (thicker than the minimum spline diameter?) (black, mill scale)
- k. Positional relationship between the spline nut and the spline shaft end shape (keyway of the spline nut, flange mounting hole)
- I. Indication of chamfering for each part
- m. Shape of chamfer on the spline shaft end (see A3-70)
- n. Intended purpose of the though hole in the spline shaft if any
- o. o'. Snap ring groove
- p. Maximum length
- g. Precedented or not

Housing Inner-diameter Tolerance

When fitting the spline nut to the housing, tight fitting is normally recommended. If the accuracy of the Ball Spline does not need to be very high, clearance fitting is also acceptable.

Table1 Housing Inner-diameter Tolerance

•					
Housing Inner-diam-	General conditions	H7			
eter Tolerance	When clearance needs to be small	J6			

Note) For the housing inner-diameter tolerance of Rotary Ball Spline model LTR, H7 is recommended.

Positions of the Spline-nut Keyway and Mounting Holes

The keyways formed on the outer surface of straight nuts for Ball Spline models are positioned where balls under a load are placed as shown in Fig.1.

The flange-mounting holes of the flange types are positioned as shown in Fig.2.

When placing an order, indicate their positions in relation to the keyway or the like to be formed on the spline shaft.

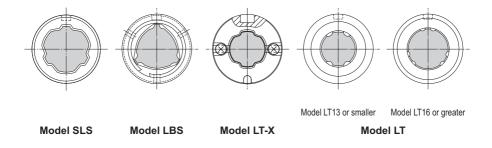


Fig.1 Positions of Keyways

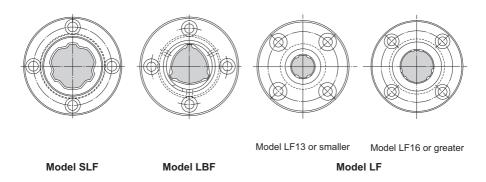


Fig.2 Positions of Flange Mounting Holes

Lubrication

To prevent foreign material from entering the spline nut and the lubricant from leaking, special synthetic resin seals with high wear resistance are available for the Ball Spline.

Spline nuts with seals (seal for both ends type UU, and seal for one end) contain high-quality lithium-soap group grease No. 2. However, if using them at high speed or with a long stroke, replenish grease of the same type through the greasing hole on the spline nut after running in.

Afterward, replenish grease of the same type as necessary according to the service conditions.

The greasing interval differs depending on the conditions. Normally, replenish the lubricant (or replace the product) roughly every 100 km of travel distance (six months to one year) as a rule of thumb.

For a Ball Spline model type without a seal, apply grease to the interior of the spline nut or to the raceways of the spline shaft.

Material and Surface Treatment

Depending on the service environment, the Ball Spline requires anticorrosive treatment or a different material. For details of anticorrosive treatment and material change, contact THK.

Contamination Protection

Entrance of dust or other foreign material into the spline nut will cause abnormal wear or shorten the service life. Therefore, it is necessary to prevent detrimental foreign material from entering the Ball Spline. When entrance of dust or other foreign material is a possibility, it is important to select effective seals and/or dust-control device that meets the environment conditions.

For the Ball Spline, a special synthetic rubber seal that is highly resistant to wear is available as a contamination protection accessory. If desiring a higher contamination protection effect, a felt seal is also available for some types. For details about the felt seal, contact THK. In addition, THK produces round bellows. Contact us for details.

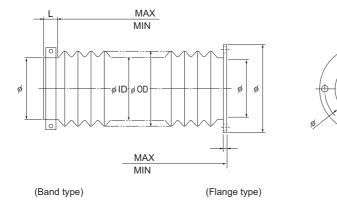
Table1 Dust prevention accessory symbol

Symbol	Contamination protection accessory
No Symbol Without seal	
UU	Rubber seal attached on both ends of spline nut
U	Rubber seal attached on either end of spline nut
DD Felt seal attached on both ends of spline nut	
D	Felt seal attached on either end of spline nut
ZZ	Rubber seal attached on both ends of support bearings
Z	Rubber seal attached on either end of support bearings

4-ø

Specifications of the Bellows

Bellows are available as a contamination protection accessory. Use this specification sheet.



Specifications of the Bellows

Supported Ball Screw models:

Number of Units To Be Manufactured:

Dimension	ns of the	Bellows				
Stroke:() mm	MAX:() mm	MIN:() mm	
Permissible of	outer diamete	er:(øOD) Desire	ed inner diam	eter:(ø ID)
How It Is	Used					
Installation d	irection:(hor	zontal, verti	cal, slant)	Speed:() mm/sec.	min.
Motion:(recip	procation, vib	ration)				
Condition	s					
Resistance to	oil and wate	er: (necessar	y, unnecessa	ry) Oil r	name ()
Chemical res	istance: Nam	e () × () %
Location: (in	door, outdoo	r)				
Remarks:						

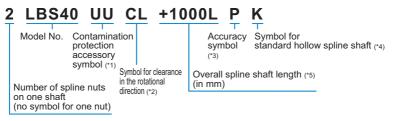
Model No. Ball Spline

Model Number Coding

Model number configurations differ depending on the model features. Refer to the corresponding sample model number configuration.

[Ball Spline]

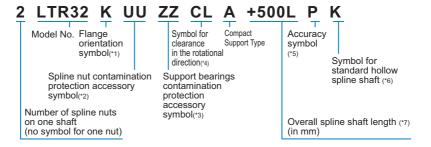
Models SLS, SLS-L, SLF, LBS, LBST, LBF, LBR, LBH, LT, LF and LT-X



(*1) See A3-113. (*2) See A3-29. (*3) See A3-33. (*4) See A3-69. (*5) See A3-109.

[Rotary Ball Spline]

Models LTR, LTR-A, LBG and LBGT



(*2) See A3-113. (*3) See A3-113. (*4) See A3-29. (*5) See A3-33. (*6) See A3-106. (*7) See A3-109.

(*1) No Symbol: standard K: flange inversed

[Handling]

- (1) Please use at least two people to move any product weighing 20 kg or more, or use a dolly or another conveyance. Doing so may cause injury or damage.
- (2) Do not disassemble the parts. This will result in loss of functionality.
- (3) Tilting a spline nut or spline shaft may cause them to fall by their own weight.
- (4) Take care not to drop or strike the Ball Spline. Doing so may cause injury or damage. Giving an impact to it could also cause damage to its function even if the product looks intact.
- (5) When assembling, do not remove the spline nut from the spline shaft.
- (6) When handling the product, wear protective gloves, safety shoes, etc., as necessary to ensure safety.

[Precautions on Use]

- (1) Prevent foreign material, such as cutting chips or coolant, from entering the product. Failure to do so may cause damage.
- (2) If the product is used in an environment where cutting chips, coolant, corrosive solvents, water, etc., may enter the product, use bellows, covers, etc., to prevent them from entering the product.
- (3) Do not use the product at temperature of 80°C or higher. Except for the heat-resistant models, exposure to higher temperatures may cause the resin/rubber parts to deform/be damaged.
- (4) If foreign material such as cutting chips adheres to the product, replenish the lubricant after cleaning the product.
- (5) Micro-strokes tend to obstruct oil film to form on the raceway in contact with the rolling element, and may lead to fretting corrosion. Take consideration using grease offering excellent fretting prevention. It is also recommended that a stroke movement corresponding to the length of the spline nut be made on a regular basis to make sure oil film is formed between the raceway and rolling element.
- (6) Do not use undue force when fitting parts (pin, key, etc.) to the product. This may generate permanent deformation on the raceway, leading to loss of functionality.
- (7) Skewing or misalignment of the spline shaft support and spline nut can shorten service life substantially. Inspect the components carefully and make sure they are mounted correctly.
- (8) The spline nut must contain all its internal rolling elements (balls) when mounted on the spline shaft. Using a spline nut with any balls removed may result in premature damage.
- (9) Please contact THK if any balls fall out of the spline nut; do not use the spline nut if any balls are missing.
- (10) To mount the spline nut on the spline shaft, first locate the alignment indicators on both components, then insert the shaft through the opening in the spline nut, without forcing it, and adjust the position until the indicators are aligned. Forcing the shaft could cause balls to fall out. When mounting a spline nut equipped with a seal or preload, first lubricate the outer surface of the spline shaft.
- (11) Manipulate the spline nut gently, using a jig, when inserting it into the housing, taking care not to strike the side plate, end cap, or seal.
- (12) If an attached component is insufficiently rigid or mounted incorrectly, the bearing load will be concentrated at one location and performance will decline significantly. Make sure the housing and base are sufficiently rigid, the anchoring bolts are strong enough, and the component is mounted correctly.
- (13) If desiring to have a flanged-type Ball Spline additionally machined, such as having a dowel pin hole, contact THK.

