## Ice Engineering Guidelines – US Army Corps of Engineers

# CHAPTER 6 Ice Force on Structures

#### 6-1. Introduction the engine of the visit of the control of the co

- a. Any structure placed in an environment where the presence of ice is a hazard to its integrity and stability needs to be designed to withstand the forces generated by ice moving against it. A designer should also consider how the cold may affect the intended operations of a structure, because freezing of ice may hinder some of the normal warm weather operations. These guidelines are intended for structures placed in inland waters, e.g., lakes, rivers, and coastal waters. To estimate ice forces on an offshore structure see API (1995).
- b. An ice sheet moves under the influence of shear stresses imparted by wind and water and by thermal expansion (as long as the ice sheet is intact). It transmits the accumulated forces to a structure situated in its path. The shear drag forces attributable to wind and water can be transmitted over large distances through an intact ice cover. In many situations, these environmental forces can be large, and the ice sheet fails during its interaction with a structure. The ice failure process limits the large environmental force being transmitted to the structure. Unless the environmental forces can be estimated with confidence to be small, the methodology to estimate ice forces from floating ice is generally to determine the forces required to fail an ice sheet in the vicinity of a structure. An ice sheet fails by crushing, splitting, bending, buckling, or a combination of these modes. For a given failure mode and structure shape, theoretical formulations or experimental results, along with ice properties, are used to estimate the forces required to fail an ice sheet. The forces are estimated for one, two, or all possible failure modes, and the failure mode with the lowest estimated force is assumed to occur at the ice-structure interface. At times, it may be necessary to conduct model tests to simulate an ice-structure interaction to determine the interaction forces. Attention should also be given to the clearance of broken ice pieces, because the advancing ice sheet will interact with the broken pieces if they accumulate in front of the structure. It is also possible that the accumulation of broken ice pieces may freeze together to form a grounded collar, which may provide some protection from further ice movement.
- c. In situations where an ice cover is made up of drifting ice floes, the impact of these floes causes a horizontal force on a structure. (Although impact from a drifting iceberg falls into this category, we will limit our discussion to drifting ice floes.) The forces generated when ice floes strike a structure depend on the mass and the initial velocity of the floes. If the kinetic energy of the moving ice floes is greater than the work done in failing the ice along the entire width of the structure, the design force is then limited by the ice failure processes mentioned above. If the kinetic energy and the momentum of drifting ice floes are small, resulting in indentation of the structure into the ice floes over a part of its width, ice forces are estimated from balancing the momentum and the energy before and after an impact.
- d. The methodology given in this Manual for estimating ice forces is based on the results of theoretical and experimental research in ice mechanics and measurements of ice forces in the field. Most recently, our understanding of processes active during crushing of ice at various indentation speeds has been increased. Data on measured ice forces on large structures have re-

cently been published. Except for the recommended values of effective pressure, the Corps guidelines for ice forces on structures are almost the same as those of the American Association of State Highway and Transportation Officials (AASHTO 1994), which in turn were adopted from the Canadian Standards Association (CSA 1988, 2000). Montgomery et al. (1984) provide the background information for the recommendations in CSA (1988). The CSA (2000) and the AASHTO (1994) codes consider dynamic and static loads on bridge piers located in rivers, lakes, and coastal waters. The dynamic loads develop when moving ice fails against a pier during spring breakup, or when currents and wind move ice sheets past piers at other times of the year. The static loads are generated by thermal expansion or contraction of the ice and by fluctuations in the water levels.

#### 6-2. Mechanical Properties of Ice

#### a. Introduction.

- (1) Because the forces necessary to fail an ice sheet depend on the mechanical properties of ice, the mechanical properties of the freshwater and sea ice are briefly reviewed below before methodologies to estimate the ice forces on a structure are given. Ice is a unique material. In the temperature range under which it is normally encountered, it is very close to its melting point. Ice can creep with very little applied stress, or it can fracture catastrophically under a high strain rate.
- (2) There are two primary ways to categorize ice. One is based on the melt from which the ice is grown (freshwater or sea water), and the other is based on the size of the ice blocks (i.e., large ice floes or accumulations of broken ice in a random ice rubble). The conditions under which ice forms will determine its grain structure, with common forms being frazil ice, columnar ice, discontinuous columnar ice, and granular ice. Both the porosity within the ice and the grain structure significantly influence the mechanical properties of the ice. Various books (e.g., Michel 1978, Ashton 1986, Cammaert and Muggeridge 1988, Sanderson 1988) cover the subjects of formation and types of ice, as well as ice properties.
- (3) The porosity attributable to brine and air pockets affects the ice properties. The brine volume  $v_b$  ( $^{\circ}/_{\circ o}$ ) is obtained from the following relation (Frankenstein and Garner 1967):

$$\nu_b = S_i (0.532 + 49.185/|T|)$$
 (6-1)

where  $S_i$  (°/ $_{\infty}$ ) = salinity, T (°C) = temperature of the ice, and the symbol °/ $_{\infty}$  refers to parts per thousand.

(4) The porosity ascribable to air can be obtained from the following relation after the bulk density  $\rho$  of ice containing salt and air are measured (Cox and Weeks 1983)

$$V_{\rm a}/V = 1 - \rho/\rho_{\rm i} + \rho S_{\rm i} F_2(T)/F_1(T)$$
 (6-2)

where

 $V_a$  = volume of air

V = bulk volume

 $\rho_i$  = density of pure ice

 $S_i = \text{salinity of ice}$ 

 $F_1(T)$  and  $F_2(T)$  = functions of temperature derived from a phase equilibrium table (Cox and Weeks 1983) and given in Figure 6-1.

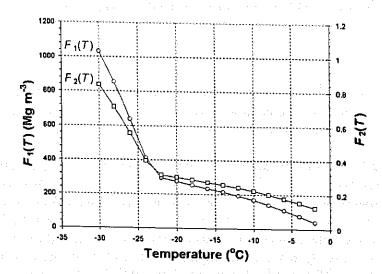


Figure 6-1. Plots of  $F_1(T)$  and  $F_2(T)$  with respect to temperature. To convert degrees C to degrees F use the following:  ${}^{\circ}F = {}^{\circ}C \times 1.8 + 32$ .

- b. Compressive Strength. Values of the uni-axial compressive strength for ice range from 0.5 to 20 MPa (72.5 to 2900 psi). The strength is a function of strain rate, temperature, grain size, grain structure, and porosity. Analyses of strength measurements have shown that the strength increases with strain rate, up to a rate of  $10^{-3}$  s<sup>-1</sup>, whereupon the strength generally decreases at higher strain rates because of brittle fracture.
- (1) In the lower strain rate range below  $10^{-3}$  s<sup>-1</sup>, the compressive strength of freshwater ice is given by (Sinha et al. 1987)

$$\sigma_{\rm c} = 212\dot{\varepsilon}^{0.34} \quad (3.07 \times 10^4 \ \dot{\varepsilon}^{0.34}) \tag{6-3}$$

where  $\sigma_c$  is in MPa (psi) and  $\dot{\varepsilon}$  is in s<sup>-1</sup>.

- (2) The above expression is for the compressive strength of ice at  $-10^{\circ}$ C (263 K or 14°F). The compressive strength at another temperature T(K) can be obtained by multiplying the strength at  $-10^{\circ}$ C (14°F) by a correction factor  $[\exp\{(Q/R)(263-T)/(263T)\}]^{1/3}$ , where Q = 65 kJ mol<sup>-1</sup> (61.6 Btu mol<sup>-1</sup>) (the activation energy for columnar ice) and R = 8.314 J mol<sup>-1</sup> K<sup>-1</sup> (1.986 Btu lb<sup>-1</sup> mol<sup>-1</sup> R<sup>-1</sup>) (the universal gas constant).
- (3) For sea ice, the following equations for compressive strength were derived from an analysis of over 400 small sample tests (Timco and Frederking 1990). These equations are:

$$\sigma_{\rm c} = 37\dot{\varepsilon}^{0.22} \left[ 1 - (v_{\rm T}/270)^{0.5} \right]$$
 for horizontally loaded columnar sea ice (6-4)

$$\sigma_{\rm c} = 160\dot{\varepsilon}^{0.22} \left[ 1 - (v_{\rm T}/200)^{0.5} \right]$$
 for vertically loaded columnar sea ice (6-5)

$$\sigma_{\rm c} = 49 \dot{\varepsilon}^{0.22} \left[ 1 - (v_{\rm T} / 280)^{0.5} \right]$$
 for granular sea ice (6-6)

where  $\dot{\varepsilon}$  is the strain rate in s<sup>-1</sup>, and  $v_{\rm T}$  is the total porosity in the ice (brine and air) in parts per thousand. The range of strain rate for these equations is  $10^{-7}$  to  $10^{-4}$  s<sup>-1</sup>. Above this strain rate, the ice can experience brittle failure with compressive strengths exhibiting a wide range of variability.

c. Flexural Strength. The flexural strength is generally lower than the compressive strength. Measurements on freshwater ice range from 0.5 to 3 MPa (72.5 to 435 psi), with an average of 1.73 MPa (for temperatures less than -5°C (23°F) (Timco and O'Brien 1994). There is very little temperature or strain rate dependence, but there is a wide scatter in the measured flexural strength with higher values from smaller samples. At temperatures close to 0°C (32°F), the strength of freshwater ice can be essentially zero if solar radiation has caused pronounced "candling." For sea ice, Timco and O'Brien (1994) compiled the results of over 900 flexural strength measurements to obtain the following dependence of the flexural strength on the brine volume.

$$\sigma_{\rm f} = 1.76e^{-5.88\sqrt{\nu_{\rm b}}} \quad (255 e^{-5.88\sqrt{\nu_{\rm b}}}) \tag{6-7}$$

where  $\sigma_f$  is in MPa (psi) and  $\nu_b$  is the brine volume fraction. The strength value for zero brine volume (1.76 MPa or 255.3 psi) agrees with the average value of 1.73 MPa (250.9 psi) determined from tests on freshwater ice.