Precautions on Use

[Lubrication]

- (1) Thoroughly wipe off anti-rust oil and feed lubricant before using the product.
- (2) Do not combine different lubricants. Mixing lubricants can cause adverse interaction between disparate additives or other ingredients.
- (3) If the product will be exposed to constant vibration or high or low temperatures, or used in a clean room, vacuum, or other special environment, apply a lubricant suitable for both the specifications and the environment.
- (4) To lubricate a product that has no grease nipple or oil hole, apply lubricant directly to the raceway surface and execute a few preliminary strokes to ensure that the interior is fully lubricated.
- (5) Bear in mind that the Ball Spline's slide resistance is affected by changes in the consistency of the lubricant, which varies according to the temperature.
- (6) The Ball Spline may encounter increased slide resistance following lubrication, due to the lubricant's agitation resistance. Make sure to put the unit through some preliminary motions to ensure that it is fully lubricated before starting up the machine.
- (7) Excess lubricant may spatter immediately after lubrication. If necessary, wipe off any spattered grease.
- (8) Because lubricant performance declines over time, lubrication must be monitored regularly and fresh lubricant applied when needed, depending on how frequently the machine is operated.
- (9) The appropriate lubrication schedule will depend on usage conditions and the surrounding environment. In general, the unit should be lubricated after every 100 kilometers of operation (every 3 to 6 months). The actual lubrication schedule and amount of lubricant used should be determined by the condition of the machinery.
- (10) With oil lubrication, the lubricant may not always be thoroughly disseminated inside the Ball Spline, depending on its mounting position. If the preferred lubrication method is oil lubrication, please consult THK in advance.

[Storage]

When storing the Ball Spline, enclose it in a package designated by THK and store it in a room in a horizontal orientation while avoiding high temperature, low temperature and high humidity.

After the product has been in storage for an extended period of time, lubricant inside may have deteriorated, so add new lubricant before use.

[Disposal]

Dispose of the product properly as industrial waste.



Ball Spline THK General Catalog

Ball Spline THK General Catalog

B Support Book

Features and Types	B 3-4
Features of the Ball Spline	
Structure and Features	
Classification of Ball Splines	
Point of Selection	B 3-8
Flowchart for Selecting a Ball Spline	
Steps for Selecting a Ball Spline	. B 3-8
Selecting a Type	. B 3-10
Studying the Spline Shaft Strength	
Predicting the Service Life	. B 3-19
Example of Calculating the Service Life	
Mounting Procedure and Maintenance	B 3-30
Assembling the Ball Spline	. 🖪 3-30
Mounting the Spline	. B 3-30
Installing the Spline Nut	. B 3-32
Installation of the Spline Shaft	. B 3-32
Lubrication	B 3-33
Options	B 3-34
Material and Surface Treatment	. B 3-34
Contamination Protection	. B 3-34
Model No	B 3-35
Model Number Coding	. B 3-35
Precautions on Use	. B 3-36

▲ Product Descriptions (Separate)

Classification of Ball Splines	A3-4	Types and Features Housing Inner-diameter Tolerance	
Point of Selection	A 3-6	Trousing inner-diameter folerance	. 🖾 5-11
Flowchart for Selecting a Ball Spline		Dimensional Drawing, Dimensional Table	
Steps for Selecting a Ball Spline		Model LT	A 3-78
Selecting a Type		Model LF	
Studying the Spline Shaft Strength		Model LT-X	
Predicting the Service Life		Model LT with Recommended Shaft End Shape.	
Selecting a Preload		Spline Shaft	
Clearance in the Rotation Direction		Accessories	
Preload and Rigidity		710000001100	. 200 07
Conditions and Guidelines for Selecting of a Preload		Rotary Ball Spline	
Determining the Accuracy		With Geared Type Models LBG and LBGT .	A3-88
Accuracy Grades		Structure and Features	
Accuracy Standards		Types and Features	
Accuracy Standards	A3-33	Housing Inner-diameter Tolerance	
High Tayous Canad Ball Culina		riousing inner-diameter folerance	. 🖾 5-51
High Torque Caged Ball Spline	2 0.00	Dimensional Drawing, Dimensional Table	
Models SLS, SLS-L and SLF		Model LBG	A3-02
Structure and Features		Model LBGT	
Types and Features		Spline Shaft	
Housing Inner-diameter Tolerance	A3-41	· Spilite Strait	. 🖾 5-30
Dimensional Drawing, Dimensional Table		Rotary Ball Spline	
Model SLS	A 3-42	With Support Bearing Type Models LTR and LTR-A.	. A3-98
Model SLF		Structure and Features	
Spline Shaft		Types and Features	. A3-10
Accessories		Housing Inner-diameter Tolerance	. A3-10
Accessories	A3-40	· ·	
High Torque Type Ball Spline		Dimensional Drawing, Dimensional Table	
Models LBS, LBST, LBF, LBR and LBH	A 3-50	Model LTR-A Compact Type	. A3-10
Structure and Features		Model LTR	. A3-10
Applications		Spline Shaft	. A3-10
Types and Features		 Permissible Rotational Speed for Rotary Ball Splines . 	
Housing Inner-diameter Tolerance		Maximum Manufacturing Length by Accuracy .	. A3-10
ŭ		Doint of Doning	EX 0 44
Dimensional Drawing, Dimensional Table		Point of Design	
Miniature Ball Spline		Checking List for Spline Shaft End Shape .	
Model LBS (Medium Load Type)	A3-58	Housing Inner-diameter Tolerance	
Model LBST (Heavy Load Type)		Positions of the Spline-nut Keyway and Mounting Holes .	. A3-11
Model LBF (Medium Load Type)		041	5 0 44
Model LBR		Options	
Model LBH		Lubrication	
Model LBS with Recommended Shaft End Shape		Material and Surface Treatment	
Spline Shaft		Contamination Protection	
Accessories		Specifications of the Bellows	. A3-11
		Model No.	A 3-11
Medium Torque Type Ball Spline Models LT, LF and LT-X	A3-74	Model Number Coding	
Structure and Features		December of U	 0 44
		Precautions on Use	. A3-11

Features of the Ball Spline

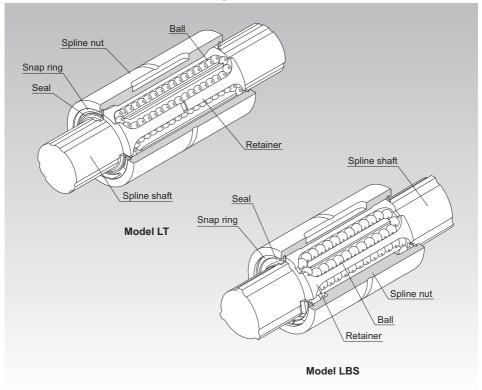


Fig.1 Structure of Ball Spline Models LBS and LT

Structure and Features

The Ball Spline is an innovative linear motion system in which balls accommodated in the spline nut transmit torque while linearly moving on precision-ground raceways on the spline shaft.

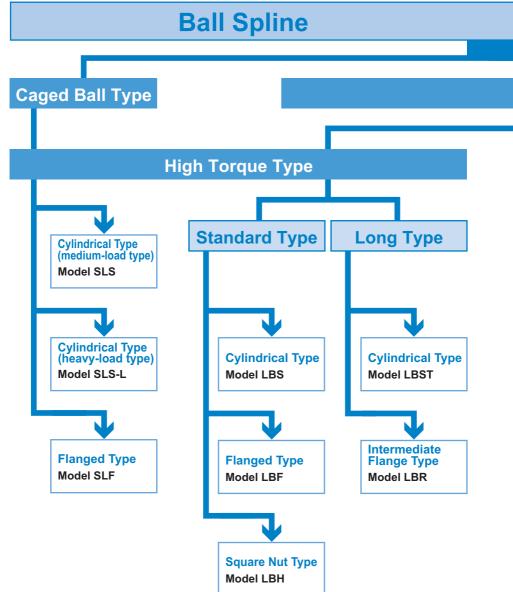
Unlike the conventional structure, a single spline nut can provide a preload with THK's Ball Spline. As a result, the Ball Spline demonstrates high performance in environments subject to vibrations and impact loads, locations where a high level of positioning accuracy is required or areas where high-speed kinetic performance is required.

In addition, even when used as an alternative to a linear bushing, the Ball Spline achieves a rated load more than 10 times greater than the linear bushing with the same shaft diameter, allowing it to compactly be designed and used in locations where an overhung load or a moment load is applied. Thus, the Ball Spline provides a high degree of safety factor and long service life.

Features and Types

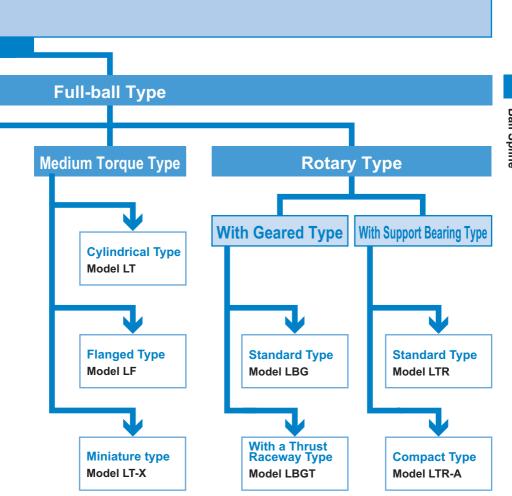
Features of the Ball Spline

Classification of Ball Splines



Features and Types

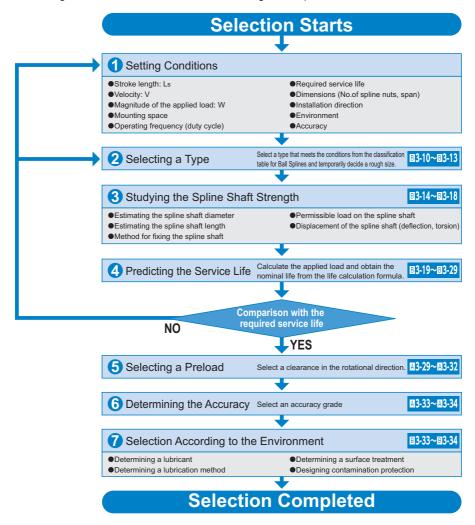
Classification of Ball Splines



Flowchart for Selecting a Ball Spline

Steps for Selecting a Ball Spline

The following is a flowchart to reference when selecting a Ball Spline.



Flowchart for Selecting a Ball Spline



Selecting a Type

There are three types of the Ball Spline: high torque type, medium torque type and rotary type. You can choose a type according to the intended use. In addition, wide arrays of spline nut shapes are available for each type, enabling the user to choose a desired shape according to the mounting or service requirements.

	Classification	Туре	Shape	Shaft diameter
aged Ball type		Type SLS Type SLS-L		Nominal shaft diameter 25 to 100mm
High torque Caged Ball type		Type SLF		Nominal shaft diameter 25 to 100mm
		Type LBS Type LBST		Nominal shaft diameter 6 to 150mm
High torque type	<u> </u>	Type LBF		Nominal shaft diameter 15 to 100mm
High tor	8	Type LBR		Nominal shaft diameter 15 to 100mm
		Type LBH		Nominal shaft diameter 15 to 50mm

^{*}For the specification table for each model, please see "A Product Descriptions."

Selecting a Type

Specification Table	Structure and features	Major application			
⊠3-42	 Redesigning the shape of the conventional high torque type spline shaft to be more circular significantly improves its torsion and flexural rigidity. Models SLS/SLF adopt the caged-ball technology to enable the circulating motion of evenly spaced balls to be maintained and high-speed response to be achieved, the cycle time of the machine can be improved. Models SLS/SLF adopt the caged-ball technology, they eliminate collision and mu- 				
A 3-44	tual friction between balls, and realize low noise, pleasant running sound and low particle generation. • Models SLS/SLF adopt the caged-ball technology to substantially increase the grease retention, thus achieving long-term maintenance-free operation. • Models SLS/SLF adopt the caged-ball technology and a new circulation method, thus achieving stable and smooth motion with small rolling fluctuation.	 Column and arm of industrial robot Automatic loader Transfer machine Automatic conveyance system Tire molding machine Spindle of spot-welding machine Guide shaft of high-speed automatic coating machine 			
⊠3-56	The spline shaft has three crests equidistantly formed at angles of 120°. On both	Riveting machine Wire winder Work head of electric discharge machine Spindle drive shaft of grinding machine Speed gears Precision indexing machine			
⊠3-62	sides of each crest, two rows (six rows in total) of balls are arranged to hold the crest from both sides. The angular-contact design of the ball contact areas allows an appropriate preload to be evenly applied. Since the balls circulate inside the spline put the outer dimensions of the spline put.	Ğ			
⊠3-64	nut, the outer dimensions of the spline nut are compactly designed. Even under a large preload, smooth straight motion is achieved. Since the contact angle is large (45°) and the displacement is minimal, high rigidity is achieved.				
⊠3-66	 No angular backlash occurs. Capable of transmitting a large torque. 				

	Classification	Туре	Shape	Shaft diameter
/pe	Medium Torque type	Type LT		Nominal shaft diameter 4 to 100mm
dium torque ty		Type LF		Nominal shaft diameter 6 to 50mm
Med		Type LT-X		Nominal shaft diameter 4 to 6mm
/ type	Ту	Type LBG Type LBGT		Nominal shaft diameter 20 to 85mm
Rotary type	Rotation	Type LTR-A Type LTR		Nominal shaft diameter 8 to 60mm

^{*}For the specifi cation table for each model, please see "A Product Descriptions."

Selecting a Type

Specification Table	Structure and features	Major application	
∆3-78	The spline shaft has two to three crests. On both sides of each crest, two rows (four to six rows in total) of balls are arranged to hold the crest from both sides. This design allows	Die-set shaft and similar applications requiring straight motion under a Spot-welding ma-	
△3-80	 an appropriate preload to be evenly applied. The contact angle of 20° and an appropriate preload level eliminate angular backlash, providing high-torque moment rigidity. 	Loading system and similar applications requiring rotation to a given angle at a fixed position Automatic gas-welding XY recorders	
₽3-82	The length and external diameter of the LT-X ball spline's outer cylinder are the same as those of an LM-series linear bush, enabling the user to replace a linear bush with the LT-X.	machine spindle and simi a Automatic spinner lar applications requiring Optical measuring a whirl-stop on one shaft	
A 3-92	A unit type that has the same contact structure as model LBS. The flange circumference on the spline nut is machined to have gear teeth, and radial and thrust needle bearings are compactly combined on the circumference of the spline nut.	Speed gears for high torque transmission	
M3-102	A lightweight and compact type based on model LT, but has a spline nut circumference machined to have angular-contact type ball raceways to accommodate support bearings.	Z axis of scalar robot Wire winder	

Studying the Spline Shaft Strength

The spline shaft of the Ball Spline is a compound shaft capable of receiving a radial load and torque. When the load and torque are large, the spline shaft strength must be taken into account.