- d. Fracture Toughness. The fracture toughness depends on the loading rate and the ice type, with less variation ascribable to temperature and grain size. Typical values for freshwater ice range from  $109 \pm 8$  kPa m<sup>0.5</sup> (0.01581  $\pm$  0.00116 ksi in.<sup>0.5</sup>), for columnar-grained S2 ice, to  $151 \pm 12$  kPa m<sup>0.5</sup> (0.0219  $\pm$  0.00174 ksi in<sup>0.5</sup>) for granular ice (Weber and Nixon 1992). In-situ measurements of the fracture properties of lake ice and sea ice revealed that fracture toughness depends on the size of the specimen, and that its range is 50–250 kPa m<sup>0.5</sup> (0.00725 to 0.03626 ksi in.<sup>0.5</sup>) (Dempsey et al. 1999a,b)
- e. Elastic Modulus. Ice deformation involves elastic and creep processes, and the large-scale modulus is usually discussed in terms of an "effective modulus" that incorporates these processes. This modulus is a strong function of loading rate, temperature, and grain size and type. The values of elastic modulus range from approximately 2 GPa  $(2.9 \times 10^5 \text{ psi})$  at low frequency loading to a high frequency value of 9 GPa  $(1.3 \times 10^6 \text{ psi})$  (Sinha et al 1987, Cole 1995a,b).
- f. Broken Ice Properties. Ice rubble is usually assumed to behave as a linear Mohr-Coulomb material, for which the shear stress  $\tau$  and the normal stress  $\sigma_n$  on a failure plane are related by

$$\tau = c + \sigma_{\rm n} \tan \phi \tag{6-8}$$

where c is the apparent cohesion and  $\phi$  is the effective angle of internal friction. Recent studies (Prodanovic 1979, Ettema and Urroz-Aguirre 1991, Løset and Sayed 1993, Cornett and Timco 1996) have shown that the yield envelope is non-linear but can be approximated with a linear envelope for a limited range of conditions; that cohesion is negligible for unconsolidated rubble; that  $\phi$  depends on the stress history and decreases with increasing pressure; that  $\phi$  is less than the maximum angle of repose; and that  $\phi$  depends on the strain path and pressure. Measured values of  $\phi$  range from 20° to 45°.

#### 6-3. Environmental Forces

a. Wind and Water Drag Forces. The drag force, caused by wind and water shear stresses on the top and bottom surfaces of an ice cover, can be estimated from the following expression:

$$F_d = C_d \rho A V^2 \tag{6-9}$$

where

 $C_{\rm d} = {\rm drag \ coefficient}$ 

density of air or water

A = fetch area

V = velocity of air or water measured at a certain distance above or below an ice cover.

Typical values for  $C_d$  are 0.002 for a smooth ice cover, and 0.005 for a rough ice cover (Banke and Smith 1973). Typical values for the density of air and water are 1.3 and 1000 kg m<sup>-3</sup> (0.08116 and 62.4 lb ft<sup>-3</sup>), respectively. When sufficient information is available on the wind and water velocities and the fetch area, it may be possible to estimate the wind and water drag forces. However, it is difficult, in most cases, to estimate the fetch area that contributes directly to wind and water drag forces on a structure. In most situations, the estimates of wind and water drag force are greater than the force required to fail an ice sheet, and the ice failure process limits the force to that necessary to fail the ice against the structure. If wind and water drag forces can be estimated to be less than the ice failure force, the design force on the structure is taken to be the estimate of wind and water drag forces.

- b. Thermal Ice Forces. Like other materials, ice expands with increasing temperature, and vice versa. However, unlike other materials, water expands when it changes phase from liquid to solid. These two properties, along with the creep of ice, explain the forces that develop when ice undergoes a temperature change. The temperature of ice changes because of conduction, radiation, and convection heat transfer at its surface. The depth to which temperature changes take place depends on the thickness of the ice cover, the presence or absence of snow on its top surface, and the environmental conditions (Michel 1970, 1978; Sanderson 1988).
- (1) An unrestricted ice cover will expand as a whole in response to a change in temperature. While the top layer of the ice sheet expands as a result of the change in temperature, the bottom layer, because it undergoes no temperature change, restrains the top layer from expanding. This process causes the rate of expansion of the ice sheet to depend on the ice thickness. If

one edge of the ice cover is fixed to a shore, the rest of the ice sheet expands away from the shore. A structure placed some distance away from the shore will experience an ice force as a result of ice moving past it. For distances of shores on the order of 50 kilometers (31 miles), observers have measured the ice edge to move at a rate of about 0.9 meter (3 feet) per day (Strilchuk 1977).

- (2) When an ice sheet is restricted from expansion from four or two sides, the confinement causes, respectively, biaxial or uniaxial stress (Sanderson 1984, 1988). The method of calculating thermal ice force in a confined ice sheet is as follows.
- (a) Calculate temperature change as a function of depth, taking into account heat transfer by conduction, radiation, and convection.
  - (b) Calculate the rate of thermal expansion  $\dot{\varepsilon}$  as if the ice would have been unconfined.
  - (c) Apply  $-\dot{\varepsilon}$  at those depths to satisfy the assumption of complete restraint.
- (d) Calculate the stress needed to deform the ice at those strain rates using one of Equations 6-3, 6-4, 6-5 or 6-6.
- (e) Integrate the stress through the ice thickness to obtain force per unit width. In the case of a confined ice sheet having thicknesses greater than 0.5 meter (1.6 feet), the force per unit width does not strongly depend on ice thickness, because the ice layer below the 0.5-meter (1.6-foot) depth does not undergo a change in temperature and restricts expansion of ice in the top layer. Calculations of typical thermal ice force are in the range of 200–400 kN m<sup>-1</sup> (1.5 × 10<sup>5</sup> to  $2.95 \times 10^5$  lb<sub>f</sub> ft<sup>-1</sup>), whereas some of the measured values are in the range of 100–350 kN m<sup>-1</sup> (7.4 × 10<sup>4</sup> to  $2.6 \times 10^5$  lb<sub>f</sub> ft<sup>-1</sup>) (Sanderson 1984).
- (3) The presence of cracks in an ice cover has a profound effect on thermally generated pressure within it. Metge (1976) observed three types of cracks: dry micro-cracks, wet micro-cracks, and wet large cracks. Dry micro-cracks are found at the top of an ice sheet and do not penetrate to the water below. Dry cracks close when an ice sheet thermally expands, and this closure of cracks does not result in a significant push against a structure. Wet micro-cracks are filled with water that freezes within them during cold periods. With repeated cooling, cracking, and freezing, a floating ice sheet can expand and push against a structure. Water within large wet cracks freezes only at the top surface during cold periods, and this creates a thin ice bridge across the gap of a crack. When the ice sheet expands during warm periods, these bridges are crushed, forming small pressure ridges along the crack.
- (4) In summary, factors influencing thermally generated ice forces are the magnitude and the rate of temperature increase, heat transfer at the top surface and in the ice sheet, boundaries resisting expansion of an ice cover, creep relaxation of ice pressure, and dry and wet cracks. Several theories (Rose 1947; Belkov 1973; Drouin and Michel 1974; Xu Bomeng 1981, 1986; Fransson 1988) have been proposed to calculate the thermally induced ice force, and thermally induced ice pressures have been reviewed by several authors (Michel 1970, Kjeldgaard and Carstens 1980, Sanderson 1984). More recently, the predicted loads of these five theories were compared to the

results of a comprehensive field data set (Comfort and Abdelnour 1994). The comparison of each model with measured data showed a wide disparity, and no model predicted the measured loads (Timco et al. 1996). The disparity between theoretical estimates and measured values of thrust on dam walls may be attributed to changes in water levels in reservoirs and large wet cracks in the ice cover.

- (5) In recent years, two measurement programs were launched.
- (a) Comfort et al. (2000a,b) undertook a 9-year program, beginning in 1991–92, to measure the loads in the ice sheet adjacent to eight dam sites in Manitoba, Ontario, Quebec, and Labrador.
- (b) Carter et al. (1998) undertook a 3-year program from 1995 to 1998 to measure the static ice forces in four reservoirs in central and northern Quebec. In both of these programs, changes in measured stress in an ice sheet correlated with changes in air temperature as well as water level.
- (6) Carter et al. (1998) proposed that thermal ice loads are limited by the instability of ice blocks between two or three parallel cracks along a dam wall. Their measurements indicate that the ice thrust changes with increasing water level; the maximum values was about 150 kN m $^{-1}$  (1.1 × 10<sup>5</sup> lb<sub>f</sub> ft<sup>-1</sup>). Comfort et al. (2000a, b) identified the importance of water level fluctuations to the ice loads on dam walls. They found the ice loads to be higher and more variable than those generated by thermal process alone when there were significant, but not excessive, water level changes. The range of ice thickness during their measurement program was 0.3-0.7 meters (1-2 feet). The maximum values of the measured line load resulting from thermal events with negligible change in water level at four dam sites in central and eastern Canada were in the range of 61 to 85 kN m  $(4.5 \times 10^4 \text{ to } 6.3 \times 10^4 \text{ lb}_f \text{ ft}^{-1})$ , with average value of 70 kN m<sup>-1</sup>  $(5.2 \times 10^4 \text{ lb}_f \text{ ft}^{-1})$  (Comfort and Armstrong 2001). Similar values resulting from thermal events, combined with significant change in water level at four dams in central and eastern Canada, were in the range of 52 to 374 kN m<sup>-1</sup>  $(3.8 \times 10^4 \text{ to } 2.8 \times 10^5 \text{ lb}_f \text{ ft}^{-1})$ , with average value of 186 kN m<sup>-1</sup> (1.4 × 10<sup>5</sup> lb<sub>f</sub> ft<sup>-1</sup>) (Comfort and Armstrong 2001). At Seven Sisters Dam in Manitoba, Comfort and Armstrong observed a significant reduction by a factor of 3-5 in ice thrust when the water level was lowered in early January by 45 centimeters (18 inches) and maintained there for the rest of the winter. More recently, they observed a similar reduction in ice thrust at the same site when the water level was lowered in late December or early January by 35 centimeters (14 inches) and then brought up to normal levels a few days later (Comfort and Armstrong 2001). These operations introduce large wet cracks and hinges in the ice sheets to limit the ice thrust to dam walls.