[Spline Shaft Receiving a Bending Load]

When a bending load is applied to the spline shaft of a Ball Spline, obtain the spline shaft diameter using the equation (1) below.

$$\mathbf{M} = \mathbf{\sigma} \cdot \mathbf{Z}$$
 and $\mathbf{Z} = \frac{\mathbf{M}}{\mathbf{\sigma}}$ (1)

M : Maximum bending moment acting on the spline shaft (N-mm)

 σ : Permissible bending stress of the spline shaft (98N/mm²)

Z : Modulus section factor of the spline shaft (mm³) (see Table3 on A3-17, Table4 on A3-18 and Table5 on A3-19)

[Reference] Section Modulus (Solid Circle)

$$Z = \frac{\pi \cdot d^3}{32}$$

Z : Section Modulus (mm³) d : Shaft outer diameter (mm)

[Spline Shaft Receiving a Torsion Load]

When a torsion load is applied on the spline shaft of a Ball Spline, obtain the spline shaft diameter using the equation (2) below.

$$T = \tau_a \cdot Z_P$$
 and $Z_P = \frac{T}{\tau_a}$ (2)

T : Maximum torsion moment (N-mm)

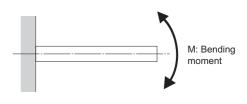
 $_{\tau_a}$: Permissible torsion stress of the spline shaft (49N/mm²)

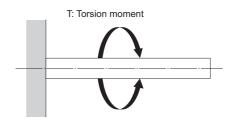
Z_P: Polar modulus of section of the spline nut (mm³) (see Table3 on **△3-17**, Table4 on **△3-18** and Table5 on **△3-19**)

[Reference] Section Modulus (Solid Circle)

$$Z_P = \frac{\pi \cdot d^3}{16}$$

 Z_P : Section modulus (mm 3) d: Shaft outer diameter (mm)





Studying the Spline Shaft Strength

[When the Spline Shaft Simultaneously Receives a Bending Load and a Torsion Load]

When the spline shaft of a Ball Spline receives a bending load and a torsion load simultaneously, calculate two separate spline shaft diameters: one for the equivalent bending moment (M_e) and the other for the equivalent torsion moment (T_e). Then, use the greater value as the spline shaft diameter.

Equivalent bending moment

$$\mathbf{M}_{\circ} = \frac{\mathbf{M} + \sqrt{\mathbf{M}^2 + \mathbf{T}^2}}{2} = \frac{\mathbf{M}}{2} \left\{ 1 + \sqrt{1 + \left(\frac{\mathbf{T}}{\mathbf{M}}\right)^2} \right\} \cdots \cdots (3)$$

 $M_e = \sigma \cdot Z$

Equivalent torsion moment

$$T_{\circ} = \sqrt{M^2 + T^2} = M \cdot \sqrt{1 + \left(\frac{T}{M}\right)^2} \cdots (4)$$

 $T_e = \tau_a \cdot Z_p$

[Rigidity of the Spline Shaft]

The rigidity of the spline shaft is expressed as a torsion angle per meter of shaft length. Its value should be limited within 1°/4.

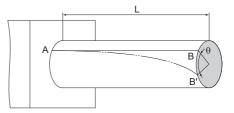
Rigidity of the shaft =
$$\frac{\text{Torsion angle}}{\text{Unit length}} = \frac{\theta \cdot \ell}{L} < \frac{1^{\circ}}{4}$$

θ : Torsion angle (°)
L : Spline shaft length (mm)

G : Transverse elastic modulus

(7.9×10⁴N/mm²) : Unit length (1000mm)

△3-18 and Table5 on **△3-19**)



[Deflection and Deflection Angle of the Spline Shaft]

The deflection and deflection angle of the Ball Spline shaft need to be calculated using equations that meet the relevant conditions. Table1 and Table2 represent these conditions and the corresponding equations.

Table3 on **△3-17**, Table4 on **△3-18** and Table5 on **△3-19** show the modulus section (Z) and the geometrical moments of inertia (I) of the spline shaft. Using Z and I values in the tables, the strength and displacement (deflection) of a typical Ball Spline model can be obtained.

Table1 Deflection and Deflection Angle Equations

Support method	Condition	Deflection equation	Deflection angle equation
Both ends free	P i2	$\delta_{\text{max}} = \frac{P\ell^3}{48EI}$	$i_1 = 0$ $i_2 = \frac{P\ell^2}{16EI}$
Both ends fas- tened	P P	$\delta_{\text{max}} = \frac{P\ell^3}{192EI}$	$i_1 = 0$ $i_2 = 0$
Both ends free	Uniform load p	$\delta_{\text{max}} = \frac{5p\ell^4}{384\text{EI}}$	$i_2 = \frac{p\ell^3}{24EI}$
Both ends fas- tened	Uniform load p	$\delta_{\text{max}} = \frac{p\ell^4}{384\text{EI}}$	$i_2 = 0$

Studying the Spline Shaft Strength

Table2 Deflection and Deflection Angle Equations

Support method	Condition	Deflection equation	Deflection angle equation
One end fas- tened	P Eg	$\delta_{\text{max}} = \frac{P\ell^3}{3EI}$	$i_1 = \frac{P\ell^2}{2EI}$ $i_2 = 0$
One end fas- tened	Uniform load p	$\delta_{max} = \frac{p\ell^4}{8EI}$	$i_1 = \frac{p\ell^3}{6EI}$ $i_2 = 0$
Both ends free	VE WE	$\delta_{\text{max}} = \frac{\sqrt{3}\text{Mo}\ell^2}{216\text{EI}}$	$i_1 = \frac{Mo\ell}{12EI}$ $i_2 = \frac{Mo\ell}{24EI}$
Both ends fas- tened	XE W. 11 Mo	$\delta_{\text{max}} = \frac{\text{Mo}\ell^2}{216\text{EI}}$	$i_1 = \frac{Mo\ell}{16EI}$ $i_2 = 0$

 δ_{max} : Maximum deflection (mm)

M₀: Moment (N-mm)

ℓ: Span (mm)

I: Geometrical moment of inertia (mm⁴)

 i_1 : Deflection angle at loading point

i₂: Deflection angle at supporting point

P: Concentrated load (N)

p: Uniform load (N/mm)

E: Modulus of longitudinal elasticity 2.06×10⁵ (N/mm^2)

[Dangerous Speed of the Spline Shaft]

When a Ball Spline shaft is used to transmit power while rotating, as the rotational speed of the shaft increases, the rotation cycle nears the natural frequency of the spline shaft. It may cause resonance and eventually result in inability to move. Therefore, the maximum rotational speed of the shaft must be limited to below the dangerous speed that does not cause resonance.

The dangerous speed of the spline shaft is obtained using the equation (6).

(0.8 is multiplied as a safety factor)

If the shaft's rotation cycle exceeds or nears the resonance point during operation, it is necessary to reconsider the spline shaft diameter.

Dangerous Speed

$$N_{c} = \frac{60\lambda^{2}}{2\pi \cdot \ell_{b}^{2}} \cdot \sqrt{\frac{E \times 10^{3} \cdot I}{\gamma \cdot A}} \times 0.8 \quad \cdots (6)$$

 N_c : Dangerous speed (min⁻¹) ℓ_b : Distance between two mounting surfaces (mm)

E: Young's modulus (2.06×10⁵ N/mm²)
I: Minimum geometrical moment of

inertia of the shaft (mm⁴)

 $I = \frac{\pi}{64} d^4$ d:Minor diameter (mm)

(see Table8, Table9 and Table10 on **A3-23**)

 γ : Density (specific gravity) (7.85 \times 10°kg/mm³)

A = $\frac{\pi}{4}$ d² d: Minor diameter (mm) (see Table8, Table9 and Table10 on **\(\Delta 3-23**)

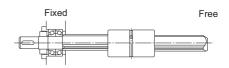
A : Spline shaft cross-sectional area (mm²)

λ : Factor according to the mounting method

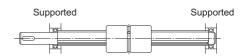
(1) Fixed - free λ =1.875 (2) Supported - supported λ =3.142

(3) Fixed - supported λ =3.927

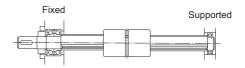
(4) Fixed - fixed λ =4.73



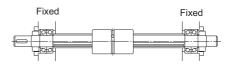
Fixed - free



Supported - supported



Fixed - supported



Fixed - fixed

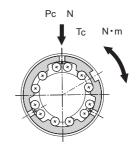
Predicting the Service Life

Predicting the Service Life

[Nominal Life]

The service life of a Ball Spline varies from unit to unit even if they are manufactured through the same process and used in the same operating conditions. Therefore, the nominal life defined below is normally used as a guidepost for obtaining the service life of a Ball Spline.