#### 6-4. Forces Limited by Ice Failure

a. Introduction. The force resulting when a moving ice sheet and a structure interact is limited to the magnitude of force necessary to fail the ice sheet in crushing, bending, buckling, splitting, or a combination of these modes. In the following, procedures to estimate forces to fail ice sheets in the above mentioned modes are given.

- (1) It is important to consider the magnitude of the area over which the ice forces act. The total force on the entire structure is important for designing foundations to resist sliding and overturning. Contact forces over small areas, or local contact pressures, are important for designing internal structural members and the external skin of a structure.
- (2) The total ice force F on a structure of width D attributable to failure of an ice sheet of thickness h, as shown in Figure 6-2, is expressed in terms of the effective pressure  $p_e$ :

$$F = p_{c}Dh \tag{6-10}$$

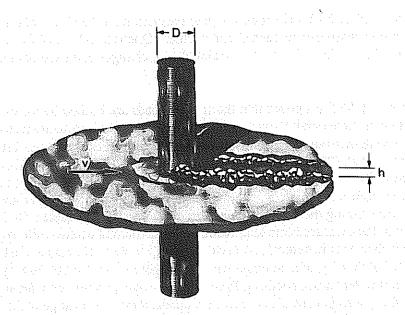


Figure 6-2. Total ice force F on a structure of width D attributable to failure of an ice sheet of thickness h.

- (3) It is important to distinguish between the contact pressure acting over the ice-structure contact area and the effective pressure. Because the actual contact area between a structure and an ice sheet is either equal to or less than the nominal contact area (product of ice thickness and structure width), the contact pressure is either equal to or higher than the effective pressure. The relative speed of an ice sheet with respect to a structure is an important factor in determining the mode of ice failure and the resulting contact areas and effective pressures.
- b. Crushing Failure. Ice crushing is one of the common modes of ice failure. Much work has been done to understand the processes taking place during ice crushing and to determine the forces generated at the interface. Because ice exists close to its melting temperature, its temperature strongly affects its properties. Ice at lower temperatures is stronger and also has more brittle characteristics. Ice creeps at low rates of loading, and it fails in a brittle manner at high loading rates. The complex behavior of ice depends on the temperature and the indentation speed. In engineering applications, one has to contend with both types of ice behavior.

- (1) General. The effective pressure depends on the mode of ice crushing, which in turn depends on the rate of indentation, or the relative speed of an ice feature with respect to a structure.
- (a) At low indentation rates, the ice deforms in creep, resulting in full contact and uniform pressure at the interface. During an interaction involving creep deformation of ice against a narrow structure, the force between an ice sheet and a structure increases gradually, attains a peak value, and then gradually reduces to a steady-state value at 50–60% of the peak force without any structural vibration (Sodhi 1991).
- (b) At high rates of ice indentation against both narrow and wide structures, ice crushes continuously in a brittle manner, resulting in non-simultaneous, partial contact and non-uniform pressure over the nominal contact area. In this mode of crushing, the variations of force over small areas of a wide structure are large, but the summation of these forces across the whole width of a structure averages out the variations in local forces, resulting in smaller variations in the total ice force across the whole structure (Kry 1978, Sodhi 1998, Sodhi 2001). During continuous brittle crushing, the structure does not respond to rapid variations of the interaction forces, and its vibrations are not so severe because it deflects to a steady-state value in response to the average force.
- (c) At intermediate speeds, the interaction between structural deformation and an advancing ice sheet produces alternating ductile and brittle crushing, resulting in ice force records taking a saw-tooth form. During each cycle of intermittent crushing, the advancing ice sheet deflects the structure while undergoing ductile deformation with increasing interaction force. When the ice sheet fails at a certain force level, the stored potential energy in the structure is released to move the structure back to its original position, resulting in high relative speeds and brittle crushing. When the transient oscillations decay, the cycle repeats, causing the structure to indent into the ice at varying speeds and to undergo either transient or steady-state vibrations (Jefferies and Wright 1988; Sodhi 1991, 1995, 2001).
- (d) Figure 6-3 shows a map of ice crushing failure during interactions with rigid and compliant structures. During an ice interaction with a rigid structure, the ice fails in the ductile and brittle modes, and there is a sharp transition at an indentation speed that has been found to be close to 3 mm s<sup>-1</sup> (0.01 ft s<sup>-1</sup>) (Sodhi et al. 1998, Masterson et al. 1999). During an ice interaction with a compliant structure, the failure modes are ductile and brittle at low and high rates of indentation, respectively. At intermediate indentation speeds, the ice fails in the alternating ductile—brittle mode as a result of variable indentation rates into the edge of an ice sheet. Because of this, there are two transition speeds, ductile to intermittent and intermittent to continuous brittle crushing, during interactions with compliant structures. A video display of the interfacial pressure during three modes of ice crushing (ductile deformation, alternating ductile-brittle and continuous brittle crushing) can be seen at the following URL:

  www.crrel.usace.army.mil/permanent/ice crushing.

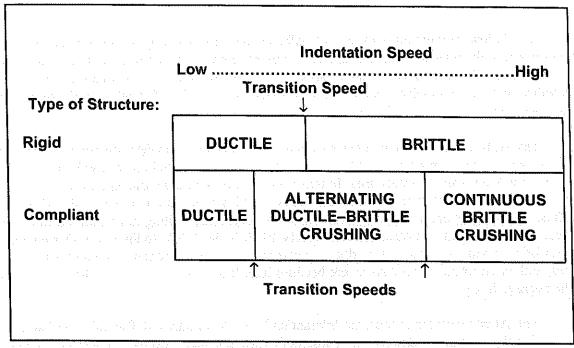


Figure 6-3. Failure map of ice crushing with respect to indentation speed and structural compliance.

(2) Ductile Deformation of Ice. Results of small-scale indentation tests on freshwater columnar ice (Michel and Toussaint 1977) indicate that the effective pressure for ductile (creep) deformation of ice at strain rates between  $10^{-8}$  s<sup>-1</sup> and  $5\times10^{-4}$  s<sup>-1</sup> is

$$p_{e} = Cmk\sigma_{0}(\dot{\varepsilon}/\dot{\varepsilon}_{0})^{0.32}, \tag{6-11}$$

where

C = indentation factor (=2.97)

m = shape factor (=1 for flat indentors)

k = contact factor (=1 for the first peak force, and =0.6 the for steady-state pressure after the first peak force)

 $\sigma_0$  = uniaxial compressive strength of columnar ice at a temperature of  $-10^{\circ}$ C (14°F) and at a strain rate of  $\dot{\varepsilon}_0$ =5×10<sup>-4</sup> s<sup>-1</sup> (=7 MPa or 1015 psi)

 $\dot{\varepsilon} = v/(4D)$  = empirically defined strain rate

 $v_{\rm eff} = in indentation rate$  which cooling there is a substitution of the cooling rate of the state of

D = indenter width.

This relation is similar to the strain rate dependence of uniaxial compressive strength for freshwater columnar ice.

(a) Figure 6-4 shows this comparison with good agreement between the plots of uniaxial compressive strength versus strain rate and the plots of effective indentation pressure divided

by 2.97 versus the empirical strain rate  $(\nu/4D)$ . This expression may also be used for creep indentation of sea ice at any temperature by using the compressive strength of sea ice (Equations 6-2 to 6-4). While Figure 6-4 shows good correlation between effective indentation pressure and the uniaxial compressive strength of columnar ice at the appropriate strain rates, there is no confirmation whether the indentation factor and the empirical definition of strain rate  $(\nu/4D)$  will remain applicable for very large aspect ratios (D/h). Such confirmation is perhaps an impossibility because of creep buckling of floating ice sheets against wide structures at a lower effective pressure than that required for in-plane creep indentation.

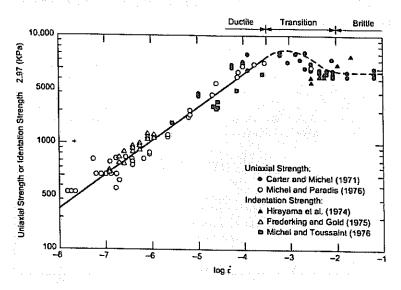


Figure 6-4. Plots of uniaxial compressive strength versus strain rate and the plots of effective indentation pressure divided by 2.97 versus the empirical strain rate (v/4D). (1 kPa × 0.145 = 1 psi.)

- (b) Other methodologies to estimate the ice force ascribable to creep deformation of ice are the plastic limit analysis (Croasdale et al. 1977), and the reference stress method (Ponter et al. 1983, Sanderson 1988, also given in API 1995).
- (3) Brittle Crushing. For edge indentation into floating ice sheets, the main characteristics of brittle crushing are the line-like contact in the middle third of the ice sheet thickness, the non-simultaneous contact in different parts of the contact line, and the non-uniform pressure in the contact area (Joensuu and Riska 1989, Sodhi et al. 1998, Sodhi 2001). This is caused by fracturing of ice at a high rate of loading, resulting in flaking failure of ice. Line-like contacts in the form of an "X" have also been observed during medium-scale indentation tests, in which spherical indentors were pushed into walls of ice at speeds greater than 3 mm s<sup>-1</sup> (0.01 ft s<sup>-1</sup>) (Frederking et al. 1990; Gagnon 1998). The results of full-scale measurements of ice forces, and medium- and small-scale tests indicate that the effective pressure for brittle crushing and for high aspect ratio (D/h) is in the range of 1 and 3 MPa (145 to 435 psi), which is less by a factor of three to four in comparison to the maximum pressure that develops at the high end of speed range for ductile deformation of ice over small areas or small aspect ratios (D/h). The reason for the reduction in effective pressure can be attributed to the actual contact area during brittle

crushing being much smaller than full contact during ductile deformation of ice. The following is an expression to estimate ice force F on a structure of width D for continuous brittle crushing of ice of thickness h at high indentation rates:

 $F=A_{r}pDh$ 

where p is the effective pressure (1.5 to 2 MPa, or 217.5 to 290 psi) for brittle crushing of ice, and  $A_r = (5h/D+1)^{0.5}$  is an empirical factor to account for the aspect ratio effect of high effective pressure over small aspect ratios.

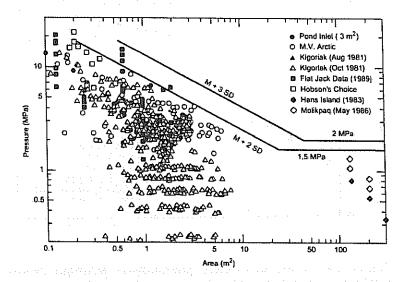


Figure 6-5. plots of effective pressures measured during small- and medium-scale tests, ship ramming, and large-scale field monitoring of ice forces versus nominal contact area. (1 MPa × 145 = 1 psi.)

(4) Empirical Approach. Figure 6-5 shows plots of effective pressures measured during small- and medium-scale tests, ship ramming, and large-scale field monitoring of ice forces versus nominal contact area (Masterson and Frederking 1993). Others have also compiled the socalled pressure-area plots(Iyer 1988, Metge et al. 1988, Sanderson 1988). In plotting these data, no regard is given to the speed of indentation into the ice. There is a large scatter in the data on effective pressures for contact areas less than 5 m<sup>2</sup>, and this can be attributed to variations in indentation speed. The results of small-scale tests show that there is a decrease in effective pressure with increasing indentation speed, even when the contact area is kept constant (Sodhi 1991, 2001). Lack of scatter in the data for effective pressure for areas greater than 100 meters<sup>2</sup> (1076 feet2) can be attributed to brittle crushing having been active at high indentation speeds and the creep buckling of floating ice sheet against wide structures preventing the development of high indentation pressure at low ice speeds (Blanchet 1998). The effective pressure, measured during crushing of first-year ice against the 100-meter-wide (328-foot-wide) Molikpaq structure at ice speeds greater than 100 mm s<sup>-1</sup> (0.328 ft s<sup>-1</sup>), was in the range of 1 to 2.5 MPa (145 to 363 psi) (Wright et al. 1986, Wright and Timco 1994). Effective pressure in the range of 1-3 MPa (145 to 435 psi) have also been measured on indentors during small-scale tests in the same velocity range (Sodhi 1992, 2001). These two observations indicate that, when continuous brittle crushing is active, the effective pressure is independent of the nominal contact area. Because high contact pressure can act over a small area resulting from ductile deformation of ice, the trend in the upper bound of effective pressure versus contact area (Figure 6-5) shows a decrease in effective pressure with increasing contact area. Though this trend is known as a scale effect in the literature, the real reason for the decrease in effective pressure with increasing contact area is the possibility of high pressure developing over a small area because of ductile deformation and crushing of the ice in the brittle mode over a large contact area or high aspect ratio (D/h).

(a) Two lines in Figure 6-5, labeled as M+2SD and M+3SD, signify trend lines of mean (M) plus two and three standard deviations (SD) of the data, respectively. These are given by:

M+2SD: 
$$p(MPa) = 8.1 A^{-0.5}$$
 for  $0.1 m^2 < A < 29 m^2$ , and  $p = 1.5 MPa$  for  $A > 29 m^2$ , (6-12)  
(M+2SD:  $p(psi) = 1175 A^{-0.5}$  for  $1 \text{ ft}^2 < A < 312 \text{ ft}^2$ , and  $p = 217.5 \text{ psi for } A > 312 \text{ ft}^2$ )

M+3SD: 
$$p(MPa) = 13 A^{-0.5}$$
 for 0.1 m<sup>2</sup> <  $A$  < 42 m<sup>2</sup>, and  $p$  = 2 MPa for  $A$  > 42 m<sup>2</sup>. (6-13)  
(M+3SD:  $p(psi) = 1885 A^{-0.5}$  for 1 ft<sup>2</sup> <  $A$  < 452 ft<sup>2</sup>, and  $p$  = 290 psi for  $A$  > 452 ft<sup>2</sup>)

(b) Both of these equations have been recommended in API (1995). The recommended pressures in the Canadian codes for offshore structures (CSA 1992) are similar. A designer needs to choose the design pressure either higher or lower than the values obtained from Equations 6-12 or 6-13, depending on location of the structure. For example, the Molikpaq structure and its structural components have been designed using pressures given by Equation 6-12 with no visible local damage to the structure, whereas the ice pressure measured on subarctic regions such as Cook Inlet have been less than the pressures given by Equation 6-12.

#### c. Bending Failure.

- (1) Sloping Structure. When a floating ice sheet moves against an upward or downward sloping structure, the sheet is pushed either up or down, and breaks by bending into blocks. As the ice sheet continues to be pushed up or down, the broken slabs are further broken into slabs that are typically 4 to 8 times the ice thickness. The force on the structure is limited by the amount required to fail the ice sheet in bending and to overcome the weight and frictional forces of the broken ice blocks. If the structure is narrow, the broken pieces of ice may be able to go around the structure. For wide structures, the broken pieces of ice either ride up to clear over the top of the structure or forms an ice rubble mound. Procedures to estimate ice forces on sloping and conical structures are given in textbooks (Ashton 1986, Cammaert and Muggerdige 1988, Sanderson 1988).
- (a) API (1995) gives equations for determining the ice forces on a sloping structure, where the broken ice pieces are assumed to ride up the sloping surface and fall off into the water on the other side. Figure 6-6 shows forces during an interaction of a floating ice sheet of thickness h being pushed against a wide sloping surface at an angle  $\alpha$  with the horizontal. If the ice blocks are lifted up a height z along the sloping surface, the weight of the broken ice sheet on the sloping surface has a magnitude per unit width of  $W = \rho_1 ghz/\sin \alpha$ , where  $\rho_1 g$  is the specific weight of ice, and h is the ice thickness. The normal force per unit width on the surface is  $N = W \cos \alpha$ , and the tangential force along the surface is  $\mu N$ , where  $\mu$  is the coefficient of friction between the

surface and the ice. As shown in Figure 6-6, the force T acting between the broken ice on the sloping surface and the top of the floating ice sheet has a magnitude per unit width of  $T=W(\sin\alpha + \mu\cos\alpha)$ .

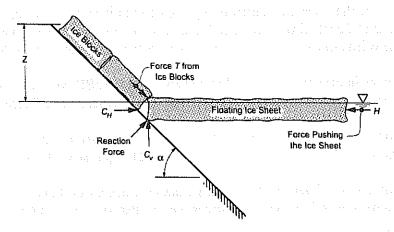


Figure 6-6. forces during an interaction of a floating ice sheet of thickness h being pushed against a wide sloping surface at an angle  $\alpha$  with the horizontal.

- (b) The reaction force (Figure 6-6) acting on the contact between the sloping structure and the advancing ice sheet has components  $C_H$  and  $C_V$  in the horizontal and vertical directions, respectively. The total horizontal force per unit width is given by  $C_H + T\cos\alpha$ . As the structure pushes the advancing ice sheet up, the vertical force acting at the end of the ice sheet has a magnitude per unit width equal to  $C_V T\sin\alpha$ .
- (c) Under the assumption that there is no moment acting on the floating ice sheet, the vertical force component  $C_V$  per unit width required to break the floating ice sheet and push it up is given by:

$$C_{\rm V} = \frac{\sigma h^2 + 6\ell e^{-\frac{\pi}{4}} T \sin \alpha + Th \cos \alpha}{6\ell e^{-\frac{\pi}{4}} - h \tan(\alpha + \arctan \mu)},$$
(6-14)

where

 $\sigma$  = flexural strength of ice sheet

h = ice thickness

 $\alpha$  = angle between the sloping surface and the horizontal

 $\ell = [Eh^3/\{12(1-v^2)\rho_w g\}]^{1/4}$  (the characteristic length of floating ice sheet)

E =effective elastic modulus of ice

 $\nu$  = Poisson's ratio of ice

 $\rho_{wg}$  = specific weight of ice.

For typical bending rates, the effective elastic modulus of freshwater ice is in the range of 1-3 GPa  $(1.45 \times 10^5 \text{ to } 4.35 \times 10^5 \text{ psi})$ , and Poisson's ratio is about 1/3. The range of the coefficient of friction between ice and a structure is between 0.1 for freshly coated surfaces and 0.5 for

rusty, rough surfaces. There are, at present, no guidelines available for the coefficient of friction on rough surfaces or on riprap protected surfaces.

(d) The horizontal force C<sub>H</sub> per unit width from the structure on the ice sheet is

$$C_{\rm H} = C_{\rm V} \tan(\alpha + \arctan \mu) \tag{6-15}$$

The total force H per unit width generated during the interaction to break the ice sheet at a distance away from the contact zone and to push the broken ice block along the sloping surface is given by:

$$H = C_H + T\cos\alpha$$
.

Besides other parameters, the total horizontal force on a sloping structure of width D is composed of an icebreaking force proportional to  $\sigma h^2 D/\ell$  and an ice ride up force proportional to  $\rho_{ighz}D$ . For narrow structures, the ice breaking component is greater than ice ride up component, whereas the opposite is true for wide structures. Using the above two-dimensional formulation to estimate ice force on a sloping structure gives an order of magnitude of forces, which can be compared to estimates of ice force from other failure modes.