Nominal life is the total travel distance that 90% of a group of identical ball splines independently operating under the same conditions can achieve without showing flaking (scale-like pieces on a metal surface).



[Calculating the Nominal Life]

The nominal life of a Ball Spline varies with types of loads applied during operation: torque load, radial load and moment load. The corresponding nominal life values are obtained using the equations (7) to (10) below. (The basic load ratings in these loading directions are indicated in the specification table for the corresponding model number.)

When a Torque Load is Applied

$$L = \left(\frac{f_{\tau} \cdot f_{c}}{f_{w}} \cdot \frac{C_{\tau}}{T_{c}}\right)^{3} \times 50 \quad \cdots (7)$$

When a Radial Load is Applied

$$L = \left(\frac{f_{\tau} \cdot f_{c}}{f_{w}} \cdot \frac{C}{P_{c}}\right)^{3} \times 50 \quad \dots (8)$$

f_⊤ : Temperature factor

(see Fig.1 on **B3-21**)

fc : Contact factor

(see Table3 on **B3-21**)

fw : Load factor (see Table4 on **B3-21**)

When a Torque Load and a Radial Load are Simultaneously Applied

When a torque load and a radial load are simultaneously applied, calculate the nominal life by obtaining the equivalent radial load using the equation (9) below.

P_E = P_C+
$$\frac{4 \cdot T_C \times 10^3}{i \cdot dp \cdot cos\alpha}$$
(9)

P_E : Equivalent radial load (N) cosα : Contact angle i =Number of rows of balls under a load Type LBSα=45° i =2 (LBS10 or smaller) Type SLSα=40° i =3 (LBS15 or greater)

Type LTα=70° i =2 (LT13 or smaller) i =3 (LT16 or greater)

dp : Ball center-to-center diameter (mm) (see Table8, Table9 and Table10 on M3-23)

When a Moment Load is Applied to a Single Nut or Two Nuts in Close Contact with Each Other

Obtain the equivalent radial load using the equation (10) below.

When a Moment Load and a Radial Load are Simultaneously Applied

Calculated the nominal life from the sum of the radial load and the equivalent radial load.

Calculating the Service Life Time

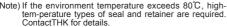
When the nominal life (L) has been obtained in the equation above, if the stroke length and the number of reciprocations per minute are constant, the service life time is obtained using the equation (11) below.

Predicting the Service Life

■f_T: Temperature Factor

If the temperature of the environment surrounding the operating Ball Spline exceeds 100°C. take into account the adverse effect of the high temperature and multiply the basic load ratings by the temperature factor indicated in Fig.1. In addition, the Ball Spline must be of a high

temperature type. Note) If the environment temperature exceeds 80°C, high-



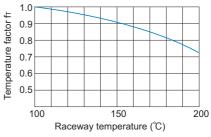


Fig.1 Temperature Factor (f_T)

■fc: Contact Factor

When multiple spline nuts are used in close con-

tact with each other, their linear motion is affected by moments and mounting accuracy, making it difficult to achieve uniform load distribution. In such applications, multiply the basic load rating (C) and (C₀) by the corresponding contact factor in Table3. Note) If uneven load distribution is expected in a large machine, take into account the respective contact factor indicated in Table3.

■fw: Load Factor

In general, reciprocating machines tend to involve vibrations or impact during operation. It is extremely difficult to accurately determine vibrations generated during high-speed operation and impact during frequent start and stop. When loads applied on a Ball Spline cannot be measured, or when speed and impact have a significant influence, divide the basic load rating (C or C₀), by the corresponding load factor in the table of empirically obtained data on Table4.

Table3 Contact Factor (f_c)

Number of spline nuts in close contact with each other	Contact factor fo	
2	0.81	
3	0.72	
4	0.66	
5	0.61	
Normal use	1	

Table4 Load Factor (fw)

Vibrations/ impact	Speed (V)	f _w		
Faint	Very low V≦0.25m/s	1 to 1.2		
Weak	Slow 0.25 <v≦1m s<="" td=""><td>1.2 to 1.5</td></v≦1m>	1.2 to 1.5		
Medium	Medium 1 <v≦2m s<="" td=""><td>1.5 to 2</td></v≦2m>	1.5 to 2		
Strong	High V>2m/s	2 to 3.5		

[Calculating the Average Load]

When the load applied on the spline shaft fluctuates according to varying conditions, such as an industrial robot arm traveling forward while holding a workpiece and traveling backward with empty weight, and a machine tool handling various workpieces, this varying load condition must be taken into account in service life calculation.

The average load (P_m) is a constant load under which the service life of an operating Ball Spline with its spline nut receiving a fluctuation load in varying conditions is equivalent to the service life under this varying load condition.

The following is the basic equation.

$$\mathbf{P}_{m} = \sqrt[3]{\frac{1}{\mathbf{L}} \cdot \sum_{n=1}^{n} (\mathbf{P}_{n}^{3} \cdot \mathbf{L}_{n})}$$

$$\mathbf{P}_{m} : \text{Average Load} \qquad (N)$$

$$\mathbf{P}_{n} : \text{Varying load} \qquad (N)$$

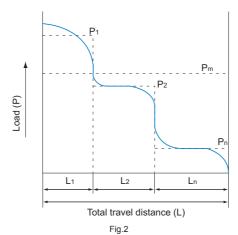
$$\mathbf{L} : \text{Total travel distance} \qquad (mm)$$

$$\mathbf{L}_{n} : \text{Distance traveled under } \mathbf{P}_{n} \qquad (mm)$$

When the Load Fluctuates Stepwise

$$P_{m} = \sqrt[3]{\frac{1}{L} (P_{1}^{3} \cdot L_{1} + P_{2}^{3} \cdot L_{2} \cdot \cdots + P_{n}^{3} \cdot L_{n})} \cdots \cdots (12)$$

 $\begin{array}{llll} P_m & : Average \ Load & (N) \\ P_n & : Varying \ load & (N) \\ L & : Total \ travel \ distance & (m) \\ L_n & : Distance \ traveled \ under \ load \ P_n \ \ (m) \end{array}$



Predicting the Service Life

When the Load Fluctuates Monotonically

$$\mathbf{P}_{m} \doteq \frac{1}{3} \left(\mathbf{P}_{min} + 2 \cdot \mathbf{P}_{max} \right) \cdots \cdots (13)$$

 $P_{\mbox{\tiny min}}$: Minimum load

(N)

P_{max}: Maximum load

(N)

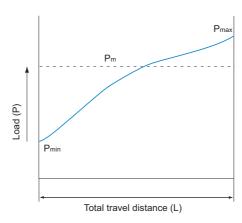
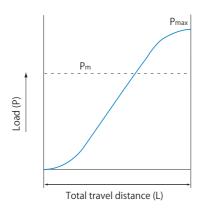


Fig.3

• When the Load Fluctuates Sinusoidally

(a)
$$P_m = 0.65P_{max} \cdots (14)$$



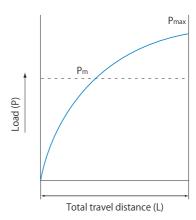


Fig.4

[Equivalent Factor]

Table11 on A3-26, Table12 on A3-27 and Table13 on A3-28 show equivalent radial load factors calculated under a moment load.