- (e) The above formulation does not take into account non-simultaneous and incomplete contact between the edge of an ice sheet and a wide structure, as is observed in small-scale tests (Izumiyama et al. 1992, Kovacs and Sodhi 1988). If broken ice pieces accumulate above and below the water surface near the ice-structure contact, the estimate of ice forces should take into account the effect of forces acting on the top and bottom surfaces of the advancing ice sheet (Määttänen and Hoikannen 1990, Croasdale and Cammaert 1993).
- (2) Conical Structure. There are several methodologies to estimate the ice forces on a conical structure: the elasticity method (e.g., Nevel 1992), the plasticity method (e.g., Ralston 1977, 1980; Izumiyama et al. 1992; Lau 2001), and the results of model tests. Some of these methodologies are briefly presented in API (1995). Several reviews of the literature on the ice forces on conical structures have been presented (Croasdale 1980, Wessels and Kato 1988), and theoretical and experimental results have also been compared (Chao 1992, Wang et al. 1993). Equation 6-15 has also been used to estimate the ice forces on a conical structure by the assuming the width of the conical structure to be  $\pi(R+x)$ , R being the waterline radius of the conical structure, and  $x = \sqrt{2(\pi/4)} \ell$ , where  $\ell$  is the characteristic length of floating ice sheets (Croasdale 1980).
- (3) Indentation at High Speeds. The interaction of a slowing moving ice sheet with a narrow sloping structure usually results in bending failure, resulting in large ice blocks in comparison to their ice thickness. However, when the speed of the moving ice sheet is large, the failure mode changes to shearing or crushing, resulting in small broken ice pieces. Both modes of ice failure against sloping structures have been observed in the field (Neill 1976, Lipsett and Gerard 1980) as well as during small-scale tests in the laboratory (Haynes et al. 1983, Sodhi 1987). This observation implies that ice may fail in crushing, instead of the expected bending, while moving towards a sloping structure at high speeds.

- c. Buckling Failure. The ice force on a vertical structure may be limited by the buckling of a floating ice sheet. It depends on the properties of the ice sheet, the width of structure, and the boundary conditions at the ice-structure interface. The boundary conditions along the icestructure contact line may be defined as rigid, if the ice sheet is frozen to the structure, preventing vertical displacement and rotation. They may be defined as hinged if the ice sheet is allowed to rotate, but is prevented from vertical displacement. And they may be described as free if the ice sheet can displace vertically without encountering any resistance and rotate freely. Discussions of the elastic buckling of beams and plates are given in textbooks and review papers (Hetenyi 1946, Michel 1978, Sodhi and Nevel 1980, Ashton 1986, Cammaert and Muggeridge 1988)
- (1) Elastic Buckling. For a beam of floating ice sheet, the elastic buckling force (Hetenyi 1946) is given by

$$F_{\rm b} = \alpha \rho_{\rm w} g B L_{\rm b}^2 \tag{6-16}$$

where

 $\alpha$  = factor that depends on the ratio of beam length to the characteristic length and the boundary conditions at the ends of the beam.

 $\rho_{wg}$  = specific weight of water,  $v \in B$  . = obeam width, when it conserves where the constraint we constraint where the constraint  $v \in A$ 

 $L_b$  = characteristic length of the floating ice beam and is equal to  $(Eh^3/(12\rho_{\rm w}g))^{1/4}$ 

h = ice thickness

E = modulus of elasticity of ice.

For beam lengths much longer than the characteristic length of a floating ice beam, the factor  $\alpha$ is either equal to 1 if one or both ends of the beam are free, or equal to 2 if one or both ends of the beam are either rigid or hinged (Sodhi and Nevel 1980).

(a) The elastic buckling forces of wedge-shaped, semi-infinite ice sheets is given by

$$F_{\rm p} = [C + D/(R/L)] \rho_{\rm w} g B L^2 \tag{6-17}$$

where

coefficients given in Table 6-1 for three boundary conditions and five wedge C and D angles in the range of 2 to 180°

R radius of the structure at the contact line

 $\rho_{\rm w}g$  = specific weight of water

 $\boldsymbol{B}$ = structure width

= characteristic length of the floating ice sheet and is equal to  $[Eh^3/\{12(1-v^2)\rho_w g\}]^{1/4}$ 

= ice thickness

= modulus of elasticity of ice.

A recommended value of the characteristic length of freshwater ice  $L = 16h^{3/4}$ , and that of sea ice  $L = 13h^{3/4}$ , where L and h are in meters (Gold 1971).

Table 6-1
Coefficients for the estimation of buckling force

• • •	Boundary conditions at the ice-structure contact line							
	Free		Hinged		Rigid			
Angle	С	D	С	D	С	D		
2°	0.96	0.80	2.11	2.76	2.57	4.47		
30°	1.00	0.82	2.20	3.11	2.55	4.70		
90°	0.95	1.01	2.04	3.78	2.35	5.34		
150°	0.84	1.36	1.81	4.30	2.08	5.83		
180°	0.81	1.66	0.75	4.67	2.04	6.05		

- (b) Experimental studies on the buckling of floating ice sheet indicate that the non-dimensional buckling forces fall between those for free and hinged boundary conditions at the contact line (Sodhi et al. 1983). For tests in which the boundary condition was simulated as hinged, the experimental and theoretical results are close to each other (Sodhi and Adley 1984).
- (2) Creep Buckling. For creep buckling, a floating ice sheet is considered to become unstable when the vertical deflections of the ice sheet suddenly increase after a long period of slow in-plane deformation. In a series of finite element analyses of an ice sheet being pushed slowly against a 152-meter-diameter (500-foot-diameter) structure, the results show that the ice sheet deforms slowly in the vertical direction until a critical time when large deformations suddenly occur in the vicinity of the structure (Luk 1990). The critical time  $t_{\rm cr}$  at which large deformations take place can be estimated as

$$t_{\rm cr} = 0.36 \, D/\nu, \qquad \text{where the constraints are the experimental problem (6-18)}$$

where D is the width of the structure, and v is the ice velocity. For an elastic modulus of 4.83 GPa  $(7.0 \times 10^5 \text{ psi})$ , a Poisson's ratio of 0.3, and a creep exponent of 3, the results of finite element analysis show that the following relationship between the effective pressure p and the critical time  $t_{\rm cr}$  is

$$(p/\text{MPa}) = (7.07 \text{ days/}t_{cr})^{0.336}.$$
 (6-19)

d. Floe Splitting. When a floating ice floe impacts and crushes against a structure, the floe may split up after some amount of crushing. The ice-structure interaction results in deceleration of the ice floe, which creates a distributed inertia body force per unit volume over the entire floe. Depending on the geometry of the structure, the ice crushing in the contact zone produces a longitudinal force as well as a pair of self-equilibrating transverse forces, which are a fraction  $\beta$  of

the longitudinal force. Under the assumption that the linear elastic fracture mechanics is applicable, the critical force  $F_{sp}$  to split a square floe of length  $\lambda$  and of thickness h is given by

$$F_{\rm sp} = 3.3 \ h \, K_{Ic} \, \lambda^{1/2},$$
 (6-20)

where  $K_{1c}$  is the fracture toughness of ice (Bhat 1988). Similar ice splitting forces can also be estimated for different floe width-to-length ratios and different values of  $\beta$  (Dempsey et al. 1993). The fracture toughness of freshwater and saline ice through small-scale and large-scale measurements is in the range of 50 to 250 kPa m<sup>1/2</sup> (0.007 to 0.036 ksi in<sup>0.5</sup>) (Dempsey et al. 1999a,b). Small-scale tests were conducted with freshwater ice floes of different thicknesses and widths, and those experimental results were found to be close to the theoretical forces obtained from finite element analysis using linear elastic fracture mechanics (Sodhi and Chin 1995).

e. Structures Going Through Broken Ice Cover. When a structure goes through a broken up ice cover having a depth  $t_k$  and a height  $t_s$  from the water surface, the force F per unit width (Mellor 1980) is given by

$$F = \frac{1}{2} \frac{1 + \sin \phi}{1 - \sin \phi} (1 - n) \left[ \rho_{i} g t_{s}^{2} + (\rho_{w} - \rho_{i}) g t_{k}^{2} \right] + 2c \sqrt{\frac{1 + \sin \phi}{1 - \sin \phi}} (t_{s} + t_{k}), \qquad (6-21)$$

where

 $\phi$  = angle of internal friction

c = cohesive strength of rubble ice

n = porosity of rubble ice

 $\rho_i g$  = specific weight of ice

 $\rho_{\rm w}g$  = specific weight of water.

### 6-5. Forces Limited by the Momentum of an Ice Feature

- a. When an isolated ice feature impacts a structure, it may come to rest, deflect, or rebound from the structure. The interaction forces during an impact may be computed from the equations of momentum and energy of the two colliding bodies (Goldsmith 1960). Because the rate of indentation during an ice impact is usually high, brittle crushing of ice is expected to take place in the contact area, which depends on the local geometry of the ice feature and the structure.
- b. For a head-on collision in which a moving ice feature comes to a stop against a structure, the initial kinetic energy is dissipated to crush a certain volume V of ice at an effective pressure of  $p_e$  to give

$$Mv^2/2 = p_eV,$$
 (6-22)

where M is the mass of the ice feature including the added mass of water, and v is its velocity. The depth d and the area A of ice crushing can be calculated from the estimated volume V of the crushed ice and the local geometry of the ice feature and the structure. The interaction force can now be estimated as  $F = p_e A$ .

c. An eccentric impact (Figure 6-7) will rotate the ice floe, and the ice feature will retain a portion of the initial kinetic energy after the impact. By equating the initial kinetic energy to the sum of remaining kinetic energy and dissipation of energy during ice crushing, the following relationship can be shown under the assumption of brittle crushing (Nevel 1986):

$$\frac{Mv^{2}}{2} = \frac{Mv^{2}}{2} \left[ \frac{(Y/R)^{2}}{1 + (R_{g}/R)} \right] + (1 + \mu \tan \beta) p_{e}V$$
 (6-23)

where

M =mass of ice feature, including the added mass of water

v = ice feature velocity

Y = eccentricity of the center of gravity from the point of impact

R = distance of the center of gravity from the point of impact

R<sub>g</sub> = radius of gyration of the ice feature about the vertical axis through its center of gravity

 $\mu$  = ratio of tangential force to normal force in the contact area

 $\beta$  = angle of local contact geometry ( $\beta$ = 0 for head-on impact

 $p_e$  = effective pressure to crush ice

V = volume of ice crushed.

For a particular impact situation, the volume of ice crushed during an impact is estimated, and then the depth d and the area A of ice crushed are estimated from the local contact geometry. Lastly, the interaction force is estimated as  $F = p_0 A$ .

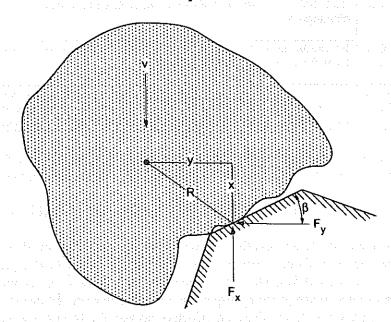


Figure 6-7. An eccentric impact that will rotate the ice floe, allowing the ice feature to retain a portion of the initial kinetic energy after the impact.

#### 6-6. Canadian and American Codes

a. To estimate dynamic ice force F on bridge piers resulting from moving ice, CSA (2000) and AASHTO (1994) codes specify the following:

$$F =$$
lesser of the  $F_c$  or  $F_b$  for  $D/h < 6$ 

and

$$F = F_c \text{ for } D/h > 6$$

where

 $F_c = C_a pDh$  (horizontal force in when ice floes fail by crushing over full width of the pier)

 $F_b = C_n ph^2$  (horizontal force in when ice floes fail in bending against a sloping pier)

D = the pier width

h = the ice thickness

 $C_a = (5h/D+1)^{0.5}$  (to account for the aspect ratio effect found in small-scale identation tests)

 $C_n = 0.5 \tan(\alpha + 15^\circ)$ 

 $\alpha$  = slope of the pier from the downstream horizontal (<75°)

p = effective ice crushing pressure for which following values have been recommended.

0.7 MPa (101.5 psi)	Ice breaks up at melting temperature and is somewhat disintegrated.
1.1 MPa (159.5 psi)	Ice breaks up or moves at melting temperature, but the ice moves in large floes and is internally sound.
1.5 MPa (217.5 melting point. Even higher pressures are recommended for ice temperatures 2 or 3°C (35.6 or 37.4°F) below melting temperatures	

- b. Further, these codes recommend reducing the dynamic ice force F by 50% of the values derived above for piers in small streams where it is unlikely to encounter large-size floes.
  - c. For oblique impacts, readers should see the CSA (2000) and AASHTO (1994) codes.
- d. It should be mentioned here that the above-recommended values of effective crushing pressure have been obtained from measurements of ice forces on two bridge piers in Alberta, Canada (Lipsett and Gerard 1980). These recommended values for effective pressure for wide structures are the same as those given by Equation 6-12. For narrow structures, the factor  $C_a$  accounting for the aspect ratio effect raises the effective pressure to higher values, similar to given by Equation 6-12 for nominal contact area less than 29 meters<sup>2</sup> (312 feet<sup>2</sup>).

#### 6-7. Vertical Ice Forces

- a. Marine structures that become frozen into an ice sheet are subjected to vertical ice forces as the ice sheet responds to changes in water level. Typically, the uplifting load resulting from changes in water level governs the design of light-duty, pile-founded docks common in marinas. Thus, reducing the vertical ice loads by active (bubblers or water jet) or passive (pile jacket or low-adhesive coatings) means will directly lead to lower costs for such structures. Theoretical estimates of vertical ice loads in the literature (e.g., Ashton 1986) depend on the assumed mode of ice failure, which is often difficult to ascertain for a particular situation. For instance, Muschell and Lawrence (1980) conducted pull-out tests after freezing a conventional capped pipe pile (filled with air) and another similar pile filled with vermiculite insulation, and found a 30% reduction in vertical ice force. In a conventional capped pipe pile, a thermal convection cell develops, resulting in freezing of thicker ice adjacent to the pile. The insulation-filled pipe interrupted this heat transfer, reducing the localized ice thickening and thus yielding a 30% reduction in pull-out force. An epoxy coating was applied to both air-filled and vermiculite-filled piles, which resulted in a net load reduction of 35 and 70%, respectively. Frederking and Karri (1983) used polyethylene and PVC piles with an average reduction in failure stress of 70% compared with similar size wooden piles used in a previous study (Frederking 1979). It was observed that the failure occurred at the ice/ice interface in the case of conventional piles, whereas the relative movement in the case of epoxy coated or plastic piles was at the pile/ice interface.
- b. While reviewing the results of pull out tests in the literature, Zabilansky (1998) noted that the researchers used the following three test techniques to conduct a test.
- (1) Socket. A hole was drilled through part of an ice sheet, a pile was placed in the counter-bored hole, and the void between the ice and the pile was filled with water. This test technique measured the anchoring capacity of the pile, which is not representative of the marina application.
- (2) Confined. Most of these tests were conducted using a testing machine to extract a pile out a block of ice. Muschell and Lawrence (1980) conducted a series of tests on a lake. They froze a pile into an ice sheet by placing it in a hole cut in the ice sheet and allowing the annulus water between the pile and the ice sheet to freeze. The test was conducted about a week later by jacking the pile out while reacting the parent ice sheet. In these tests, the reaction ring or plate was slightly larger than the pile, resulting in shear failure of ice adjacent to the pile.
- (3) Unconfined. A pile was either placed into a tank of water during freezing of an ice sheet or frozen into an ice sheet on a lake. The tests were conducted by pulling out the pile while reacting against either the edge of a tank or a support frame placed on top of the ice sheet. This mode of loading subjected the ice sheet to both bending and shear stresses, and was more representative of the observed failure mode in the field.
- c. Zabilansky (1998) compiled the data from the pull-out tests with wooden piles conducted at CRREL (Zabilansky 1986) and unconfined tests reported in the literature. He plotted the shear

stress (force/circumferential contact area) with respected to the ratio of pile diameter to ice thickness, as shown in Figure 6-8. A line of best fit through the data is given by:

$$\sigma = 300/(d/h)^0. \tag{6-24}$$

The equation for the pull out force is given as:

$$P = \sigma \pi dh = 300\pi h^{1.6} d^{0.4} \tag{6-25}$$

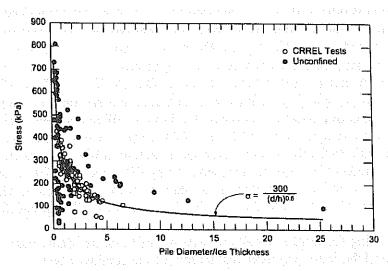


Figure 6-8. Failure shear stress vs. ratio of pile diameter to ice thickness. (1 kPa × 0.145 = 1 psi.)

#### 6-8. Summary

This Chapter estimates ice forces on the basis of ice mechanics for its failure in various modes, as well as on empirical values of effective pressure measured on full-scale structures. For ice crushing, which induces the highest effective pressure on a structure, the effective pressure depends on the indentation speed and the aspect ratio (D/h). Most of the codes take these factors into account for estimating ice forces on structures. When an ice sheet fails in modes other than crushing, the effective pressure is generally less.

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# Appendix 'K': Ice Engineering Guidelines – US Army Corps of Engineers

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#### VI-5-8. Other Forces and Interactions

a. Impact forces. Impact force loading on coastal projects occurs when waves or solid objects collide with typically stationary coastal structure elements. Only solid body impacts are discussed in this section. Impact loads between shifting concrete armor units are discussed in Part VI-5-3c, "Structural integrity of concrete armor units."

Certain coastal structures such as thin-walled flood barriers, sheet-pile bulkheads, mooring facilities, coastal buildings, or other infrastructure may be subject to impact damage by solid objects carried by waves, currents, or hurricane-force winds. During severe storms, high winds may propel small pleasure craft, barges, and floating debris which can cause significant horizontal impact loads on structures. Likewise, floating ice masses can also cause great impact loads. Impact loads are an important consideration in design of vessel moorings and fendering systems.

Designing a structure to resist impact loads during extreme events is difficult because of uncertainty associated with impact speed and duration. In situations where impact damage by large floating objects could cause catastrophic loss, it may be prudent to limit adjacent water depth by constructing sloping rubble-mound protection fronting the structure or by placing submerged breakwaters seaward of the structure to ground large floating masses and eliminate the hazard.

Impact forces are evaluated using impulse-momentum and energy considerations found in textbooks on fundamental dynamics. However, application of these principles to particular impact problems is difficult unless reliable estimates can be made of object mass (including added mass in water), the mass initial and final velocities, duration of impact loading, and distribution of impact force over time. In addition, some evaluation must be made on whether the collision of the floating object with a coastal structure results in purely elastic behavior in which momentum is conserved, purely plastic impact with all the kinetic energy of the impact being absorbed, or some combination of the two.