Example of Calculating the Service Life

Example of Calculation - 1

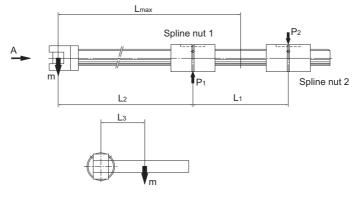
An industrial robot arm (horizontal)

[Conditions]

Mass applied to the arm end m=50kg Stroke ℓ_s =200mm Arm length at maximum stroke Lmax=400mm

L₂=325mm L3=50mm

Spline nut mounting span (estimate) L₁=150mm



A arrow view

(The Ball Spline type is LBS in this example.)

Fig.5

■Shaft Strength Calculation

Calculate the bending moment (M) and the torsion moment (T) applied on the shaft.

 $M=m \times 9.8 \times L_{max} = 196000N-mm$

 $T=m \times 9.8 \times L_3 = 24500N-mm$

Since the bending and torsion moments are applied simultaneously, obtain the corresponding bending moment (M_e) and torsion moment (T_e), and then determine the shaft diameter based on the greater value. From equations (3) and (4) on **B3-15**,

∴
$$T_e = \tau_a \times Z_P$$
 Hence,

$$Z_P = \frac{T_e}{T_a} = 4031 \text{mm}^3$$

Thus, judging from Table4 on △3-18, the nominal shaft diameter that meets Z_p is at least 40 mm.

Example of Calculating the Service Life

■Average Load P_m

Obtain an applied load value when the arm is extended to the maximum length (P_{max}), and another when the arm is contracted (P_{min}). Based on the values obtained, calculate the average load on the spline shaft nut.

$$P_{1max} = \frac{m \times 9.8(L_1 + L_2)}{L_1} \stackrel{.}{=} 1551.7N$$

$$P_{2max} = \frac{m \times 9.8 \times L_2}{L_1} \quad \ \ \dot{=} \ \, 1061.7N$$

When the arm is contracted

$$P_{1min} = \frac{m \times 9.8 \times ((L_2 - \ell_8) + L_1)}{L_1} \stackrel{.}{=} 898.3N$$

$$P_{2min} = \frac{m \times 9.8 \times (L_2 - \ell_s)}{L_1} \qquad = 408.3N$$

As this load is monotonically varying as shown in the Fig.3 on **3-23**, calculate the average load using the equation (13) on **3-23**.

The average load (P_{1m}) on spline nut 1

$$P_{1m} \doteq \frac{1}{3} (P_{1min} + 2P_{1max}) = 1333.9N$$

The average load (P_{2m}) on spline nut 2

$$P_{2m} = \frac{1}{3}(P_{2min} + 2P_{2max}) = 843.9N$$

Obtain the torque applied on one spline nut.

$$T = \frac{m \times 9.8 \times L_3}{2} = 12250N \cdot mm$$

Since the radial load and the torque are simultaneously applied, calculate the equivalent radial load using equation (9) on **3-20**.

$$P_{1E} = P_{1m} + \frac{4 \times T}{3 \times dp \times cos\alpha} = 1911.4N$$

$$P_{2E} = P_{2m} + \frac{4 \times T}{3 \times dp \times cos\alpha} = 1421.4N$$

■Nominal Life L_n

Based on the nominal life equation (8) on **3-19**, each nominal life is obtained as follows.

Nominal life of the spline nut L₁ =
$$\left(\frac{f_T \times f_C}{f_W} \times \frac{C}{P_{1E}}\right)^3 \times 50 = 68867.4 \text{km}$$

Nominal life of the spline nut
$$L_2 = \left(\frac{f_T \times f_C}{f_W} \times \frac{C}{P_{2E}}\right)^3 \times 50 = 167463.2 \text{km}$$

- f_T : Temperature factor = 1 (from Fig.1 on **B3-21**)
- f_c: Contact factor = 1 (from Table3 on **B3-21**)
- fw: Load factor = 1.5 (from Table4 on **B3-21**)
- C: Basic dynamic load rating = 31.9 kN (model LBS40)

Given the nominal life obtained for each spline nut above, the nominal life of the Ball Spline unit is equal to that of spline nut 1, which is 68867.4km.

Example of Calculation - 2

[Conditions]

Thrust position: Fs

Stroke velocity: V_{max} = 0.25m/sec Acceleration: a=0.36m/sec²

(from the respective velocity diagram)

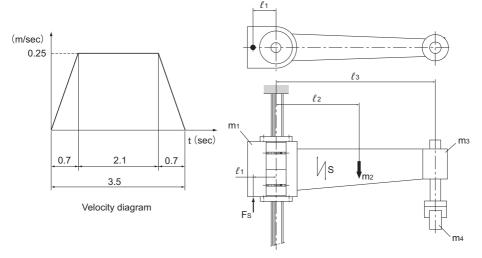
Stroke: S=700mm Housing mass: m₁=30kg Arm mass: m₂=20kg Head mass: m₃=15kg Work mass: m₄=12kg Distance from the thrust position to each mass

 ℓ_1 =200mm ℓ_2 =500mm

 ℓ_3 =1276mm

Cycle (1 cycle: 30 sec)

- 1. Descent (3.5sec) 2.Dwell (1sec): with a work
- 3. Ascend (3.5sec) 4.Dwell (7sec)
- 5. Descent (3.5sec) 6.Dwell (1sec): without a work
- 7. Ascend (3.5sec) 8.Dwell (7sec)



(The Ball Spline type is LBF in this example.)

Fig.6

Point of Selection

Example of Calculating the Service Life

■Shaft Strength Calculation

Calculate the shaft strength while assuming the shaft diameter to be 60 mm. (with double spline nut in contact with each other)

■Calculating the Moment (M_n) Applying on the Spline Nut during Acceleration, Uniform Motion and Deceleration with Different Masses (mn)

Applied moment during deceleration: M₁

$$M_1 = m_n \times 9.8 \left(1 \pm \frac{a}{g}\right) \times \ell_n$$
 ······(a)

Applied moment during uniform motion: M2

$$M_2 = m_n \times 9.8 \times \ell_n$$
(b)

Applied moment during deceleration: M₃

$$M_3 = m_n \times 9.8 \left(1 \pm \frac{a}{g}\right) \times \ell_n$$
 ······(c)

m_s: Mass

a : Acceleration

(m/sec2)

g: Gravitational acceleration (m/sec2)

 ℓ_0 : Offset from each loading point to the trust center (mm)

Assume:

$$A = \left(1 + \frac{a}{g}\right), B = \left(1 - \frac{a}{g}\right)$$

During descent

From equation (c), during acceleration

$$M_{m1} = m_1 \times 9.8 \times B \times \ell_1 + m_2 \times 9.8 \times B \times (\ell_1 + \ell_2) + m_3 \times 9.8 \times B \times (\ell_1 + \ell_3)$$

=398105.01N-mm

From equation (b), during uniform motion

$$M_{\text{\tiny M2}} \quad = \! m_1 \! \times \! 9.8 \! \times \! \ell_1 \! + \! m_2 \! \times \! 9.8 \! \times \! (\ell_1 \! + \! \ell_2) \! + \! m_3 \! \times \! 9.8 \! \times \! (\ell_1 \! + \! \ell_3)$$

=412972N-mm

From equation (a), during deceleration

$$\begin{array}{lll} M_{\text{m3}} &= m_1 \times 9.8 \times A \times \ell_1 + m_2 \times 9.8 \times A \times (\ell_1 + \ell_2) + m_3 \times 9.8 \times A \times (\ell_1 + \ell_3) \\ &= 427838.99 N - mm \end{array}$$

During ascent

From equation (a), during acceleration

$$M_{m1}' = m_1 \times 9.8 \times A \times \ell_1 + m_2 \times 9.8 \times A \times (\ell_1 + \ell_2) + m_3 \times 9.8 \times A \times (\ell_1 + \ell_3)$$

=427838.99N-mm

From equation (b), during uniform motion

$$M_{m2}$$
' = $m_1 \times 9.8 \times \ell_1 + m_2 \times 9.8 \times (\ell_1 + \ell_2) + m_3 \times (\ell_1 + \ell_3)$
=412972N-mm

From equation (c), during deceleration

 $\begin{array}{ll} M_{m3}' &= m_1 \times 9.8 \times B \times \ell_1 + m_2 \times 9.8 \times B \times (\ell_1 + \ell_2) + m_3 \times 9.8 \times B \times (\ell_1 + \ell_3) \\ &= 398105.01 \text{N-mm} \end{array}$

• During descent (with a work loaded)

From equation (c), during acceleration

 M_{m1} " = M_{m1} + M_4 × 9.8 × B × (ℓ_1 + ℓ_3)

=565433.83N-mm

From equation (b), during uniform motion

 M_{m2} " = M_{m2} + m_4 × 9.8 × (ℓ_1 + ℓ_3)

=586549.6N-mm

From equation (a), during deceleration

 M_{m3} " = $M_{m3} + m_4 \times 9.8 \times A \times (\ell_1 + \ell_3)$

=607665.37N-mm

• During ascent (with a work loaded)

From equation (a), during acceleration

 M_{m1} " = M_{m1} '+ M_{m1} × 9.8 × A × (ℓ_1 + ℓ_3)

=607665.37N-mm

From equation (b), during uniform motion

 M_{m2} " = M_{m2} '+ $m_4 \times 9.8 \times (\ell_1 + \ell_3)$

=586549.6N-mm

From equation (c), during deceleration

 $M_{m3}^{""} = M_{m3}^{"} + m_4 \times 9.8 \times B \times (\ell_1 + \ell_3)$

=565433.83N-mm

 $M_1=M_{m1}=M_{m3}'=398105.01N-mm$

M₂=M_{m2}=M_{m2}'=412972N-mm

 $M_3=M_{m3}=M_{m1}$ '=427838.99N-mm

M₁'=M_{m1}"=M_{m3}"=565433.83N-mm

M₂'=M_{m2}''=M_{m2}'''=586549.6N-mm

 $M_3'=M_{m3}''=M_{m1}'''=607665.37N-mm$

Point of Selection

Example of Calculating the Service Life

■Calculating the Equivalent Radial Load Considered to be Applied to the Spline Nut with Different Moments Relational expression between moment Mn and Pn

 $P_n = M_n \times K$(d)

: Equivalent radial load (N) : Applied moment (N-mm)

Κ : Equivalent factor

(from Table12 to **A3-27**)

(If two spline nuts of LBF60 contact with each other, K = 0.013)

Calculate the equivalent radial load with different applied moments using equation (d).

 $P_{m1} = P_{m3}' = M_1 \times 0.013 = 5175.4N$

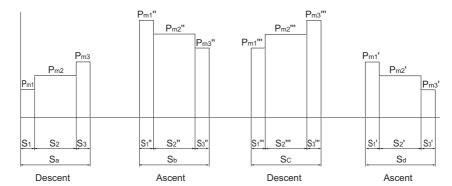
 $P_{m2} = P_{m2}' = M_2 \times 0.013 = 5368.6N$

 $P_{m3} = P_{m1}' = M_3 \times 0.013 = 5561.9N$

 P_{m1} " = P_{m3} " = M_1 ' × 0.013 \doteqdot 7350.7N

 P_{m2} " = P_{m2} " = M_2 ' × 0.013 \rightleftharpoons 7625.2N

 P_{m3} " = P_{m1} " = M_3 ' × 0.013 \rightleftharpoons 7899.7N



$$\begin{cases} P_1 = P_{m1} = P_{m3}' \stackrel{.}{=} 5175.4N \\ P_2 = P_{m2} = P_{m2}' \stackrel{.}{=} 5368.6N \\ P_3 = P_{m3} = P_{m1}' \stackrel{.}{=} 5561.9N \end{cases}$$

$$P_4 = P_{m1}" = P_{m3}"" = 7350.7N$$

 $P_5 = P_{m2}" = P_{m2}"" = 7625.2N$
 $P_6 = P_{m3}" = P_{m1}"" = 7899.7N$

$$(S = S_a = S_b = S_c = S_d = 700 mm)$$

 $S_1 = S_1 = S_1' = S_1'' = 87.5 mm$
 $S_2 = S_2 = S_2' = S_2'' = 525 mm$
 $S_3 = S_3 = S_3' = S_3'' = 87.5 mm$

■Calculating the Average Load Pm

Using equation (12) on **B3-22**.

$$P_{m} = \sqrt[3]{\frac{1}{4 \times S}} \left[2 \left\{ (P_{1}^{3} \times S_{1}) + (P_{2}^{3} \times S_{2}) + (P_{3}^{3} \times S_{3}) \right\} + 2 \left\{ (P_{4}^{3} \times S_{3}) + (P_{5}^{3} \times S_{2}) + (P_{6}^{3} \times S_{1}) \right\} \right]$$

$$\stackrel{=}{\rightleftharpoons} 6689.5N$$

■Calculating the Rated Life L from the Average Load

Using equation (8) on **B3-19**,

$$L = \left(\frac{f_{T} \cdot f_{C}}{f_{W}} \cdot \frac{C}{P_{m}}\right)^{3} \times 50$$

= 7630 km

f⊤ : Temperature factor = 1

(from Fig.1 on **B3-21**)

: Contact factor=0.81

(from Table3 on **B3-21**)

fw : Load factor=1.5

(from Table4 on **B3-21**)

C : Basic dynamic load rating = 66.2 kN

(model LBF60)

Given the result above, the nominal life of model LBF60 with double spline nuts used in close contact with each other is 7,630 km.

Assembling the Ball Spline

Mounting the Spline

Fig.1 and Fig.2 shows examples of mounting the spline nut. Although the Ball Spline does not require a large strength for securing it in the spline shaft direction, do not support the spline only with driving fitting.

Note) On both ends of the spline nut of Caged Ball Ball Spline model SLS, resin end caps are installed. Hitting them or pressing hard may cause damage. You must take care not to apply an excessive load.

Straight nut type

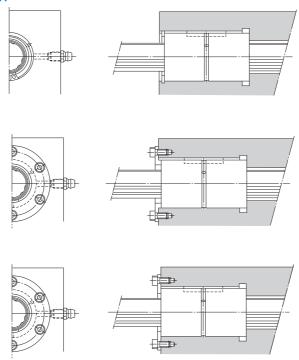
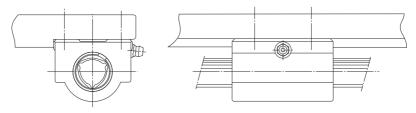


Fig.1 Examples of Fitting the Spline Nut

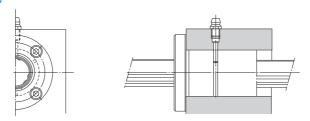
Mounting Procedure and Maintenance

Assembling the Ball Spline

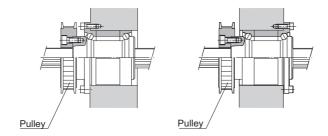
Model LBH



Flanged type



Model LTR



Model LBG

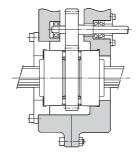


Fig.2 Examples of Fitting the Spline Nut

Installing the Spline Nut

When installing the spline nut into the housing, do not hit the side plate or the seal, but gently insert it using a jig (Fig.3).

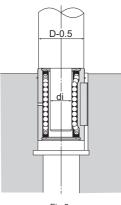


Fig.3

Table1 Dimensions of the Jig for Model LBS

Unit: mm

Nominal shaft diameter	15	20	25	30	40	50	60	70	85	100	120	150
di	12.5	16.1	20.3	24.4	32.4	40.1	47.8	55.9	69.3	83.8	103.8	131.8

Table2 Dimensions of the Jig for Model LT

Unit: mm

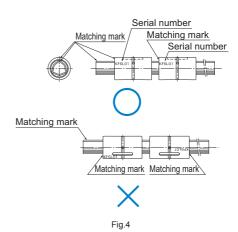
Nominal shaft diameter	6	8	10	13	16	20	25	30	40	50	60	80	100
di	5.0	7	8.5	11.5	14.5	18.5	23	28	37.5	46.5	56	75.5	94.5

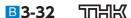
Installation of the Spline Shaft

When installing the spline shaft into the spline nut, identify the matching marks (Fig.4) on the spline shaft and the spline nut, and then insert the shaft straightforward while checking their relative positions.