Fendering systems in ports and harbors are designed to absorb low-velocity impacts by vessels during docking maneuvers and seiching motions. Design of fendering systems is adequately covered by numerous textbooks and design standards. Examples of typical design references in the coastal engineering literature include Quinn (1972) and Costa (1973). The modes of kinetic energy absorption by fendering systems were studied theoretically by Hayashi and Shirai (1963). Otani, Horii, and Ueda (1975) presented field measurements related to absorption of impact kinetic energy of 50 large tankers. They observed that most berthing velocities are generally below 6 cm/s, and that measured impact energy was substantially larger than calculated using the design standards that existed at that time. Kuzmanovic and Sanchez (1992) discussed protective systems for bridge piers and pilings, and they gave procedures for accessing the equivalent static force acting on bridge piers due to vessel impacts.

- b. Ice forces. A description of ice loading and how it may impact various types of coastal structures in the context of site-specific design criteria is given in Part VI-3-5, "Ice." Other general references include Chen and Leidersdorf (1988); Gerwick (1990); and Leidersdorf, Gadd, and Vaudrey (1996). The following section presents methods for calculating ice forces under specific loading conditions.
  - (1) Horizontal ice forces.
  - (a) Solid ice forces.
  - Large horizontal forces can result when solid sheet ice, or large chunks of solid ice that have broken free, come in contact with vertical-front coastal structures. Most ice sheets are large enough that impact forces are limited by ice failure in the weakest mode permitted by the mechanics of interaction as the structure penetrates the ice, i.e., crushing, splitting, shear, or bending. For smaller ice blocks or wide structures, the maximum impact force may be limited by the kinetic energy available at the moment of impact (HQUSACE 1982). Ice impact calculations should be based on impulse-momentum considerations, but such calculations will be difficult because of uncertainty in estimating a value for ice block velocity.
  - Wind and water current drag acting on large floating blocks of ice press the ice blocks against structures creating large pressures at the points of contact. The force due to drag on a block of ice can be calculated for wind and water currents using the following formula (PIANC 1992)

$$F_d = C_{sf} \rho A (u - u_i)^2$$
 (VI-5-309)

where

 $C_{sf}$  = coefficient of skin friction between wind and ice or water and ice (see Table VI-5-89)

 $\rho$  = fluid density (air or water)

Table VI-5-89 Values of Skin Friction Coefficient, C <sub>st</sub> (PIANC 1992)					
	eranieko errenakoa	Smooth Ice	Rough (Pack) Ice		
Wind drag Water drag		0.001 - 0.002 0.002 - 0.004	0.002 - 0.003 0.005 - 0.008		

A =horizontal area of ice sheet

u = fluid velocity (10 m above ice for air or 1 m below ice for water)

 $u_i$  = velocity of ice in the direction of u

- Separate drag calculations should be performed for both wind and water currents with the results treated as vector forces on the ice mass. Because drag force is directly proportional to ice surface area, larger ice sheets will exert greater forces.
- Once an ice sheet has come to rest against a structure,  $u_i$  is zero, and the total drag force can be calculated. Intact ice sheets should be treated as solid bodies with the resultant loads vectorially distributed among the structure/ice contact points using force and moment balance. The total force may be somewhat uniformly distributed along a lineal vertical wall. However, if the ice block comes in contact at only a few discrete points, the contact pressure may be very large. In these cases, the calculated force due to drag may exceed the force necessary for local crushing of the ice, in which case the local crushing strength becomes the limiting force applied to the structure.
- (b) Localized ice crushing forces.
- The limiting ice force on a vertical structure is determined by the crushing failure strength of the ice in compression. A theoretical expression for the horizontal ice crushing force was given in Korzhavin in a 1962 Russian publication (Ashton 1986) as

$$\frac{F_c}{bh_i} = mIkx_{\sigma_c} \qquad (V1-5-310)$$

where

 $F_c$  = horizontal crushing force

b =structure horizontal width or diameter  $h_i =$ thickness of ice sheet

m =plan shape coefficient

I = indentation coefficient

k = contact coefficient

x = strain rate function

 $\sigma_c$  = ice compressive failure strength in crushing

- This formula is usually applied to piles and pier structures rather than long vertical walls. The plan shape coefficient, m, is 1.0 for flat surfaces, 0.9 for circular piles, and  $0.85[sin (\beta/2)]^{1/2}$  for wedge-shaped structures having a wedge angle of  $\beta$ . The indentation coefficient, I, has been found experimentally to be a function of the aspect ratio,  $b/h_i$ , and it is usually presented in graphical form. The contact coefficient, k, is a function of ice velocity and width of structure, and it varies between values of 0.4 to 0.7 for ice velocities between 0.5 and 2.0 m/s. The strain rate coefficient is also a function of ice speed. Ashton (1986) provided further details about the theoretical development of Equation VI-5-310 and its associated coefficients.
- In a Russian publication, Afansev (Ashton 1986) combined the coefficients 1, k, and x of Equation V1-5-310 into a single coefficient, C, giving the formula

$$\frac{F_c}{bh_l} = C m \sigma_c \tag{V1-5-311}$$

• Afansev established the following empirical relationship for C based on model tests using laboratory-grown, saline ice.

$$C = \left(5\frac{h_i}{b} + I\right)^{1/2} \qquad \text{for } 1 < \frac{b}{h_i}$$

$$C = 4.17 - 1.72 \left(\frac{b}{h_i}\right) \qquad \text{for } 0.1 < \frac{b}{h_i} < 1$$
(VI-5-312)

- The lower formula in Equation VI-5-312 is a linear interpolation as recommended in Ashton (1986).
- In Equation VI-5-311 values of the shape coefficient are the same as given for the Korzhavin formula (Equation VI-5-310).
- The Canadian Standards Association Bridge Code (Canadian Standards Association 1978) recommended an even more simplified version of Equation VI-5-310 given by

$$\frac{F_c}{bh_i} = \sigma_c \tag{VI-5-313}$$

using the range of values for sheet ice compressive crushing strength shown in Table VI-5-90. Equation VI-5-313 and the crushing strength values of Table VI-5-90 were also adopted by the American Association of State Highway and Transportation Officials.

Use of Equation VI-5-313 implies that the product  $C \cdot m = 1$  in Equation VI-5-311, which corresponds to large values of  $b/h_i$ . This is a realistic assumption for large bridge piles and piers, but ice crushing forces on smaller diameter piles should be calculating using the appropriate strength values from Table VI-5-90 in Equation VI-5-311.

Table VI-5-90 Values of Effective Ice Crushing Strength, σ <sub>c</sub>				
Ice Crushing Stress	Environmental Situation			
0.7 MPa (100 psi)	Ice breakup occurs at melting temperatures and the ice moves in small pieces that are essentially disintegrated.			
1.4 MPa (200 psi)	Ice breakup occurs at melting temperatures, but the ice moves in large pieces that are generally sound.			
2.1 MPa (300 psi)	Ice breakup consists of an initial movement of the entire ice sheet or large sheets of sound ice impact piers.			
2.8 MPa (400 psi)	ice breakup occurs with an ice temperature significantly below the melting point and ice movement consists of large sheets.			

- (c) Thermal ice forces. Equations are available for predicting ice temperature based on an energy balance between the atmosphere and the ice sheet. However, the required parameters (air temperature, air vapor pressure, wind, and cloud cover) needed to calculate thermal expansion are difficult to estimate. Thermal strain is equal to the ice thermal expansion coefficient times the change in ice temperature. For restrained or partially restrained ice sheets a nonlinear, time dependent stress-strain law is used to predict thermal stresses (HQUSACE 1982). Because of stress relaxation due to creep, the rate of thermal change is an important factor; and even a thin snow cover can drastically reduce thermal stresses in ice sheets.
  - A design rule-of-thumb for thermal expansion loads per unit horizontal length on dams and other rigid structures is 145 - 220 kN/m (10,000 - 15,000 lbs/ft) (HQUSACE 1982). Movable structures should allow for 73 kN/m (5,000 lbs/ft). These values are based on field measurements.
  - Thermal expansion of water frozen between elements of a coastal structure can result in dislocation of individual elements or cracking of armor units making the protection vulnerable to wave attack.
  - (2) Ice forces on slopes.
  - (a) Ride-up of ice on slopes.
  - When horizontally moving ice encounters a sloping structure, a component of the horizontal force pushes the ice up the slope. This action induces a bending failure in the ice sheet at loads less than required for ice crushing failure. Ashton (1986) showed the derivation of a simple two-dimensional theory for calculating the horizontal force exerted by ice on a sloping structure as illustrated in Figure VI-5-144. (Ashton also included discussion and analysis of the more complex case of ice ride-up on three-dimensional structures).

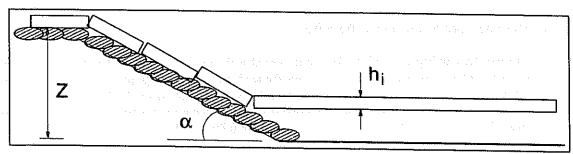


Figure VI-5-144. Ice riding up on structure slope

• For the two-dimensional case the horizontal force per unit width of structure was given by the expression

$$\frac{F_h}{b} = C_1 \sigma_f \left( \frac{\rho_w g \, h_i^5}{E} \right)^{1/4} + C_2 Z \, \rho_i g \, h_i$$
 (VI-5-314)

with

$$C_1 = 0.68 \left[ \frac{\sin \alpha + \mu \cos \alpha}{\cos \alpha - \mu \sin \alpha} \right]$$
 (VI-5-315)

and

 $F_h$  = total horizontal force

$$C_2 = \left[ \frac{(\sin \alpha + \mu \cos \alpha)^2}{\cos \alpha - \mu \sin \alpha} + \frac{\sin \alpha + \mu \cos \alpha}{\tan \alpha} \right]$$
 (V1-5-316)

b =horizontal width of structure contact zone

 $h_i$  = ice sheet thickness

 $\sigma_f$  = flexural strength of ice (0.5 = 1.5 MPa)

 $\rho_w$  = water density

 $\rho_i = \text{ice density } (915 = 920 \text{ kg/m}^3)$ 

E = modulus of elasticity of ice (1,000 = 6,000 MPa)

Z = maximum vertical ice ride-up distance

g = gravitational acceleration

 $\alpha$  = structure slope angle relative to horizontal

 $\mu$  = structure slope friction factor (0.1 = 0.5)

- The first term in Equation VI-5-314 is interpreted as the force necessary to break the ice in bending, and the second term is the force that pushes the ice blocks up the sloping structure. The modulus of elasticity varies from 1,000 MPa for very salty water up to about 6,000 MPa for fresh water (Machemehl 1990). Ashton (1986) warned that this simple two-dimensional theory will be inadequate for narrow structures because the zone of ice failure will be wider than the structure.
- Low values of friction factor ( $\mu = 0.1$ ) are associated with smooth slopes such as concrete or carefully layed block protection, whereas high values ( $\mu = 0.5$ ) are applicable for randomly-placed stone armor, riprap, or filled geotextile bags. For slopes steeper than 1:1, the horizontal ice force increases

rapidly for the higher friction factors, and there is a risk of the dominant failure mode being crushing or buckling rather than bending. Milder slopes with smooth surfaces are much more effective in reducing horizontal ice forces. Croasdale, Allyn, and Roggensack (1988) discussed several additional aspects related to ice ride-up on sloping structures.

 Quick "rough" estimates of horizontal forces on sloping structures can be made using a variation of Equation VI-5-313 as proposed in Ashton (1986), i.e.,

$$\frac{F_h}{b} = K_h h_i \sigma_c \tag{VI-5-317}$$

where  $K_h$  is approximated from a curve given in Ashton (1986) by the formula

$$K_h = 1 - 0.654 f^{0.38}$$
 (VI-5-318)

with

$$f = \frac{1 - \mu \tan \alpha}{\mu + \tan \alpha} \tag{V1-5-319}$$

and  $\sigma_c$  is the ice compressive strength as given in Table VI-5-90. As slope angle increases,  $K_h$  approaches a value of unity which represents failure by crushing. For decreasing slope angles,  $K_h$  decreases because of the increasing tendency of the ice to fail in bending. Values of  $K_h$  less than 0.2 should never to used in Equation VI-5-317.

- (b) Adfreeze loads. When ice that is in contact with a coastal structure is stationary for a sufficient time, the ice will freeze to the structure or its elements. Adfreeze loads result if the ice then moves either horizontally by dynamic ice thrust or vertically due to changing water level. This is more of a problem in lakes with slowly varying water levels than in tidal waters.
  - Little guidance is available on adfreeze stresses with adhesion strength varying between 140 kPa to 1050 kPa for freshwater ice (PIANC 1992). Adfreeze may dislodge individual armor stones on rubble-mound slopes creating a weakness in the armor layer. This can be prevented by using oversized stones or interlocking armor on the slope. A survey of riprap structures at Canadian hydropower reserviors concluded that plucking of individual stones frozen to ice could be largely prevented by sizing the riprap median diameter ( $d_{50}$ ) greater than the expected maximum winter ice thickness (Wuebben 1995).
- (3) Vertical ice forces. Ice frozen to coastal structures can create vertical forces due to ice buoyancy effects when water level rises, or by ice weight when water level falls. These vertical forces will persist until the ice sheet fractures due to bending or the adfreeze force is exceeded.
  - (a) Cylindrical piles.
  - In cases where the ice sheet freezes around a pile, forces will be exerted on the pile if the water level rises or falls. A rising water level will lift the ice sheet, and under certain conditions the uplift force on the pile may be sufficient to pull the pile free. Similarly, during falling water levels the weight of the ice sheet will exert a downward force on a pile which may be sufficient to buckle a slender pile.

Kerr (1975) studied vertical loads on cylindrical piles and presented equations for calculating loads under the conservative assumption that the water level change is rapid enough to assure elastic ice behavior before failure. A closed-form solution to the governing equation was obtained in terms of Bessel functions, and Kerr presented a numerically evaluated solution in graphical form as shown on Figure VI-5-145. The graphical solution is dimensional, and it has the functional form of

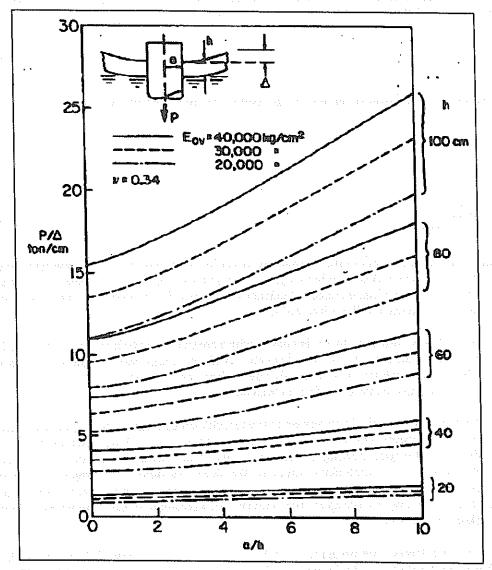


Figure VI-5-145. Vertical ice forces on a cylindrical pile (Kerr 1975)

$$P = f (a, h, E_{av}, \Delta, \nu)$$
 (VI-5-320)

where

P = uplift force in metric tons (tonnes)

a = pile radius (cm)

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h = ice plate thickness (cm)

 $E_{av}$  = averaged Young's modulus for ice (kg/cm<sup>2</sup>)

 $\Delta$  = water level rise (cm) up to the thickness of the ice

v =Poisson's ratio

Kerr's solution gives estimates of the maximum vertical load assuming the ice sheet does not fail in shear or bending before the maximum load on the pile is reached. For example, the maximum uplift force on a pile with radius a = 100 cm surrounded by a 40-cm-thick ice sheet having an average Young's modulus of  $30,000 \text{ kg/cm}^2$  would be estimated from Figure VI-5-145 using a value of a/h = 2.5 giving  $P_{max} = 3.7 \Delta$ . The total maximum force for a 5-cm water level rise would be

$$P_{\text{max}} = 3.7 \Delta = 3.7 (5 cm) = 18.5 tonnes$$

- Kerr (1975) pointed out that the same analysis applied for falling water levels with only a sign change, thus Figure VI-5-145 can also be used decreasing water levels.
- (b) Vertical walls.
- Uplift or downward forces per unit horizontal length caused by vertical movement of ice sheets frozen to vertical walls can be approximated using the following formula (PIANC 1992)

$$\frac{F_{\nu}}{b} = \rho_{\nu} g \, \Delta h \, L_c \tag{VI-5-321}$$

where the characteristic length  $L_c$  is given as

$$L_{c} = \left[ \frac{E h_{i}^{3}}{12 \rho_{w} g (1 - v^{2})} \right]^{1/4}$$
 (VI-5-322)

and

 $F_v$  = total vertical force acting on the wall

b = horizontal length of wall

 $\Delta h$  = change in water level  $\frac{1}{2}$   $\frac{1}$ 

 $\rho_w = \text{density of water}$ 

g = gravitational acceleration

E =modulus of elasticity of ice

 $h_i$  = ice thickness

v = Poisson's ratio (0.31-0.35)

- As previously mentioned, the modulus of elasticity for ice varies with brine volume from about 1,000 MPa for very salty water to about 6,000 MPa for freshwater ice. For freshwater ice,  $L_c$  is typically between 15 to 20 times the ice thickness with a reasonable rule-of-thumb being  $L_c = 17 h_i$ .
- (c) Sloping structures. The additional vertical load caused by the ride-up and piling of ice on sloping structures needs to be evaluated for the local conditions and specific type of structure. Ice piled up on the slope could initiate slumping of the armor layer on steeper slopes. During rising waters, individual armor stones or revetment units might be lifted out by adfreeze forces.
  - (4) Aspects of slope protection design.
- (a) Much of our understanding of successful slope protection design in cold coastal regions stems from practical experience as documented in the technical literature. In general the design philosophy recognizes that little can be done to prevent ice contact with slope protection structures. Therefore, emphasis is placed on minimizing potential ice damage using a variety of techniques.
- (b) Leidersdorf, Gadd, and McDougal (1990) reviewed the performance aspects of three types of slope protection used for coastal projects related to petroleum activities in the Beaufort Sea. For water depths less than 2 m, sacrificial beaches appeared to function well. In water depths ranging from 7 m to 15 m, gravel-filled geotextile bags were able to withstand the larger wave forces, but they were susceptible to ice damage and required regular maintenance. Linked concrete mat armor (Leidersdorf, Gadd, and McDougal 1988; Gadd and Leidersdorf 1990) withstood both wave and ice loads in depths up to about 14 m. Mats were recommended for projects with a lengthy service life so that high initial capital costs would be offset by lower maintenance expenses.

Wuebben (1995) reviewed the effects of ice on riprap structures constructed along ice-prone waterways. This paper provided a good summary of successful riprap revetment design and construction practices based on actual field experience. Numerous useful references documenting ice effects on riprap are included in Wuebben's paper.

The following rules-of-thumb for arctic slope protection were given in Chen and Leidersdorf (1988) and summarized in PIANC (1992).

- Cover layers and underlayers should be strong enough to withstand local penetration by thick ice sheets.
- Smooth slopes without protrusions will reduce loads and allow the ice to ride up more easily without plucking out individual armor elements. (However, wave runup will be greater.)
- Flexible cover layers consisting of graded riprap may help absorb impacts by smaller ice blocks during wave action without appreciable damage. Sand bags are effective for structures with intended short service lives.
- Mild structure slopes are essential because they reduce the risk of ice penetration into the slope.
   Maximum slope 15 deg is recommended in the zone of ice attack.
- Compound slopes with a nearly horizontal berm above the swl provide a platform for piled-up ice in regions which experience frequent ride-up of ice sheets.

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 Maximum ice loads will not occur at the same time as maximum expected wave loads. Therefore, slope design can consider each load condition separately.

#### VI-5-9. References

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