Note that forcibly inserting the shaft may cause balls to fall off.

If the spline nut is attached with a seal or given a preload, apply a lubricant to the outer surface of the spline shaft.





Mounting Procedure and Maintenance

Lubrication

Lubrication

To prevent foreign material from entering the spline nut and the lubricant from leaking, special synthetic resin seals with high wear resistance are available for the Ball Spline.

Spline nuts with seals (seal for both ends type UU, and seal for one end) contain high-quality lithium-soap group grease No. 2. However, if using them at high speed or with a long stroke, replenish grease of the same type through the greasing hole on the spline nut after running in.

Afterward, replenish grease of the same type as necessary according to the service conditions.

The greasing interval differs depending on the conditions. Normally, replenish the lubricant (or replace the product) roughly every 100 km of travel distance (six months to one year) as a rule of thumb.

For a Ball Spline model type without a seal, apply grease to the interior of the spline nut or to the raceways of the spline shaft.



Material and Surface Treatment

Depending on the service environment, the Ball Spline requires anticorrosive treatment or a different material. For details of anticorrosive treatment and material change, contact THK.

Contamination Protection

Entrance of dust or other foreign material into the spline nut will cause abnormal wear or shorten the service life. Therefore, it is necessary to prevent detrimental foreign material from entering the Ball Spline. When entrance of dust or other foreign material is a possibility, it is important to select effective seals and/or dust-control device that meets the environment conditions.

For the Ball Spline, a special synthetic rubber seal that is highly resistant to wear is available as a contamination protection accessory. If desiring a higher contamination protection effect, a felt seal is also available for some types. For details about the felt seal, contact THK. In addition, THK produces round bellows. Contact us for details.

Table1 Dust prevention accessory symbol

Symbol	Contamination protection accessory							
No Symbol	Without seal							
UU Rubber seal attached on both ends of spline nut								
U	Rubber seal attached on either end of spline nut							
DD	Felt seal attached on both ends of spline nut							
D	Felt seal attached on either end of spline nut							
ZZ	Rubber seal attached on both ends of support bearings							
Z	Rubber seal attached on either end of support bearings							

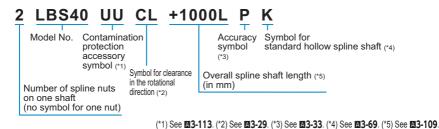
Model No. Ball Spline

Model Number Coding

Model number configurations differ depending on the model features. Refer to the corresponding sample model number configuration.

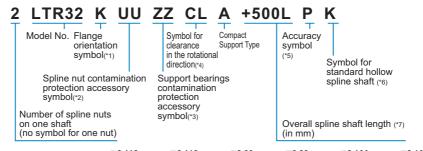
[Ball Spline]

Models SLS, SLS-L, SLF, LBS, LBST, LBF, LBR, LBH, LT, LF and LT-X



[Rotary Ball Spline]

Models LTR, LTR-A, LBG and LBGT



(*2) See M3-113. (*3) See M3-113. (*4) See M3-29. (*5) See M3-33. (*6) See M3-106. (*7) See M3-109.

(*1) No Symbol: standard K: flange inversed

[Handling]

- (1) Please use at least two people to move any product weighing 20 kg or more, or use a dolly or another conveyance. Doing so may cause injury or damage.
- (2) Do not disassemble the parts. This will result in loss of functionality.
- (3) Tilting a spline nut or spline shaft may cause them to fall by their own weight.
- (4) Take care not to drop or strike the Ball Spline. Doing so may cause injury or damage. Giving an impact to it could also cause damage to its function even if the product looks intact.
- (5) When assembling, do not remove the spline nut from the spline shaft.
- (6) When handling the product, wear protective gloves, safety shoes, etc., as necessary to ensure safety.

[Precautions on Use]

- (1) Prevent foreign material, such as cutting chips or coolant, from entering the product. Failure to do so may cause damage.
- (2) If the product is used in an environment where cutting chips, coolant, corrosive solvents, water, etc., may enter the product, use bellows, covers, etc., to prevent them from entering the product.
- (3) Do not use the product at temperature of 80°C or higher. Except for the heat-resistant models, exposure to higher temperatures may cause the resin/rubber parts to deform/be damaged.
- (4) If foreign material such as cutting chips adheres to the product, replenish the lubricant after cleaning the product.
- (5) Micro-strokes tend to obstruct oil film to form on the raceway in contact with the rolling element, and may lead to fretting corrosion. Take consideration using grease offering excellent fretting prevention. It is also recommended that a stroke movement corresponding to the length of the spline nut be made on a regular basis to make sure oil film is formed between the raceway and rolling element.
- (6) Do not use undue force when fitting parts (pin, key, etc.) to the product. This may generate permanent deformation on the raceway, leading to loss of functionality.
- (7) Skewing or misalignment of the spline shaft support and spline nut can shorten service life substantially. Inspect the components carefully and make sure they are mounted correctly.
- (8) The spline nut must contain all its internal rolling elements (balls) when mounted on the spline shaft. Using a spline nut with any balls removed may result in premature damage.
- (9) Please contact THK if any balls fall out of the spline nut; do not use the spline nut if any balls are missing.
- (10) To mount the spline nut on the spline shaft, first locate the alignment indicators on both components, then insert the shaft through the opening in the spline nut, without forcing it, and adjust the position until the indicators are aligned. Forcing the shaft could cause balls to fall out. When mounting a spline nut equipped with a seal or preload, first lubricate the outer surface of the spline shaft.
- (11) Manipulate the spline nut gently, using a jig, when inserting it into the housing, taking care not to strike the side plate, end cap, or seal.
- (12) If an attached component is insufficiently rigid or mounted incorrectly, the bearing load will be concentrated at one location and performance will decline significantly. Make sure the housing and base are sufficiently rigid, the anchoring bolts are strong enough, and the component is mounted correctly.
- (13) If desiring to have a flanged-type Ball Spline additionally machined, such as having a dowel pin hole, contact THK.

Precautions on Use

[Lubrication]

- (1) Thoroughly wipe off anti-rust oil and feed lubricant before using the product.
- (2) Do not combine different lubricants. Mixing lubricants can cause adverse interaction between disparate additives or other ingredients.
- (3) If the product will be exposed to constant vibration or high or low temperatures, or used in a clean room, vacuum, or other special environment, apply a lubricant suitable for both the specifications and the environment.
- (4) To lubricate a product that has no grease nipple or oil hole, apply lubricant directly to the raceway surface and execute a few preliminary strokes to ensure that the interior is fully lubricated.
- (5) Bear in mind that the Ball Spline's slide resistance is affected by changes in the consistency of the lubricant, which varies according to the temperature.
- (6) The Ball Spline may encounter increased slide resistance following lubrication, due to the lubricant's agitation resistance. Make sure to put the unit through some preliminary motions to ensure that it is fully lubricated before starting up the machine.
- (7) Excess lubricant may spatter immediately after lubrication. If necessary, wipe off any spattered grease.
- (8) Because lubricant performance declines over time, lubrication must be monitored regularly and fresh lubricant applied when needed, depending on how frequently the machine is operated.
- (9) The appropriate lubrication schedule will depend on usage conditions and the surrounding environment. In general, the unit should be lubricated after every 100 kilometers of operation (every 3 to 6 months). The actual lubrication schedule and amount of lubricant used should be determined by the condition of the machinery.
- (10) With oil lubrication, the lubricant may not always be thoroughly disseminated inside the Ball Spline, depending on its mounting position. If the preferred lubrication method is oil lubrication, please consult THK in advance.

[Storage]

When storing the Ball Spline, enclose it in a package designated by THK and store it in a room in a horizontal orientation while avoiding high temperature, low temperature and high humidity.

After the product has been in storage for an extended period of time, lubricant inside may have deteriorated, so add new lubricant before use.

[Disposal]

Dispose of the product properly as industrial waste.